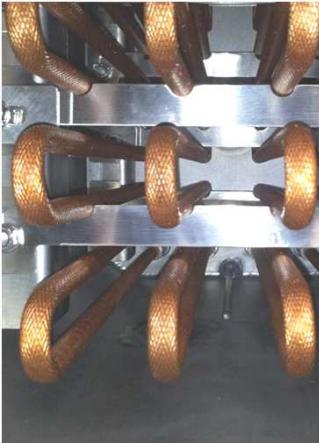
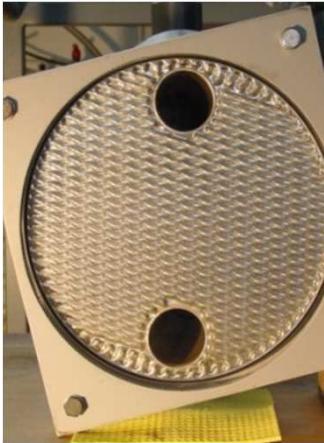


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Energy Conversion by Sorption Processes: Fundamental Options and Challenges

Felix Ziegler | Institut für Energietechnik





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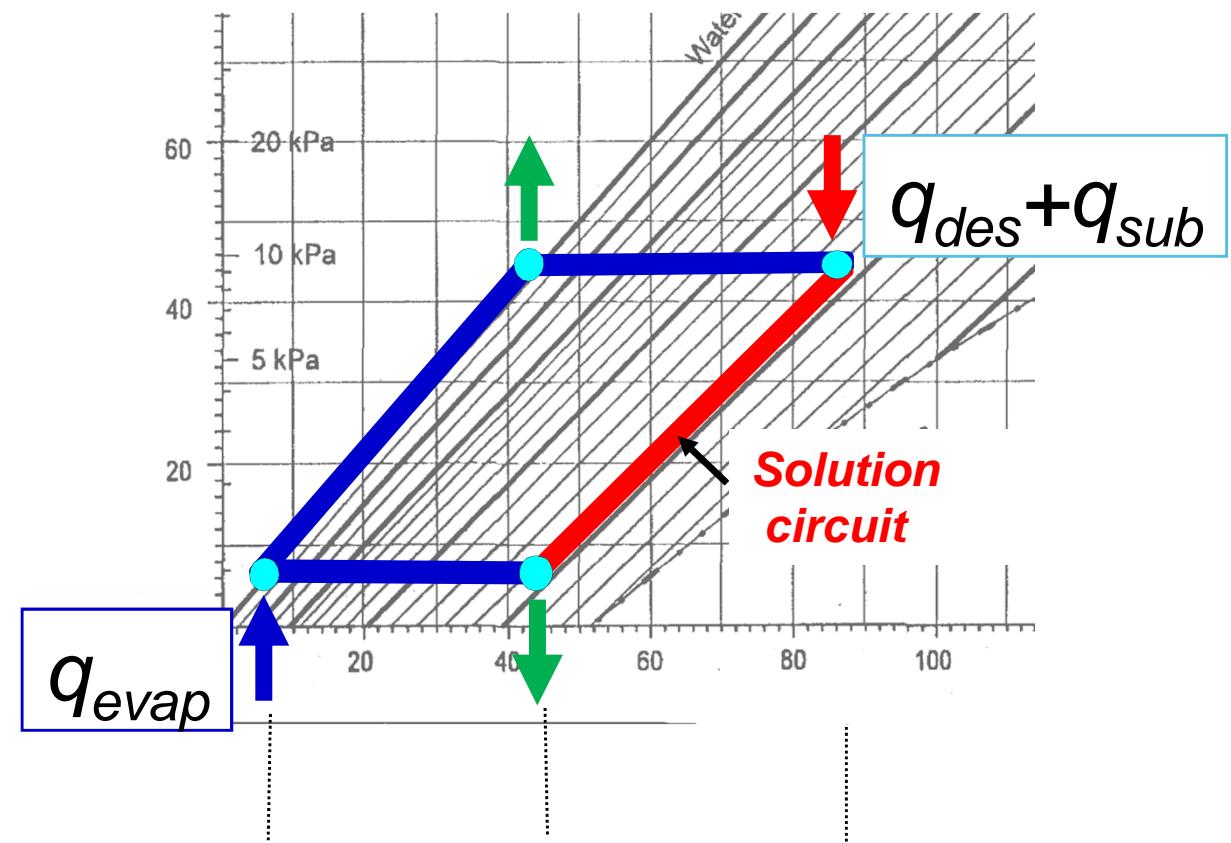
1. New fluids will not bring better COPs.
New fluids may cure problems of today's fluids.

2. The transport properties are most important.

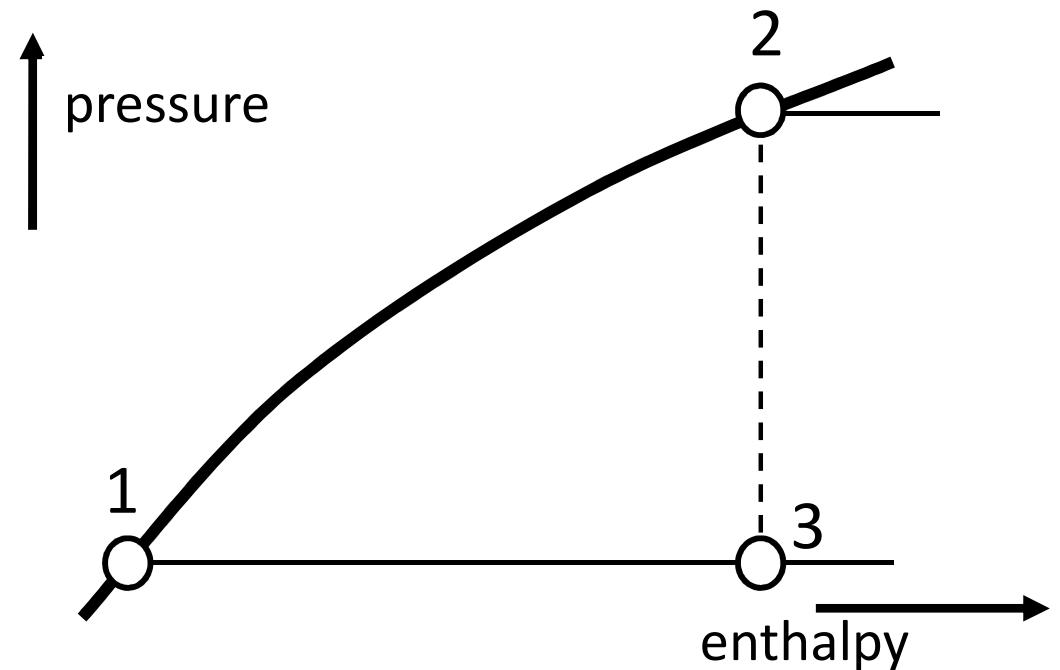
3. New cycles may bring forth new fields of application.

$$COP = \frac{q_{evap}}{q_{des} + q_{sub}}$$

$$\approx \frac{r}{r+l}$$



$$COP = \frac{q_{evap}}{q_{des} + q_{sub}}$$



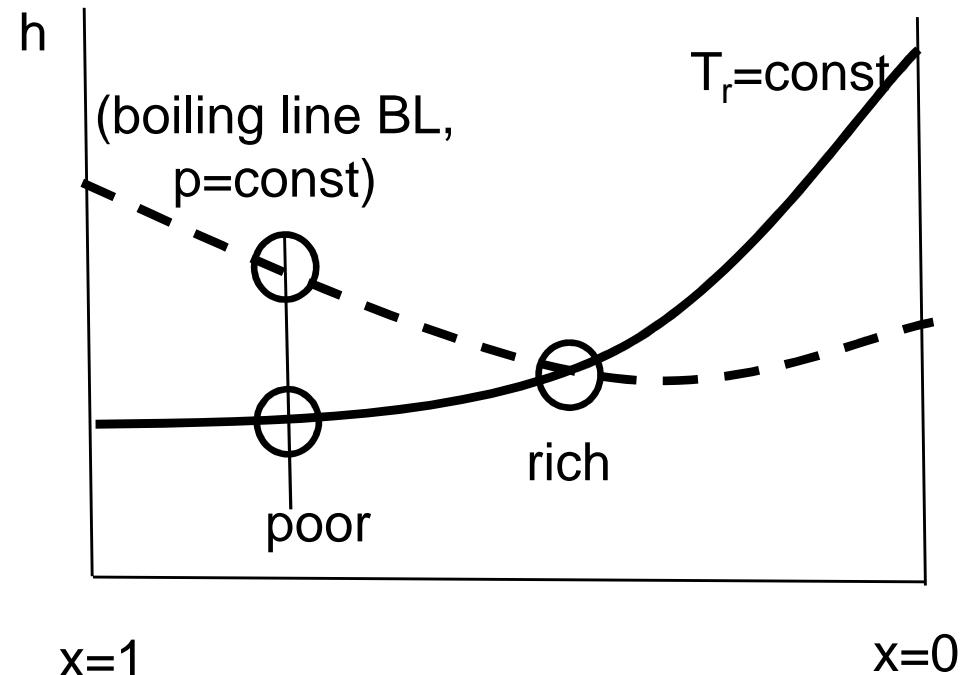
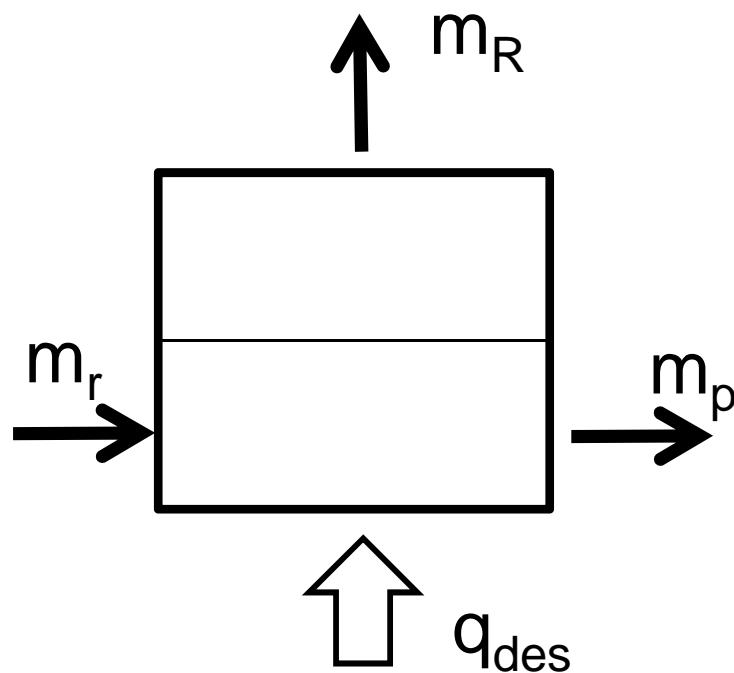
$$q_{evap} = r_0 - (h_2 - h_1) \approx r_0(1 - St_{th})$$

$$St_{th} = \frac{c_l \Delta T_{Lift}}{r_0}$$

r latent heat of vaporisation

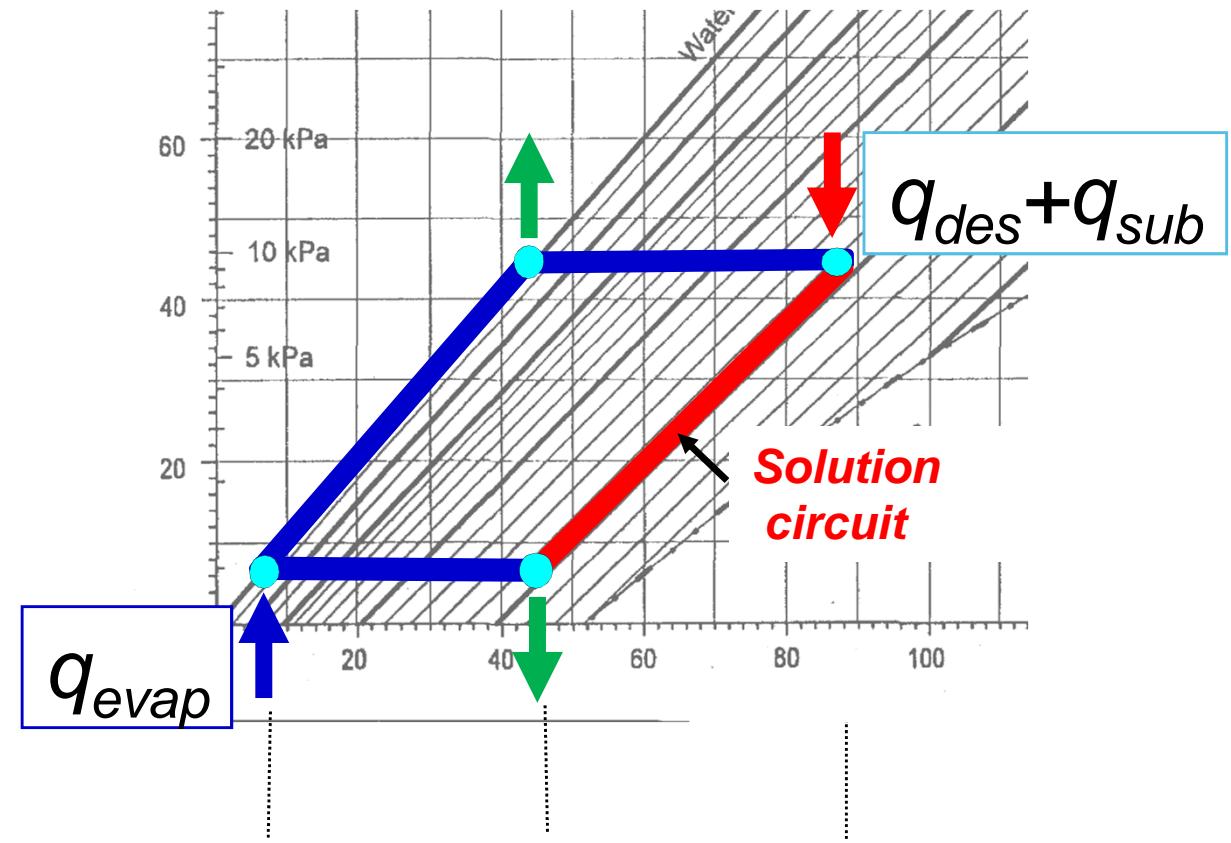
$$COP = \frac{q_{evap}}{q_{des} + q_{sub}}$$

$$q_{des} = (r + l)_r + x_r c_p \left. \frac{dT}{dx} \right|_{BL}$$



$$\lambda = \frac{l_2}{r_2}$$

Relative differential
heat of solution



$$COP \approx \frac{r_0(1 - St_{th})}{r_2 \left(1 + \lambda + f St_{sub} + \frac{x_r}{\Delta T_L} St_p \left. \frac{dT}{dx} \right|_{BL} \right)}$$

$$COP \approx \frac{r_0(1 - St_{th})}{r_2 \left(1 + \lambda + f St_{sub} + \frac{x_r}{\Delta T_L} St_p \left. \frac{dT}{dx} \right|_{BL} \right)}$$

$$St = \frac{c\Delta T}{r}$$

Stefan-numbers should be small

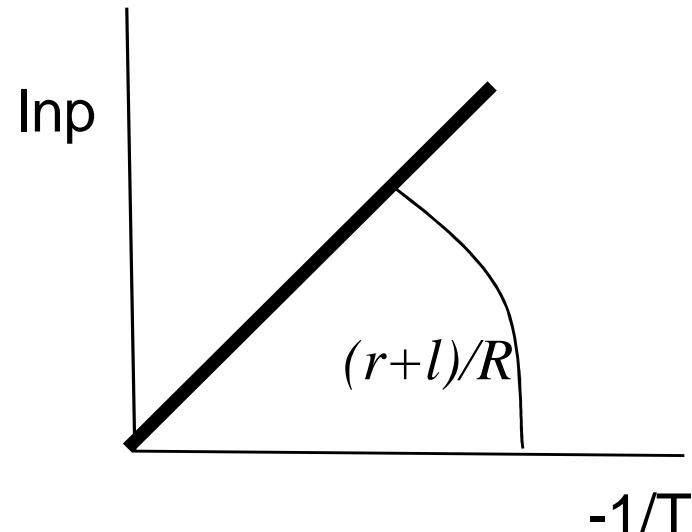
$$COP \approx \frac{r_0(1 - St_{th})}{r_2 \left(1 + \lambda + f St_{sub} + \frac{x_r}{\Delta T_L} St_p \left. \frac{dT}{dx} \right|_{BL} \right)}$$

$\lambda = \frac{l_2}{r_2}$ Relative differential heat of solution
should be small (or negative?)

Clausius-Clapeyron:

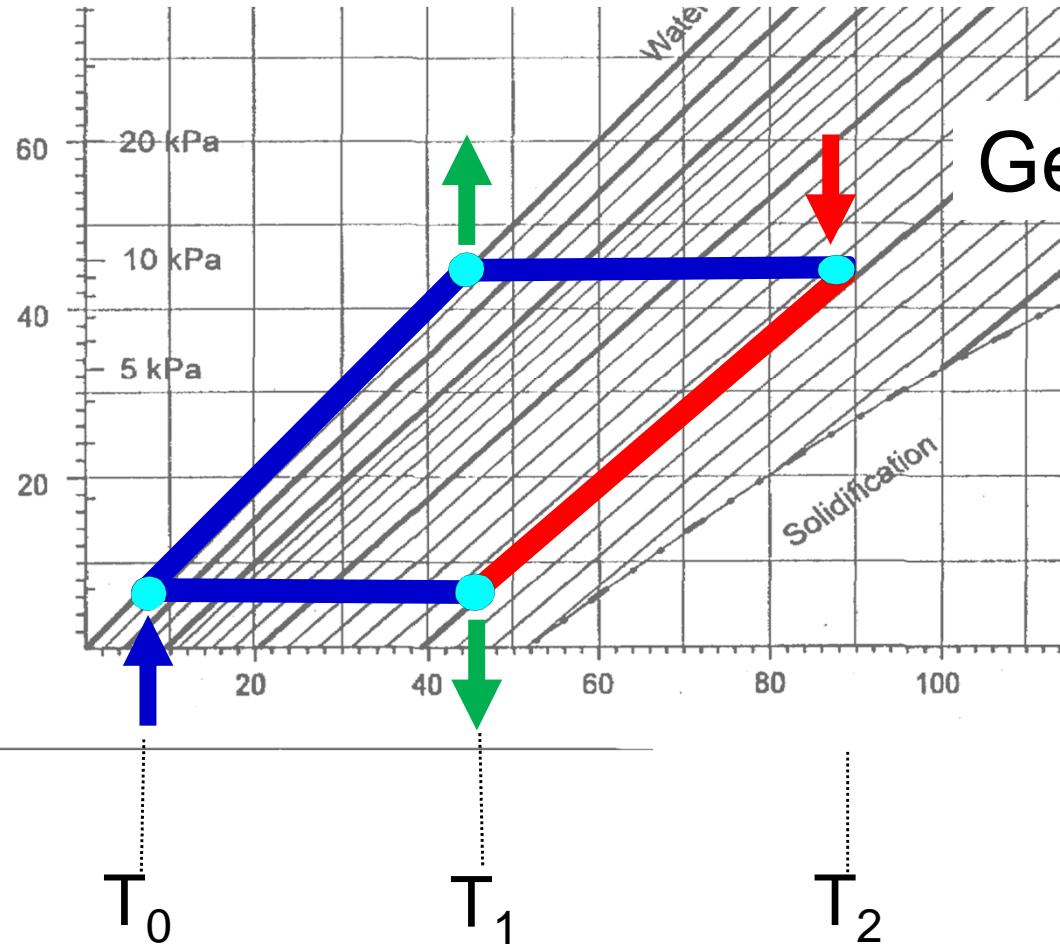
$$\frac{d \ln p}{d(-1/T)} \approx \frac{h''_{10} - h'_1}{R}$$

$$\frac{d \ln p}{d(-1/T)} \approx \frac{r+l}{R}$$



r latent heat of vaporisation

$\lambda = \frac{l}{r}$ relative differential enthalpy of solution

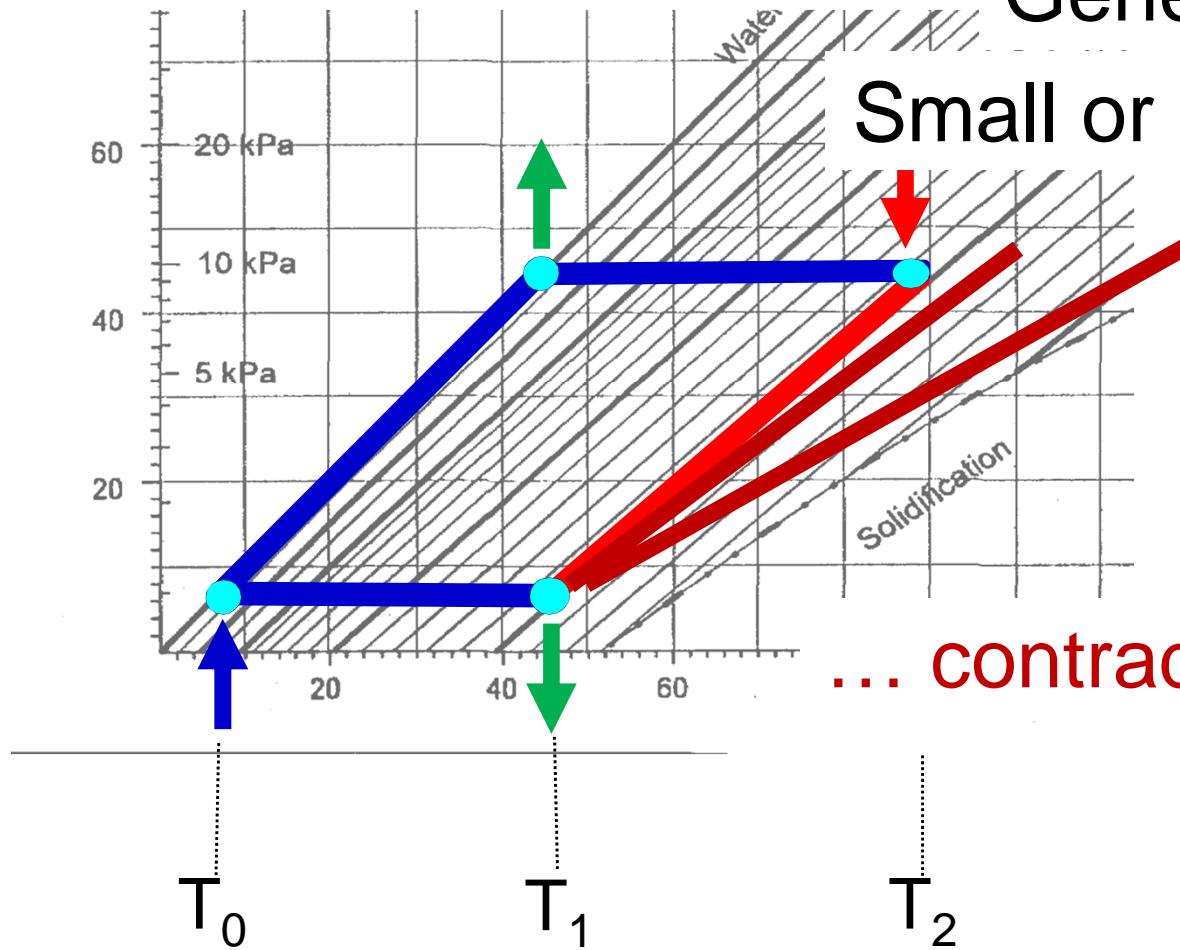


Generator temperature:
„normal“ λ

$$(T_2 - T_1) \approx (T_1 - T_0) \left(1 + 2 \frac{T_1 - T_0}{T_1} - \lambda \right)$$

Generator temperature:

Small or negative λ : rising T_G

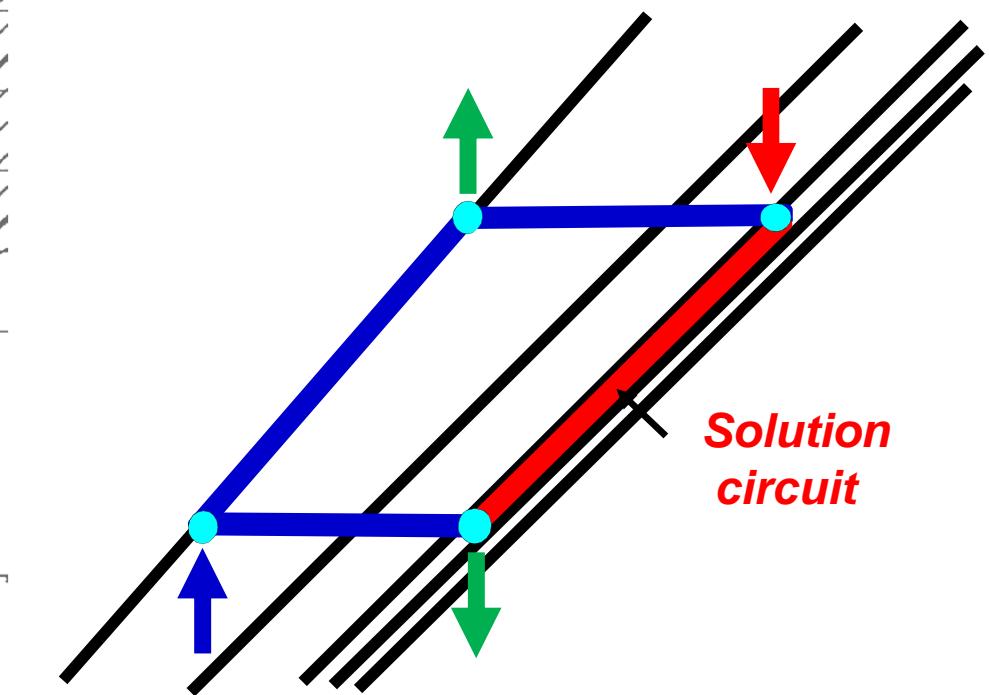
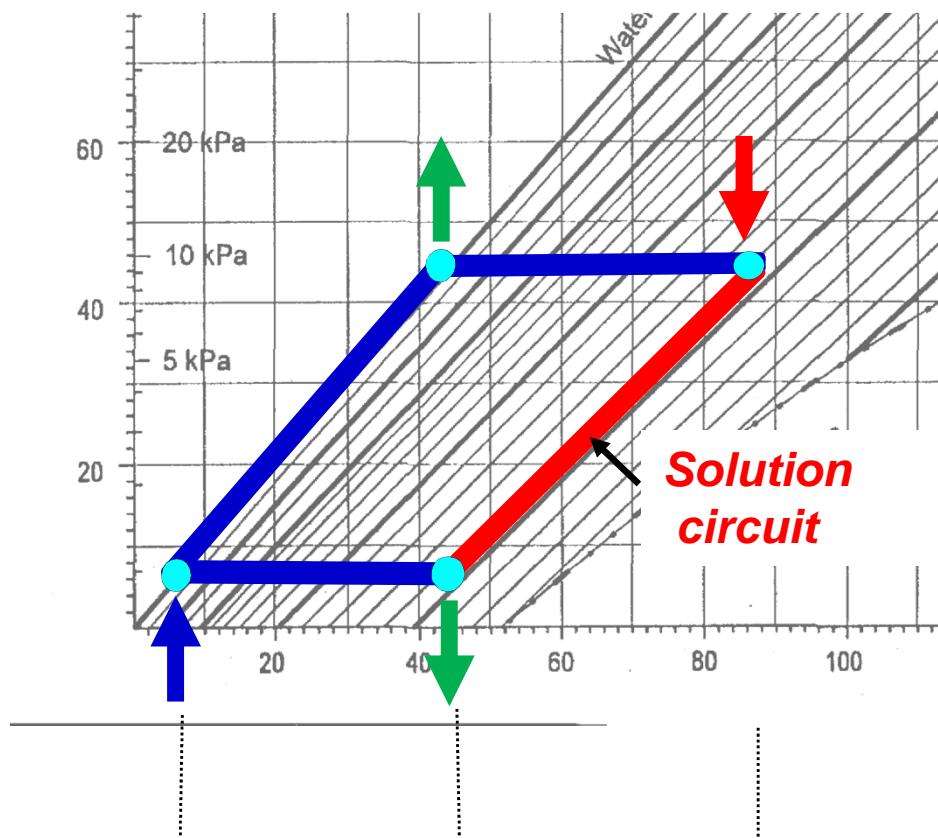


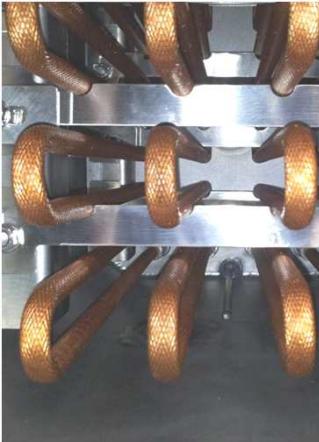
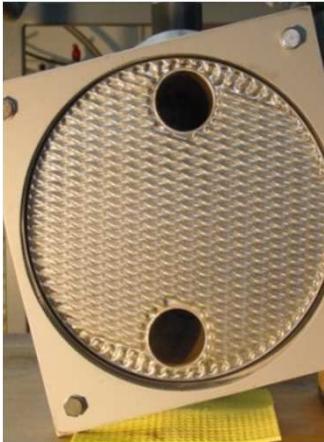
... contradicts Trouton's Rule

$$(T_2 - T_1) \approx (T_1 - T_0) \left(1 + 2 \frac{T_1 - T_0}{T_1} - \lambda \right)$$

$$COP \approx \frac{r_0(1 - St_{th})}{r_2 \left(1 + \lambda + f St_{sub} + \frac{x_r}{\Delta T_L} St_p \frac{dT}{dx} \Big|_{BL} \right)}$$

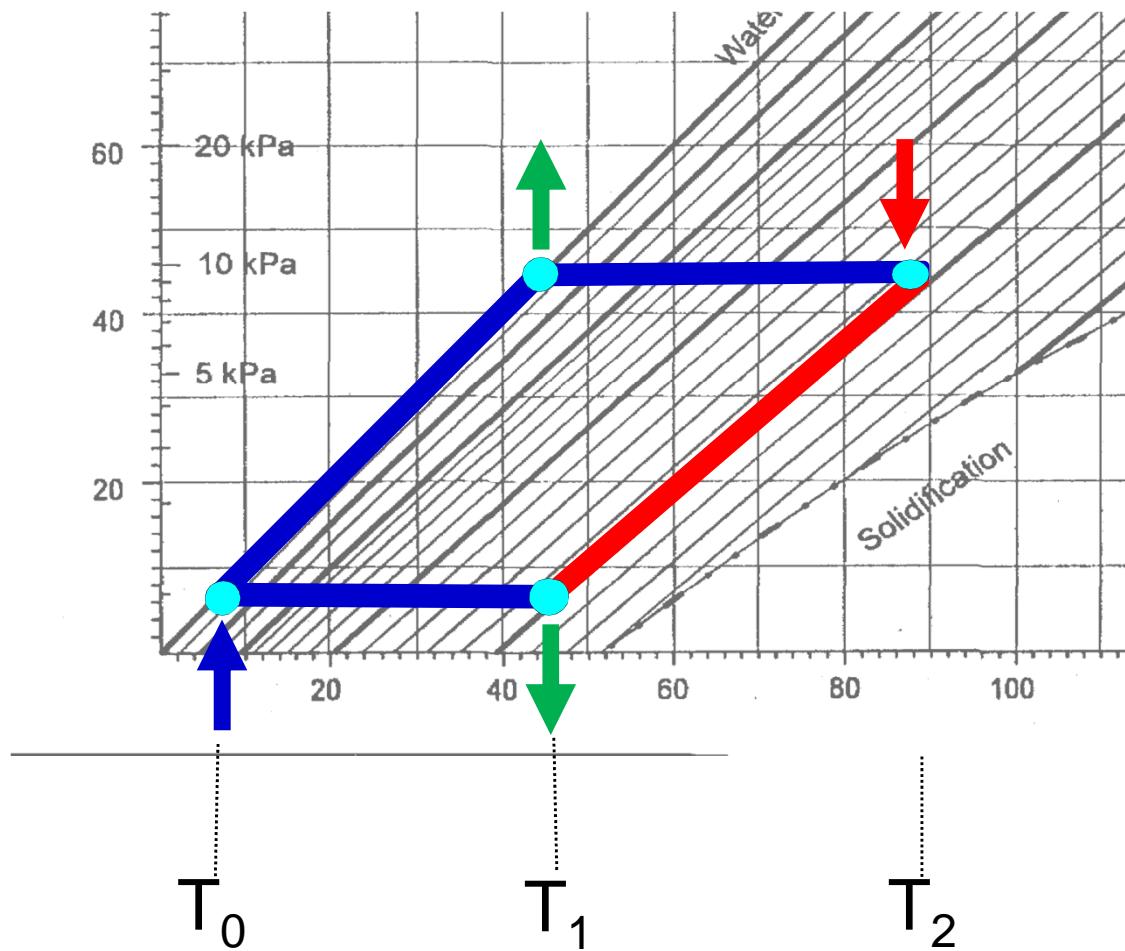
$\frac{x_r}{\Delta T_L}$	small
$\frac{dx}{dT} \Big _{BL}$	large





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1. New fluids will not bring better COPs.
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2. The transport properties are most important.
3. New cycles may bring forth new fields of application.



$$(T_2 - T_1) \approx (T_1 - T_0) \left(1 + 2 \frac{T_1 - T_0}{T_1} - \lambda \right)$$

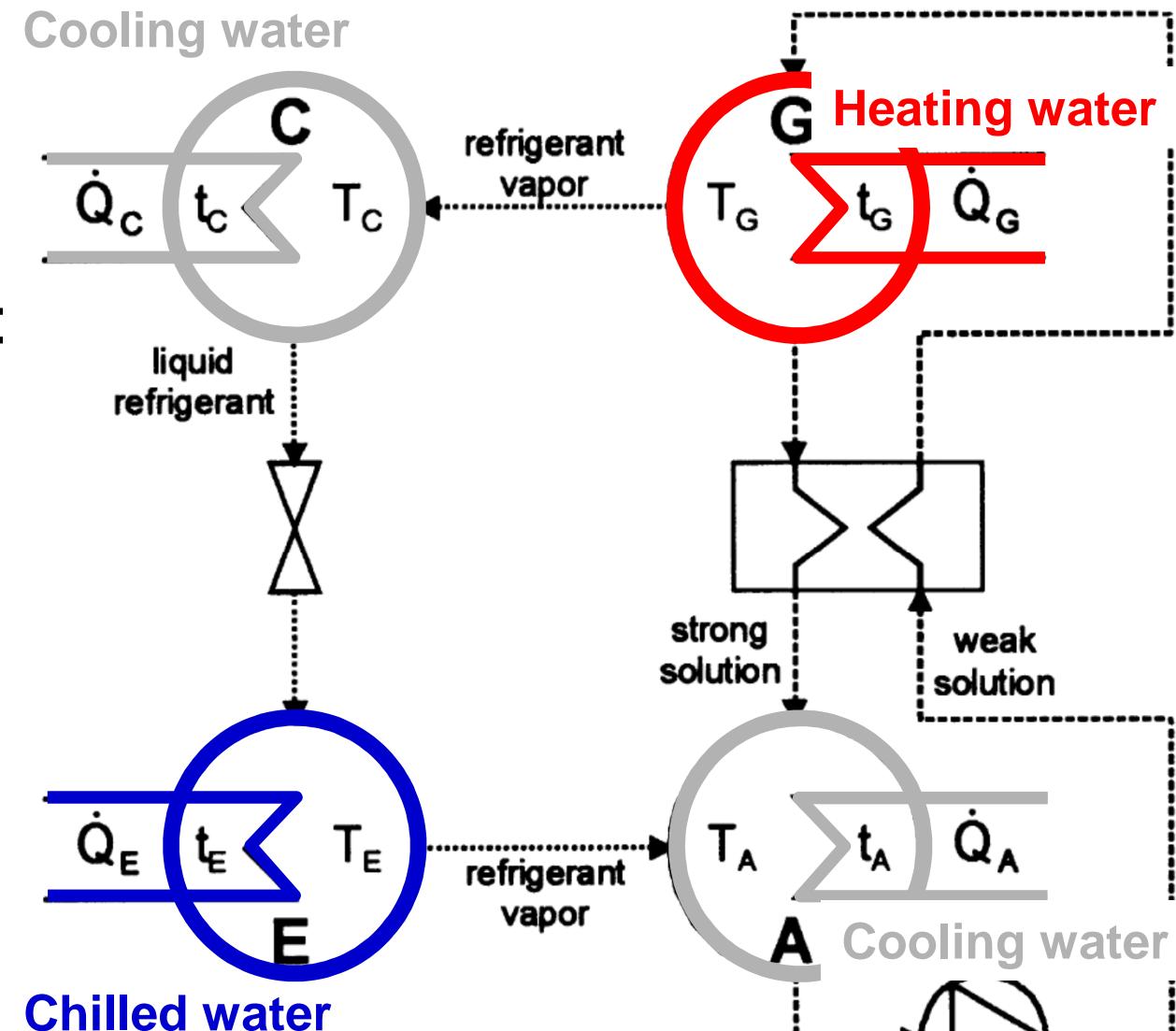
Heat transfer across each main component:

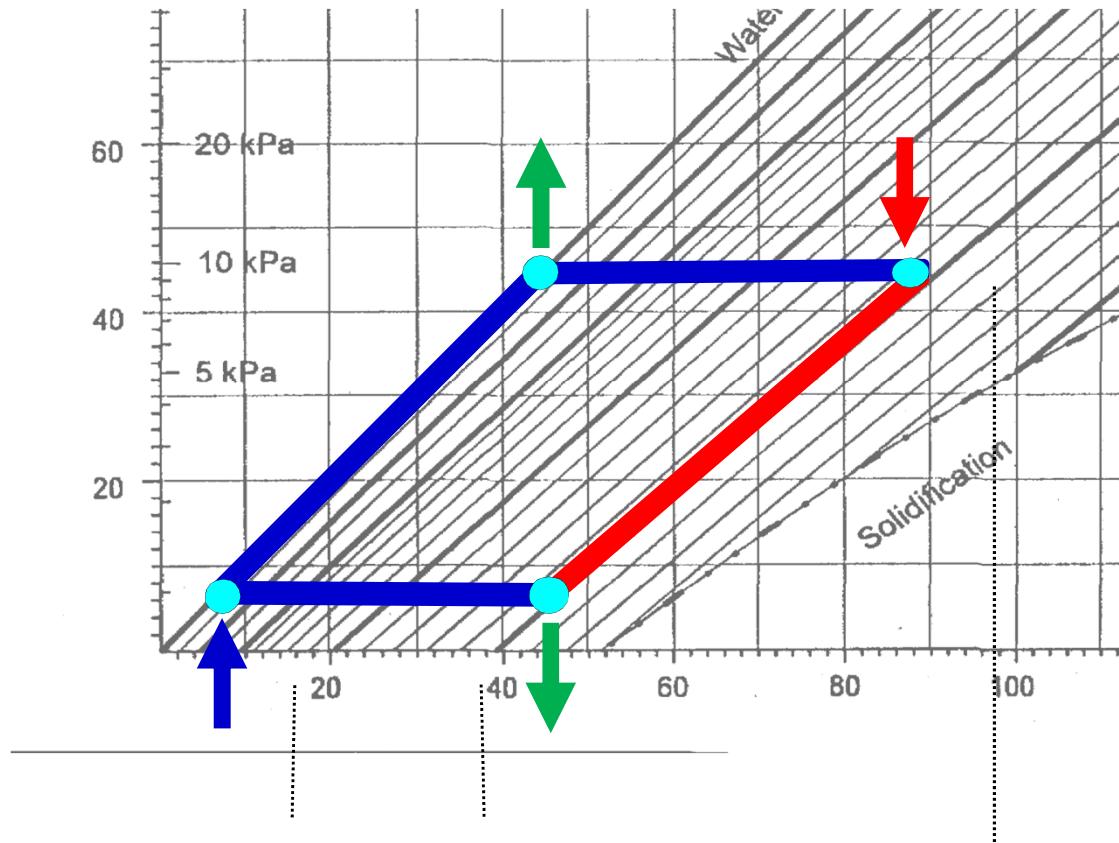
$$\dot{Q}_X = (UA)_X |T_X - t_X|$$

„internal“
(working pair)

„external“
(application)

$$\dot{Q}_X = \frac{U}{A} X \Delta T$$





ΔT is more important than λ !

Determined by transport properties

$$(t_2 - t_1) \approx (t_1 - t_0 + 2\Delta T) \left(1 - \lambda + 2 \left(\frac{t_1 - t_0}{t_1} + \frac{2\Delta T}{t_1} \right) \right) + 2\Delta T$$

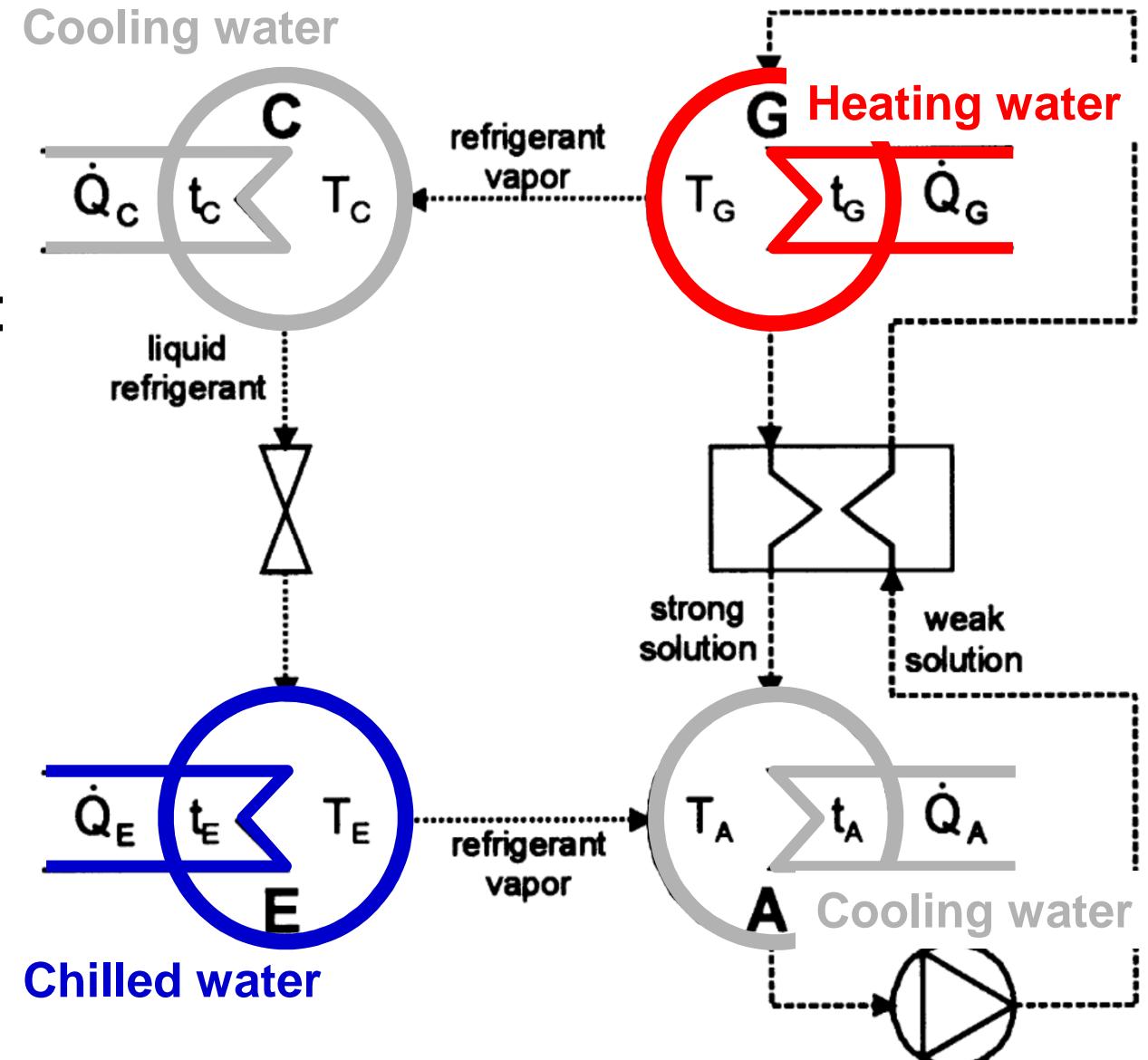
$$(t_2 - t_1) > (t_1 - t_0 + 4\Delta T)$$

Heat transfer across each main component:

$$\dot{Q}_X = (UA)_X |T_X - t_X|$$

„internal“
(working pair)

„external“
(application)



Small changes in temperatures impact strongly on power density!

Characteristic (external)
temperature difference

$$\Delta\Delta t = a_0 - b_1 + c_2$$

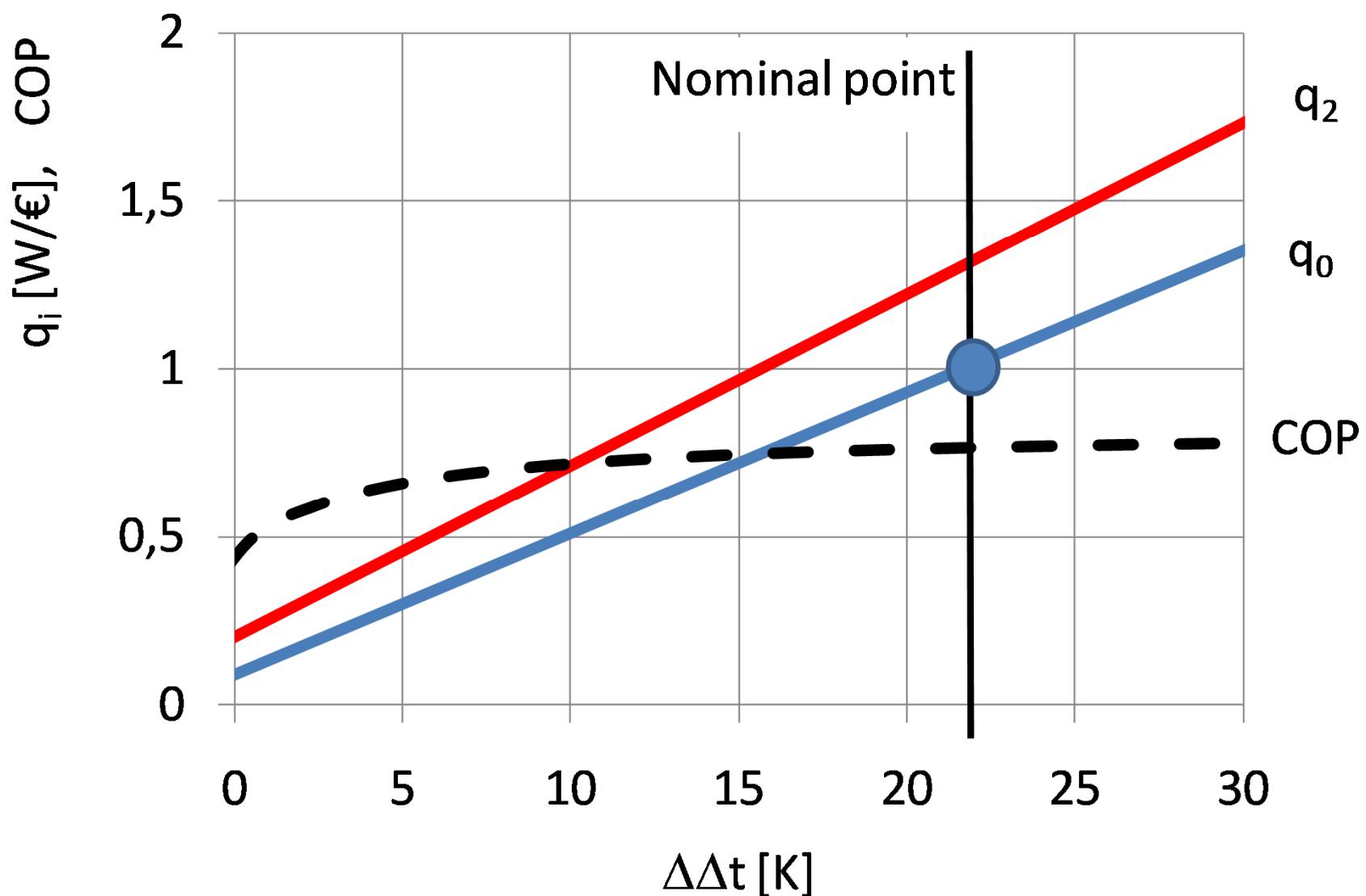
a, b, c from equilibrium properties;
not very variable

specific cooling power [W/€] $q_0 = s_0 + m_0 \Delta\Delta t$

specific heat input [W/€] $q_2 = s_2 + m_2 \Delta\Delta t$

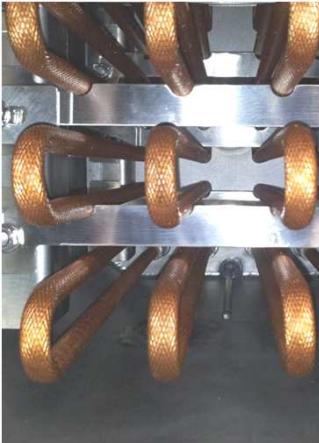
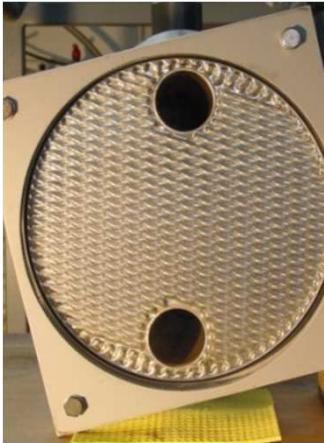
$$COP = \frac{q_0}{q_2} = \frac{s_0 + m_0 \Delta\Delta t}{s_2 + m_2 \Delta\Delta t}$$

s and m from specific cost and heat transfer
(viscosity, density, wettability, diffusivity...);
very variable!



Sorption machines are cheap if:
 t_0 is high; t_1 is low, t_2 is high
Heat and mass transfer is good

$$\Delta\Delta t = a_0 - b_1 + c_2$$



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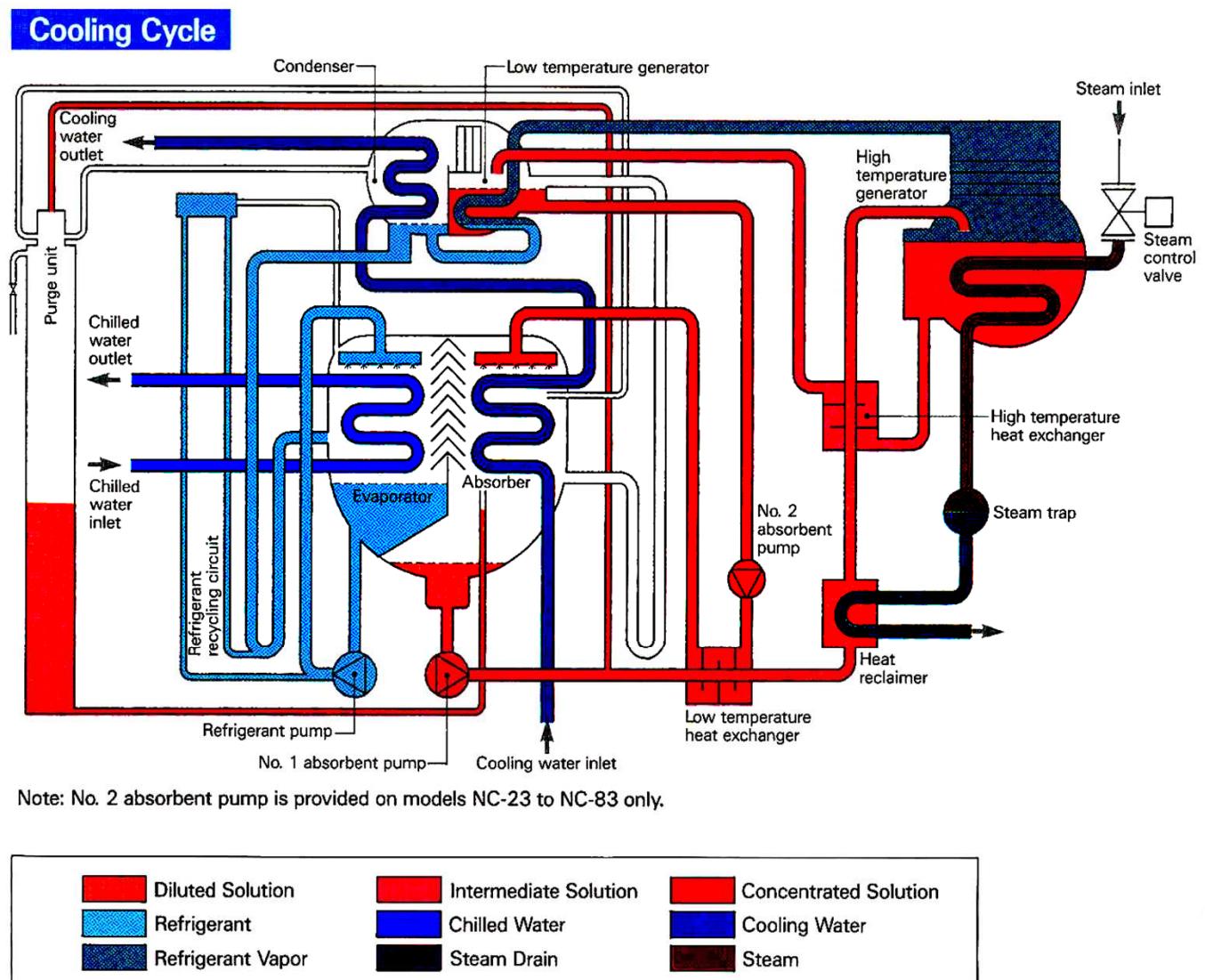
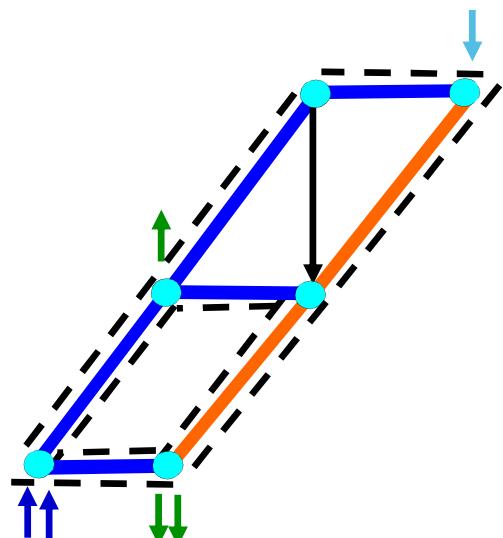


Multistage cycles are combinations of elementary cycles.

They are used to ...

... increase the COP

e.g.: double-effect



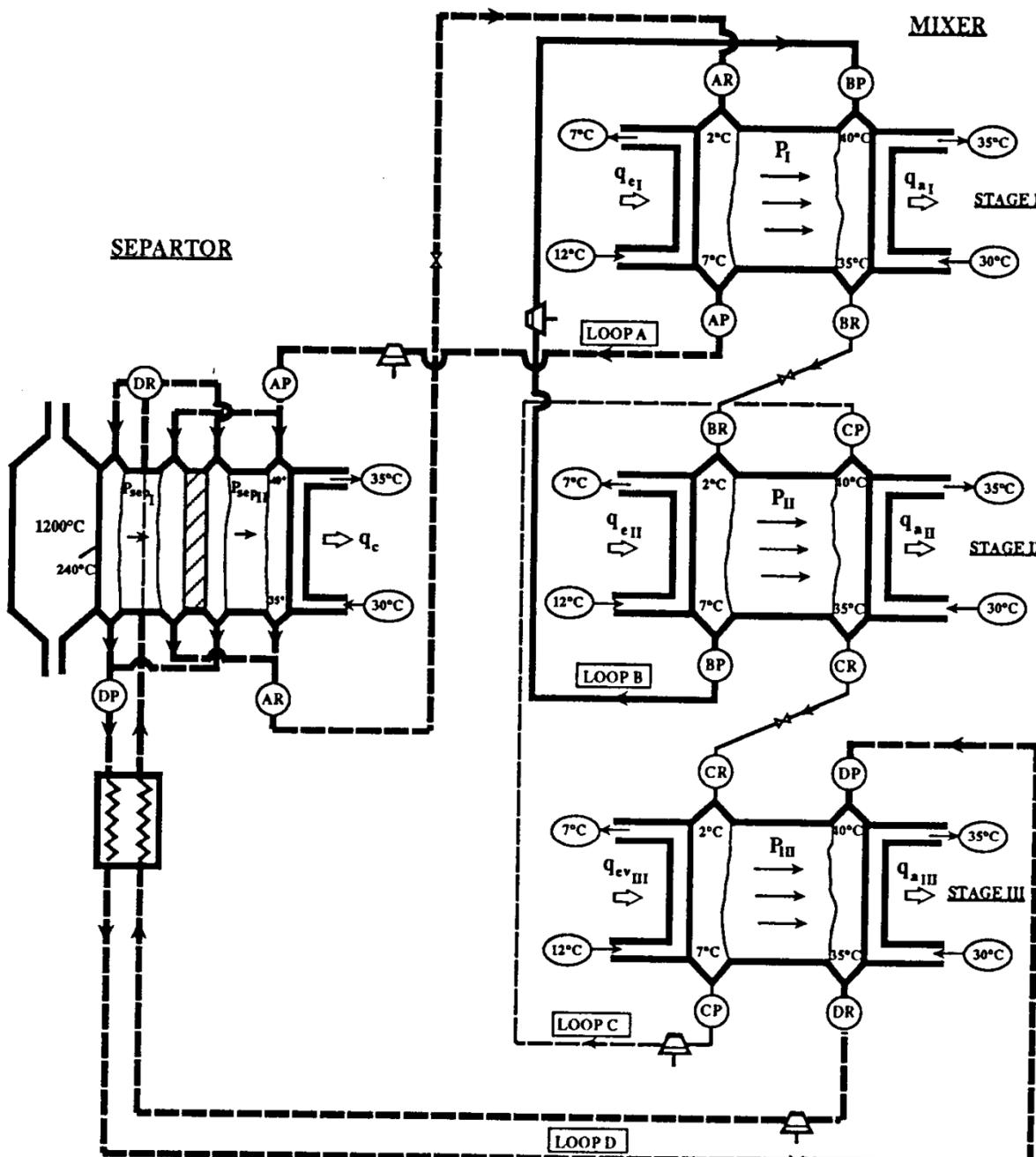
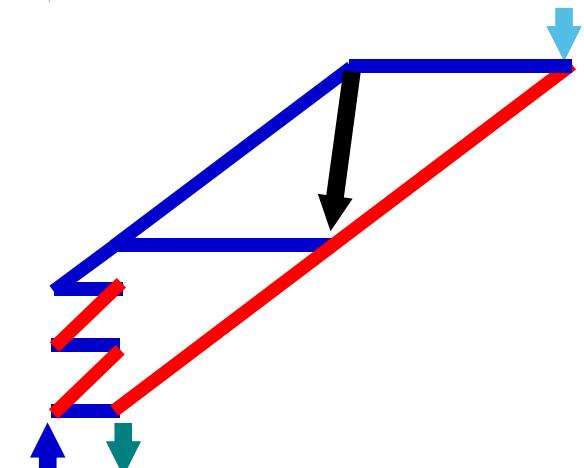


Figure 2: Working principle of multi-stage absorption heat pump



6-effect chiller
(Le Goff, 1994)

... or
to increase the temperature lift

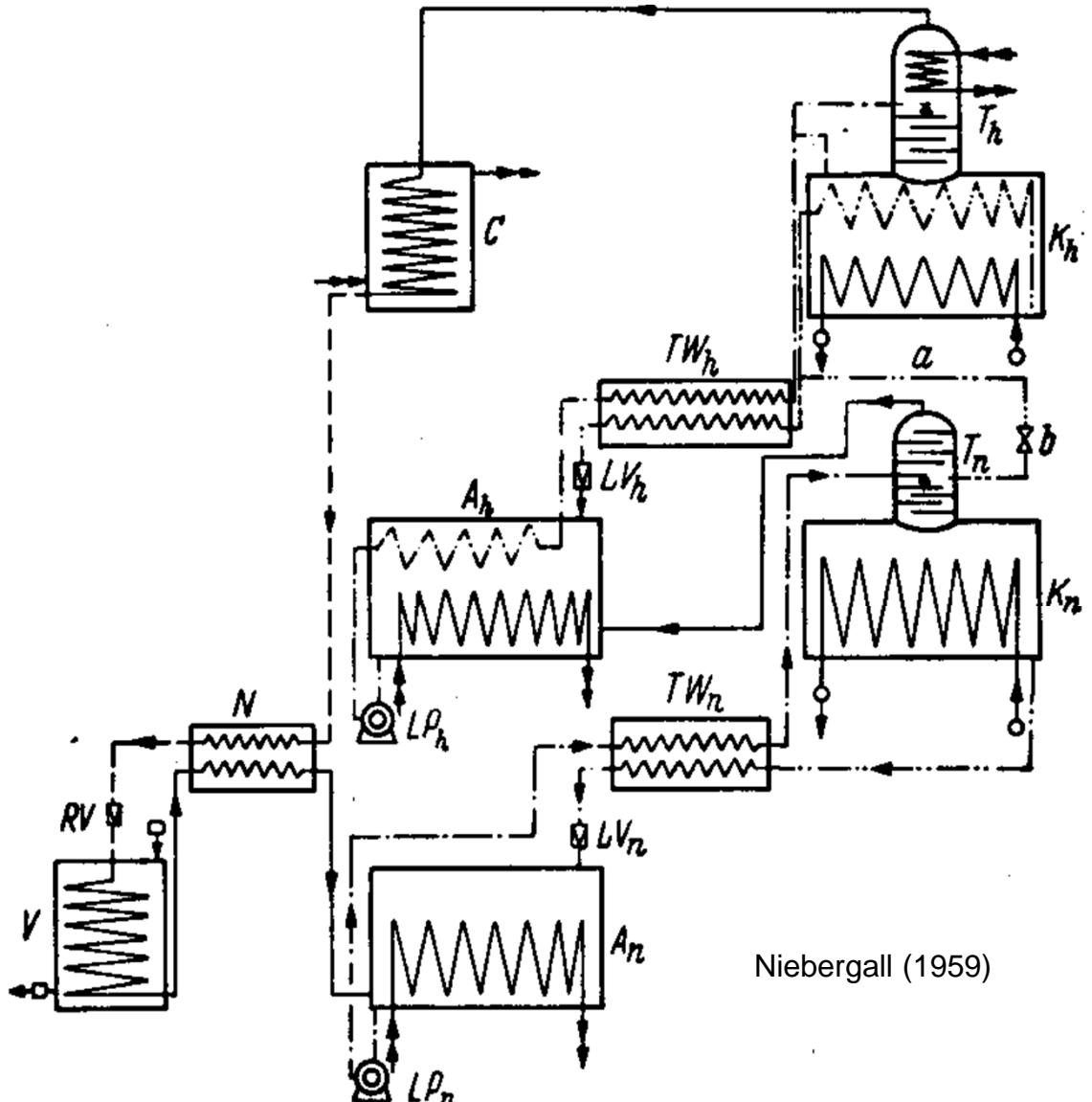
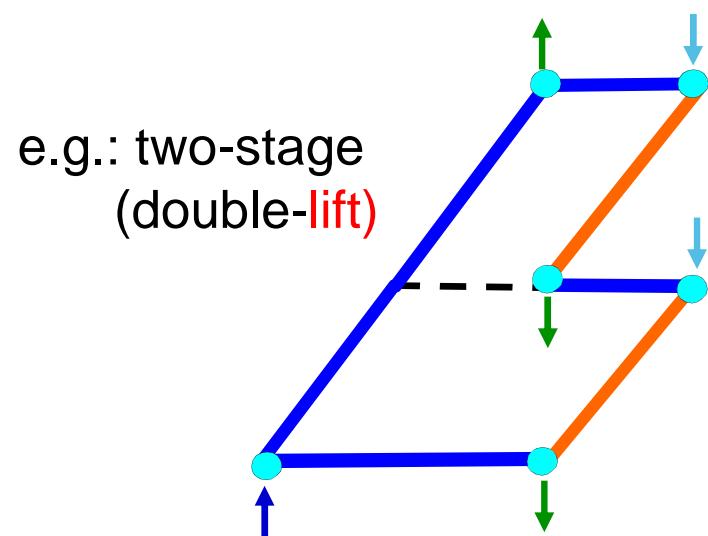
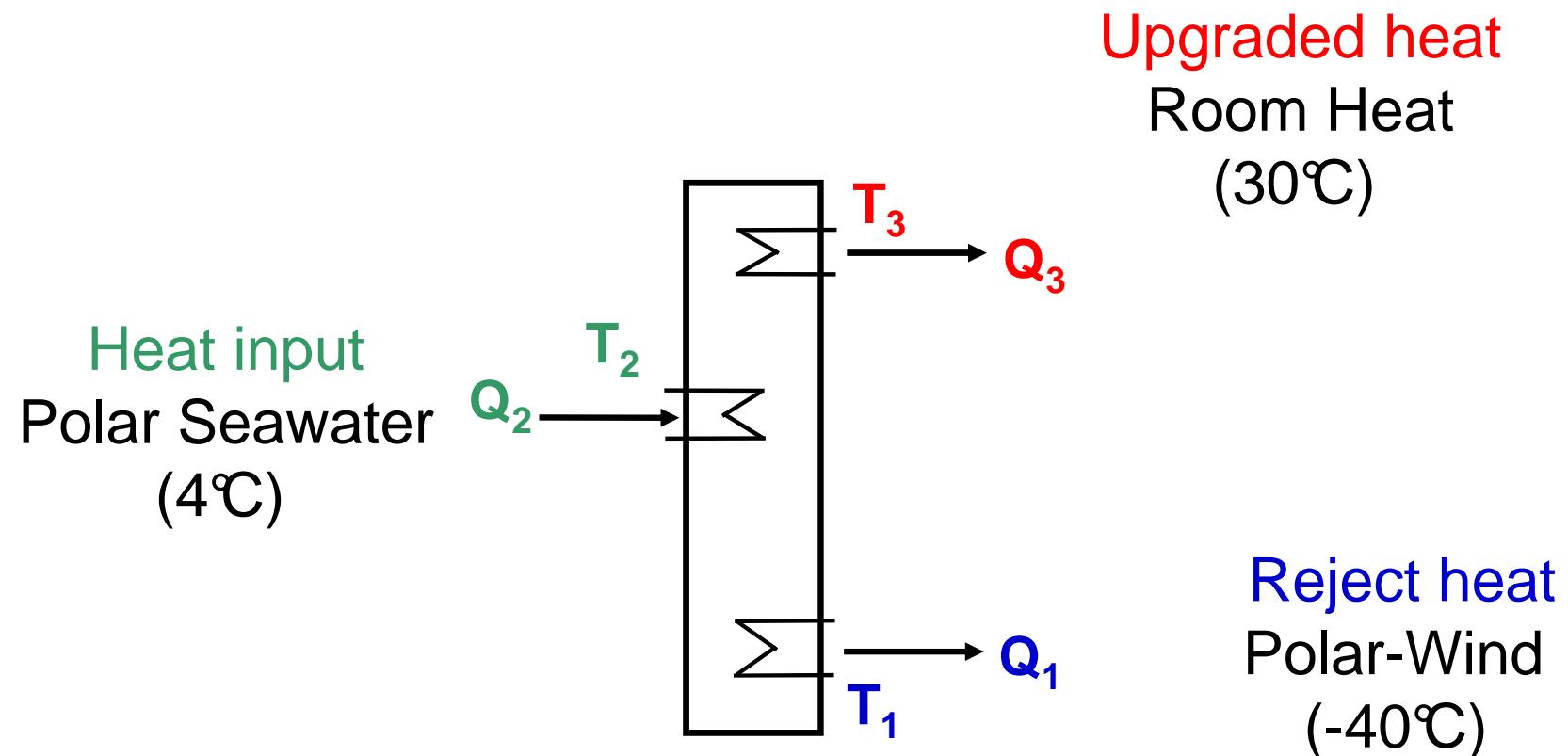


Abb. 26. Schaltbild einer zweistufigen Absorptions-Kälteanlage für Tiefkühlung gemäß Abb. 25.

Heat transformer (Templifier, heat pump type II)

Special: heat from the cold sea (Pierre LeGoff, ~1980)



compression-absorption hybrid: First practical proposition by Osenbrück (1895):

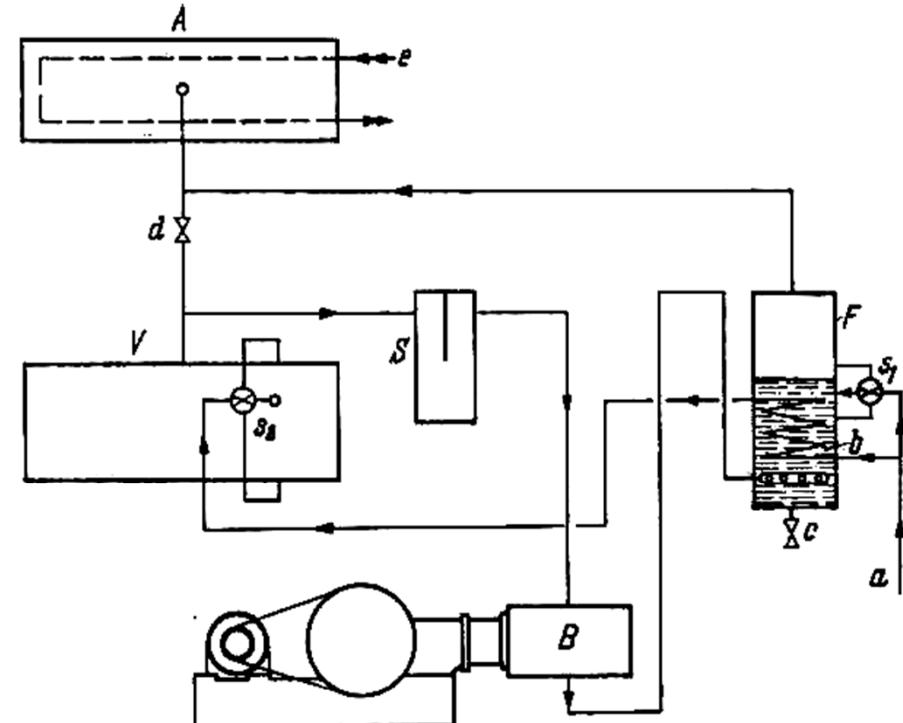
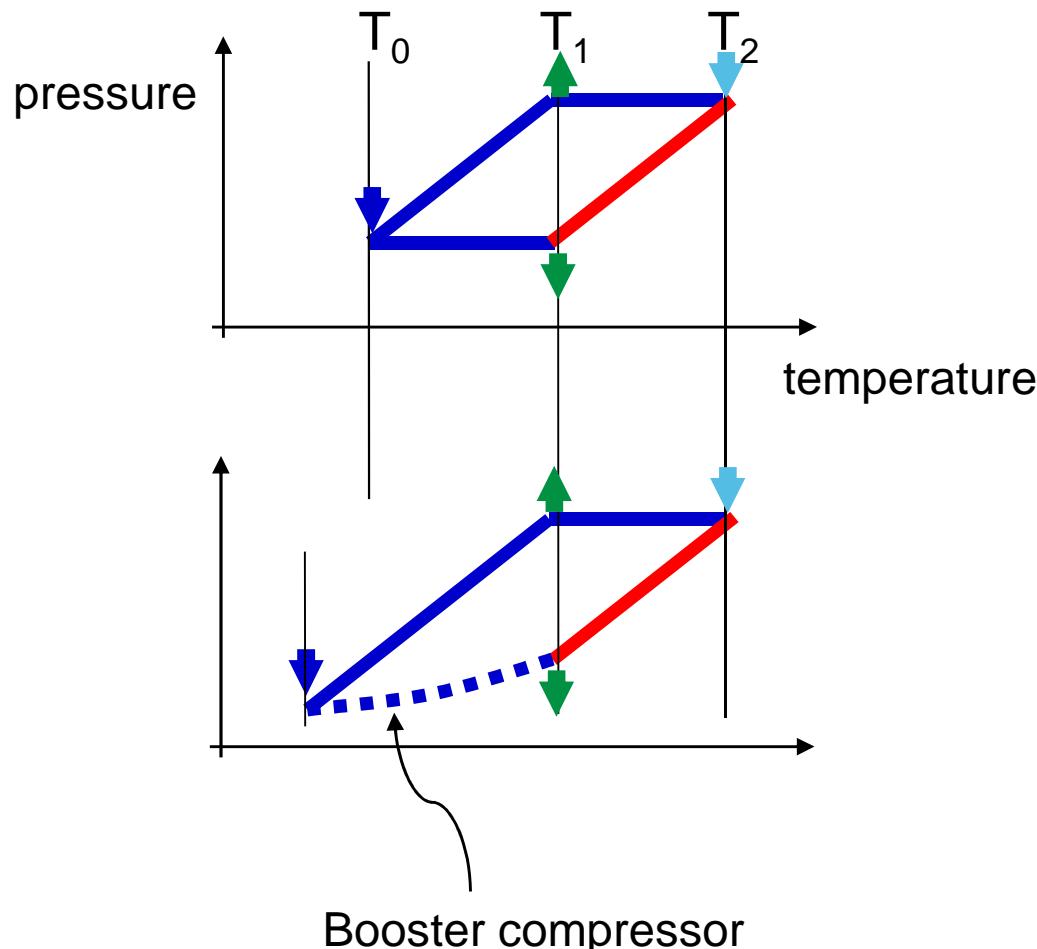
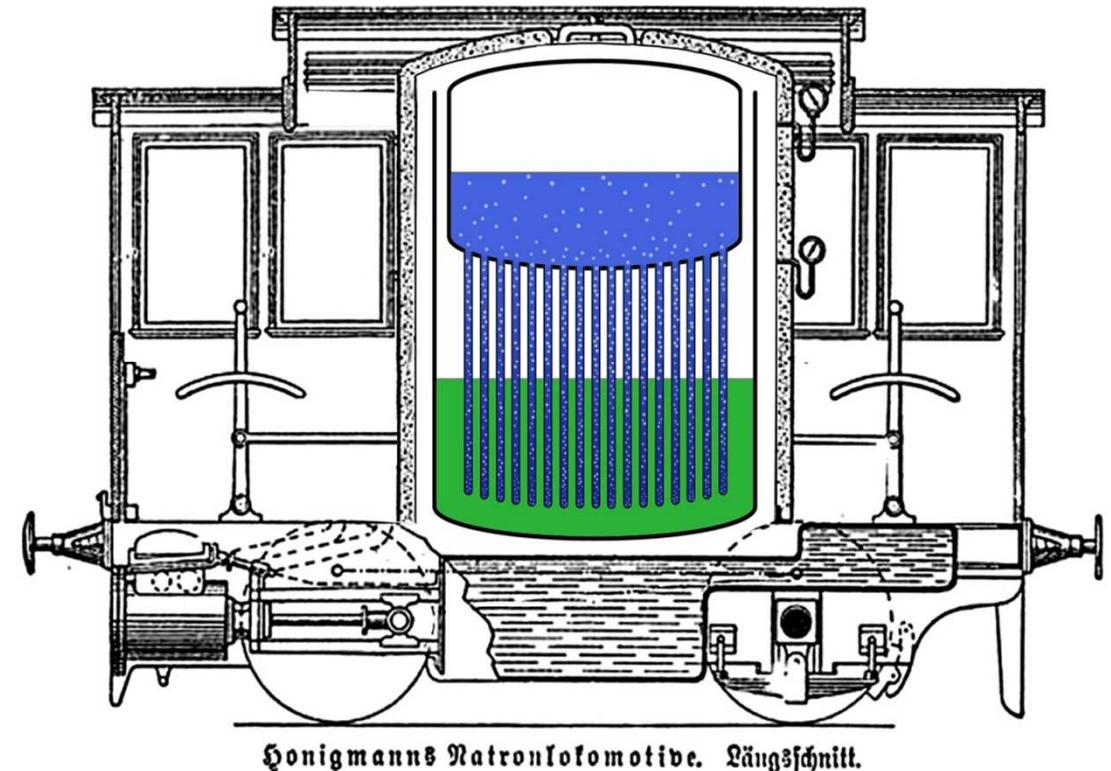


Abb. 40. Einschaltung eines Vorschaltkompressors (Booster) in eine Absorptions-Kälteanlage.

A Absorber; B Booster; F Ammoniakflüssigkeitskühler und Ölabscheider; S Abscheider; V Verdampfer; a Ammoniakeintritt; b Kühlslange für flüssiges Ammoniak; c Ölableß; d Absperrventil; e Kühlwasserzulauf für den Absorber; s_1 und s_2 Flüssigkeitsschwimmerventil.

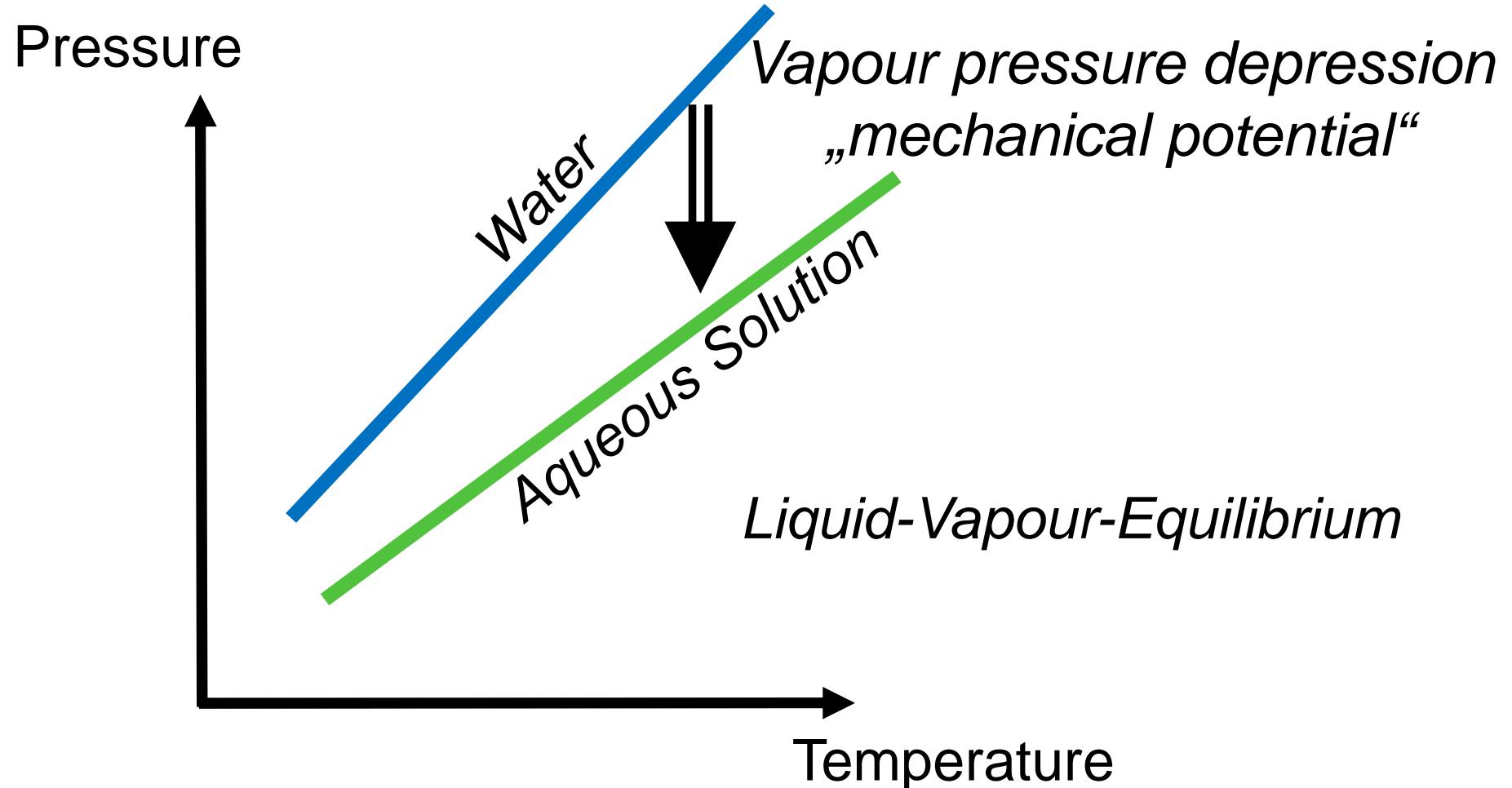
Niebergall (1959)

„Honigmann“ cycle: storage and conversion of low-grade heat

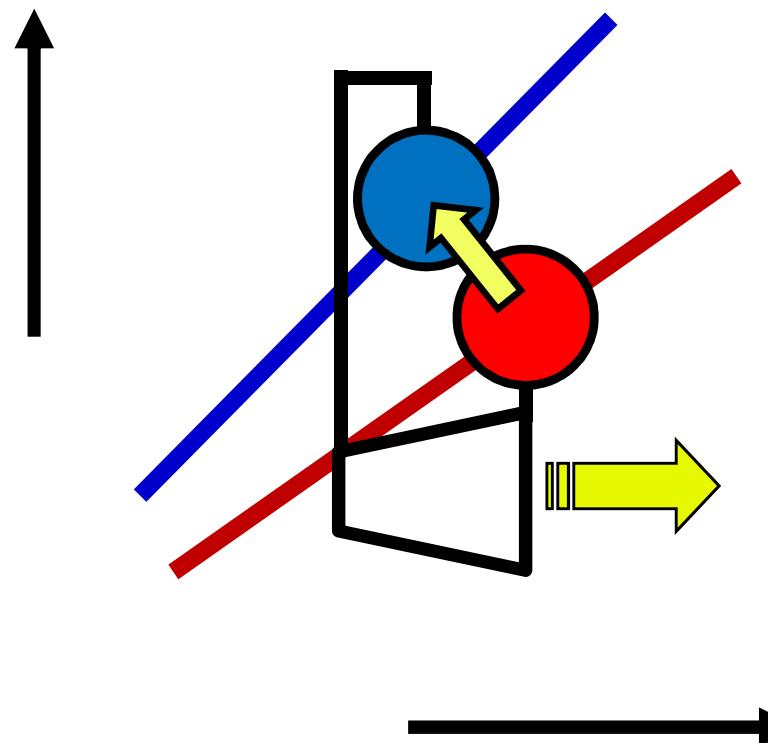


Honigmann fireless locomotive (1883)

„Honigmann“ cycle: storage and conversion of low-grade heat

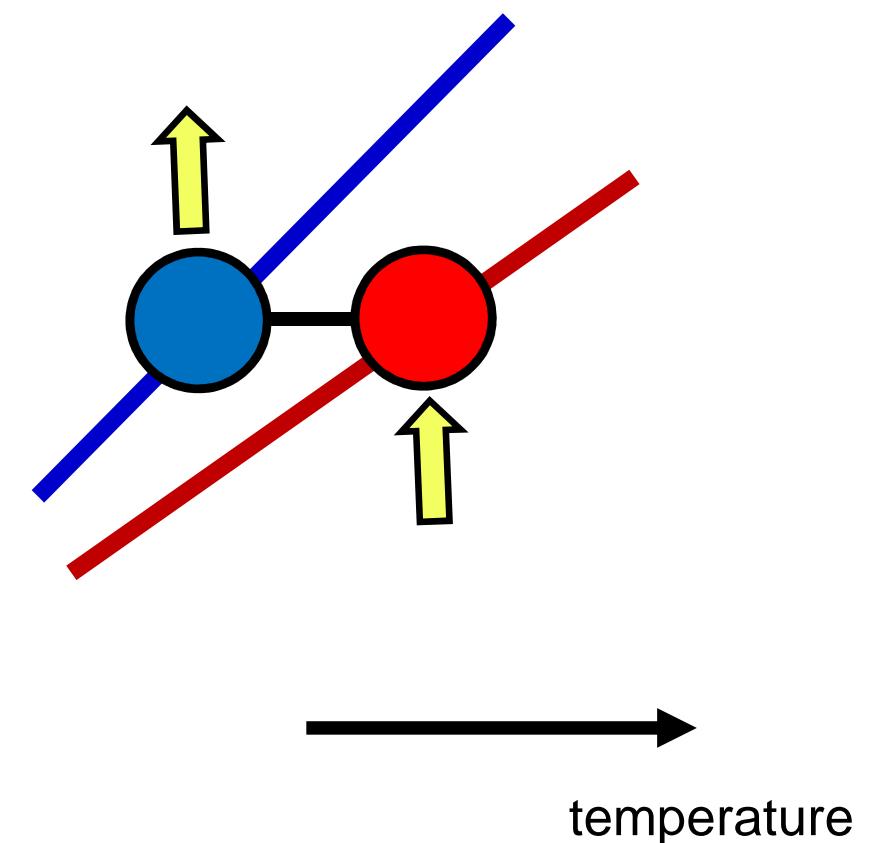


pressure

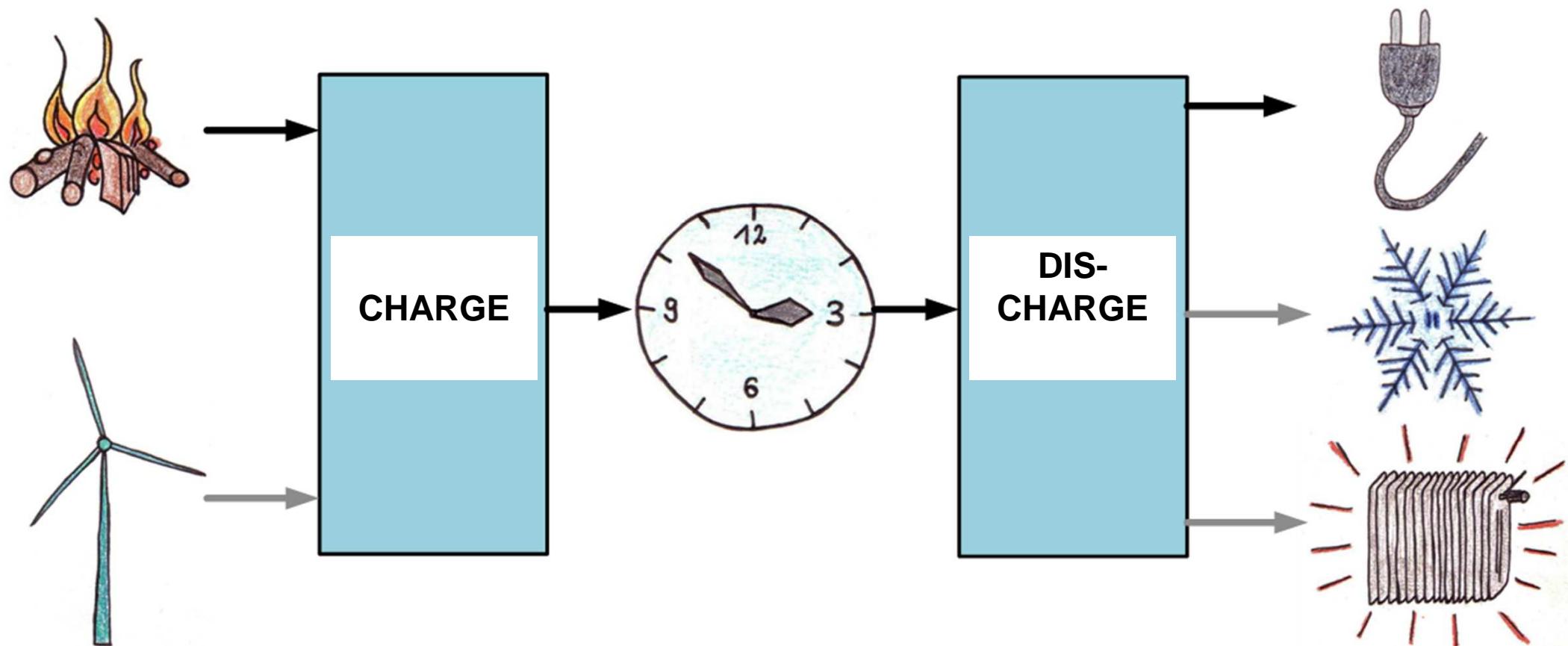


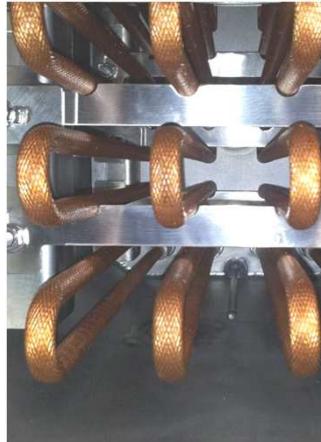
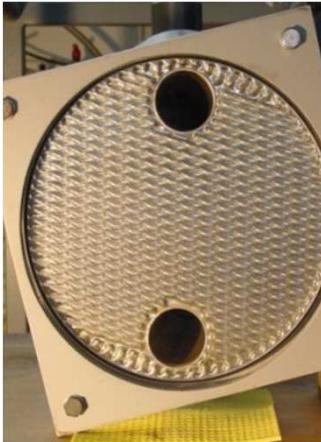
Discharging (work!)

pressure



Charging (Heat!)





1. Tailor isotherms.
2. Improve heat and mass transfer.
3. Do not let heat flows stay unused.

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