

Lecture 3: Models and measurements for thermal systems

Types of inverse problems

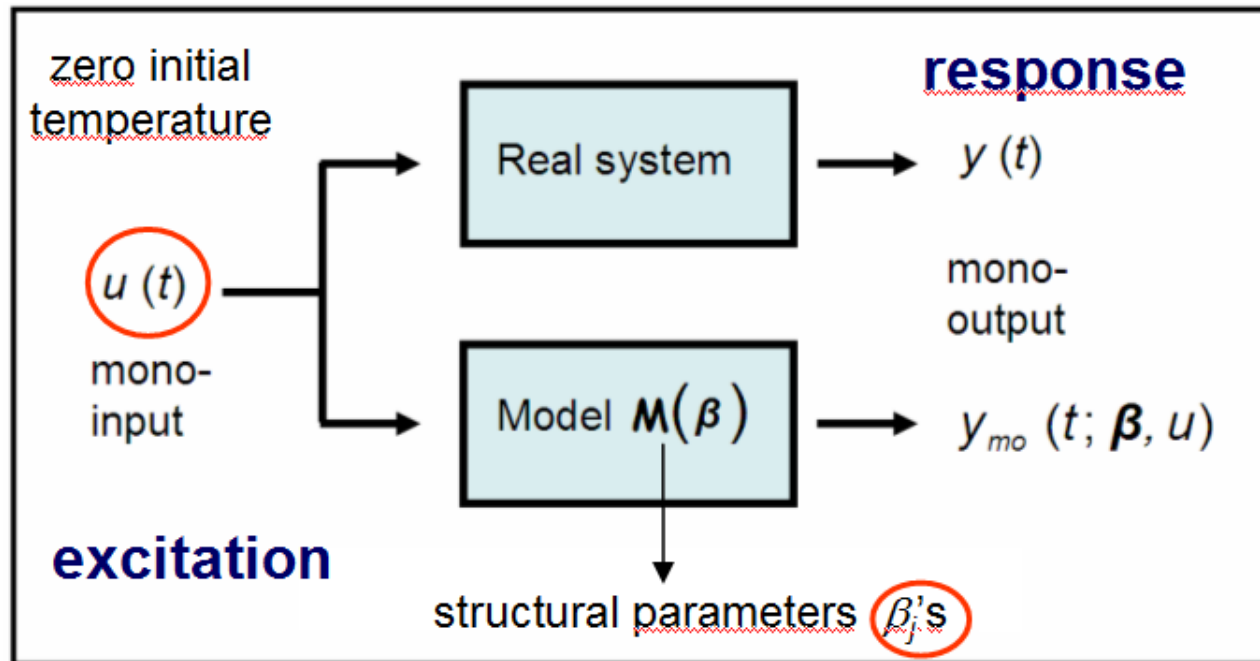
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1. Objectives, models & direct problems, internal/external representations
2. Parameterizing a function & parcimony principle
3. State-space representation, model terminology & structure, measurements
4. Different types of inverse problems, measurements & noise, bias
5. Physical model reduction

1. Objectives of a **model M** (in heat transfer)

First objective: Simulation of physical reality = **Direct Problem**

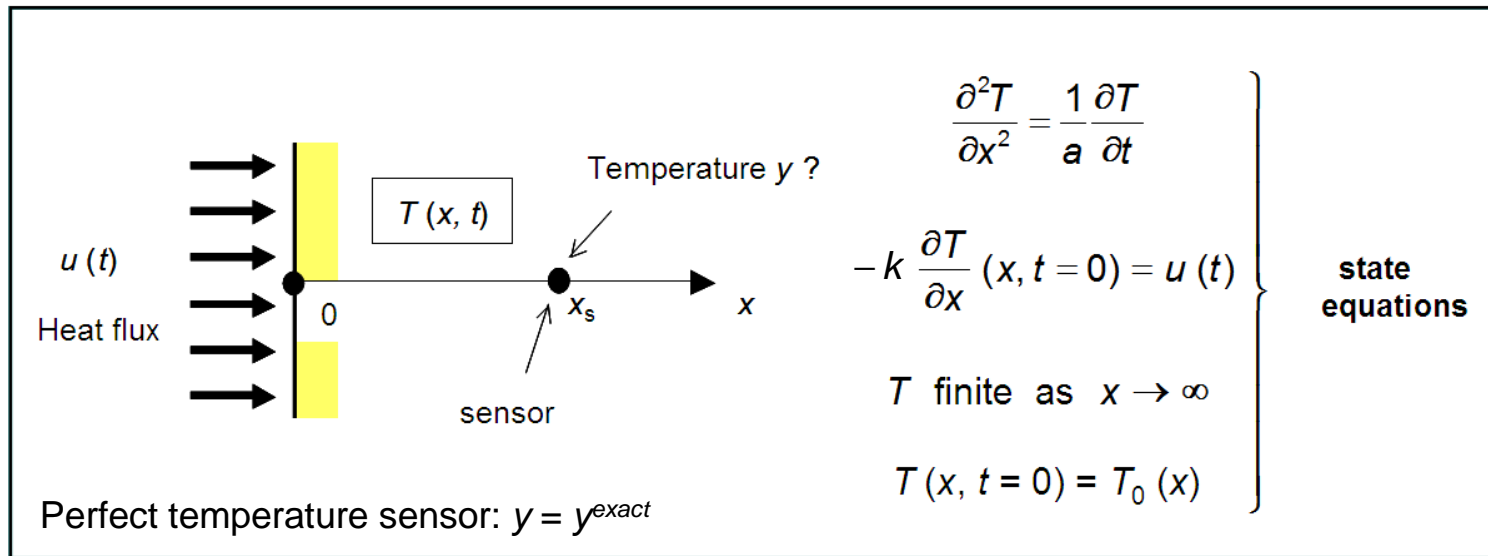


$u(t)$: heat source/flux, variable external temperature

y : measured temperature at given time t and given location

y_{mo} : modelled temperature at given time t and given location

Example: model of a semi infinite medium (1 input - 1 output)



Output equation: $y_{mo}(t) = T(x_s, t)$

Solution of direct problem: $y_{mo}(t) = y_{mo \text{ relax}}(t) + y_{mo \text{ forced}}(t)$

External representation: $y_{mo}(t) = \int_0^\infty G(x_s, x, t) T_0(x) dx + \int_0^t Z(t-\tau) u(\tau) d\tau$

Green's function

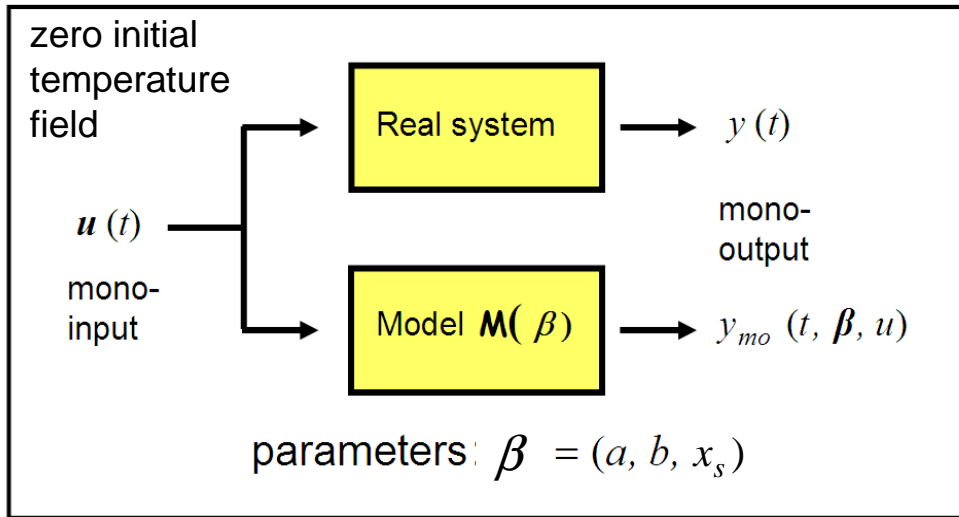
convolution product

$$G(x_s, x, t) = \frac{1}{2\sqrt{\pi a t}} \left[\exp\left(-\frac{(x_s - x)^2}{4 a t}\right) + \exp\left(-\frac{(x_s + x)^2}{4 a t}\right) \right]$$

$$Z(t) = \frac{1}{b\sqrt{\pi t}} \exp(-x_s^2/4 a t)$$

$$a = k / \rho c$$

$$b = \sqrt{k \rho c_3}$$



Thermal impedance:

$$Z(t) = \frac{1}{b \sqrt{\pi t}} \exp(-x_s^2 / 4 a t)$$

$$\bar{Z}(p) = \frac{1}{b \sqrt{p}} \exp(-x_s \sqrt{p/a})$$

p = Laplace parameter (s^{-1})

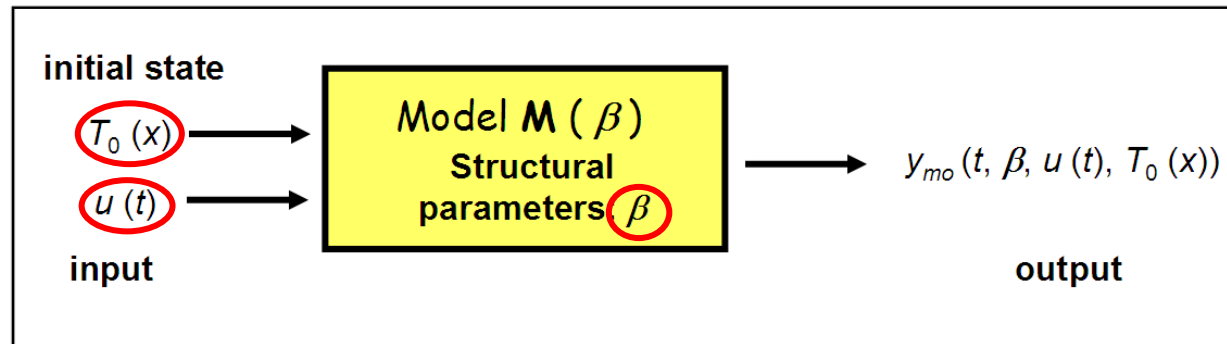
$$\bar{y}_{mo \text{ forced}}(p) = \bar{Z}(p) \bar{u}(p) \text{ with } \bar{f}(p) = \int_0^{\infty} f(t) \exp(-pt) dt$$

Parameter list $\beta = (a, b, x_s)$ \longrightarrow Parameter « vector » $\beta = \begin{bmatrix} a \\ b \\ x_s \end{bmatrix}$

$$a \text{ in } W m^{-2} - b \text{ in } J m^{-2} K^{-1} s^{-1/2} - x_s \text{ in } m \Rightarrow \|\beta\| = \left(a^2 + b^2 + x_s^2 \right)^{1/2}$$

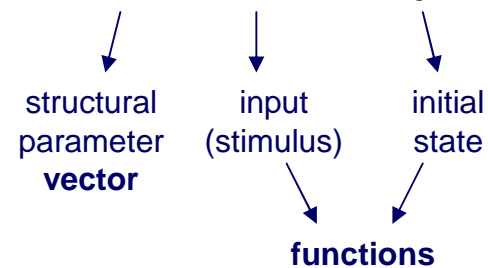
***M of the white box type (internal representation) :
internal parameters of physical nature***

Semi-infinite medium model: general case



List of data of Direct Problem: $X = \{ \boldsymbol{\beta}, u(\cdot), T_0(\cdot) \}$

$$\boldsymbol{\beta} = \begin{bmatrix} a \\ b \\ x_s \end{bmatrix}$$



2. Problem of function parameterizing & Parcimony principle

$$u(t) = \sum_{j=1}^{\infty} u_j f_j(t) \quad \text{with } \{f_1, f_2, \dots\}$$

function
on $[0, t_{sup}]$

basis of infinite number of functions: $[0, t_{sup}] \rightarrow \mathbf{R}$

$$\mathbf{u} = [u_1, u_2, \dots]^T \quad \mathbf{f}(t) = [f_1(t), f_2(t), \dots]^T \quad \Rightarrow \quad u(t) = \mathbf{u}^T \mathbf{f}(t)$$

column vectors with an infinite number of components

$$u(t) = \sum_{j=1}^{\infty} u_j f_j(t) \quad \Rightarrow \quad u_j = \langle u(t), f_j(t) \rangle$$

projection of $u(t)$ onto $f_j(t)$

in practice:

$$u_{\text{param}}(t) = \sum_{j=1}^{\overset{\text{truncation}}{\circlearrowleft n}} u_j f_j(t) \neq u(t) \quad \Rightarrow \quad u(t) \text{ replaced by } \mathbf{u} = \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_n \end{bmatrix}$$

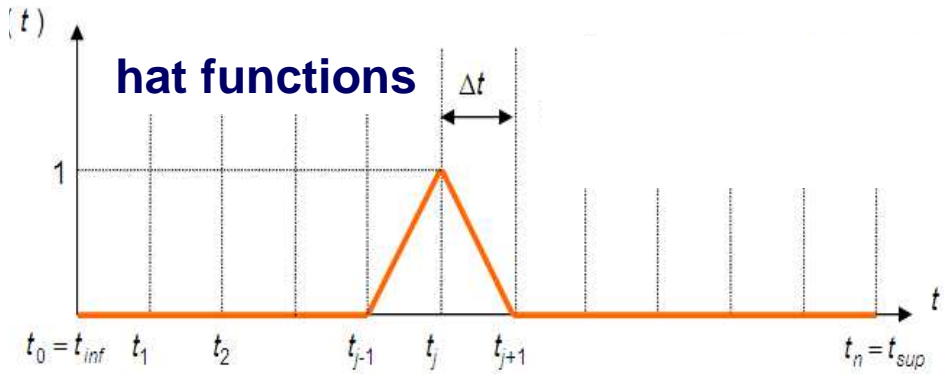
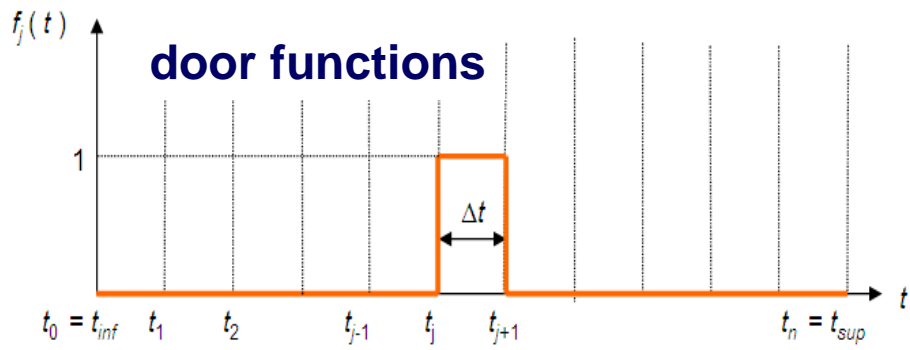
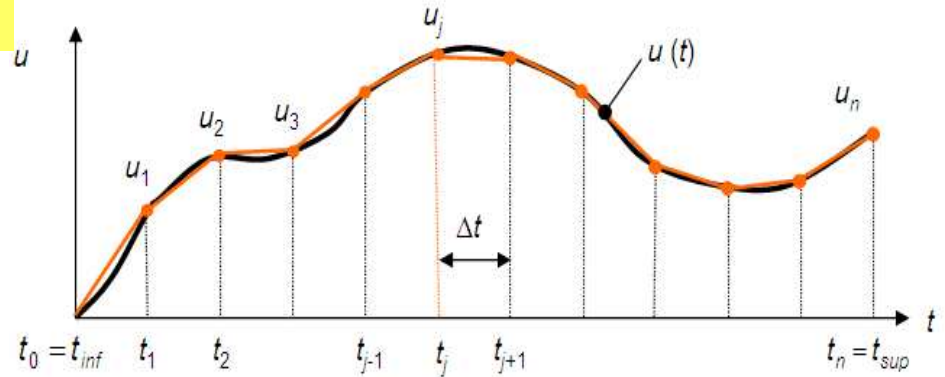
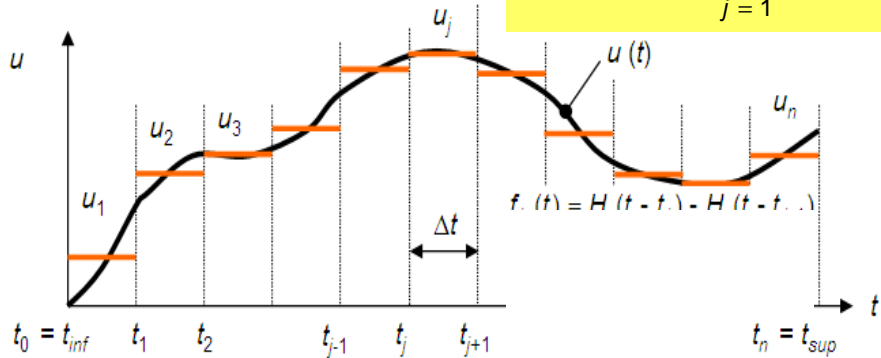
Good approximation:
high $n \Rightarrow$ large number of parameters

The model-builder has to choose 2 things:

- 1) functions f_j
- 2) their number n

Parameterization: 2 possible choices of *local* function basis

$$u_{\text{param}}(t) = \sum_{j=1}^n u_j f_j(t)$$



u_j : averaged value over an interval

u_j : local discretized value \rightarrow interpolated $u_{\text{param}}(t)$

• interesting for :

• interesting for :

$u(t)$ \rightarrow linear excitation

$u(T)$ $\beta(T)$ $\beta(P)$ $T_0(P)$
 \rightarrow non linear excitation temperature dependent thermophysical properties heterogeneous material non uniform initial temperature

Parameterization (continued)

Remarks

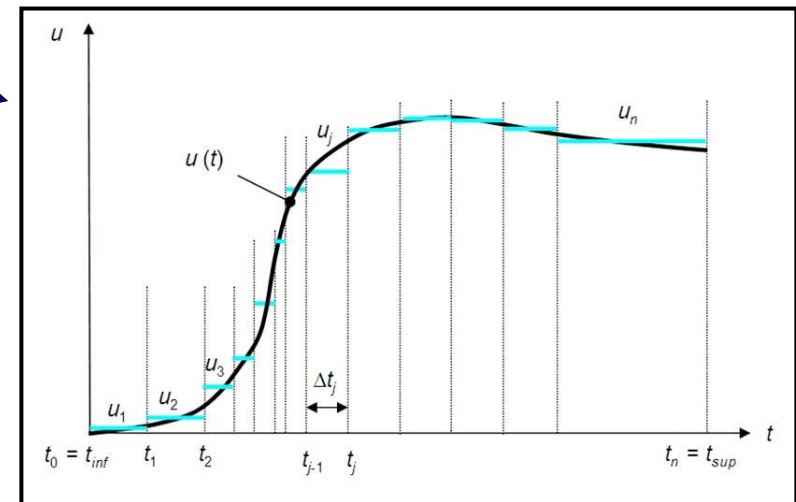
- *non-local* bases available: eigenfunctions f_j of the heat equation (method of separation of variables)

$$u_{\text{param}}(t) = \text{Fourier series, for example}$$

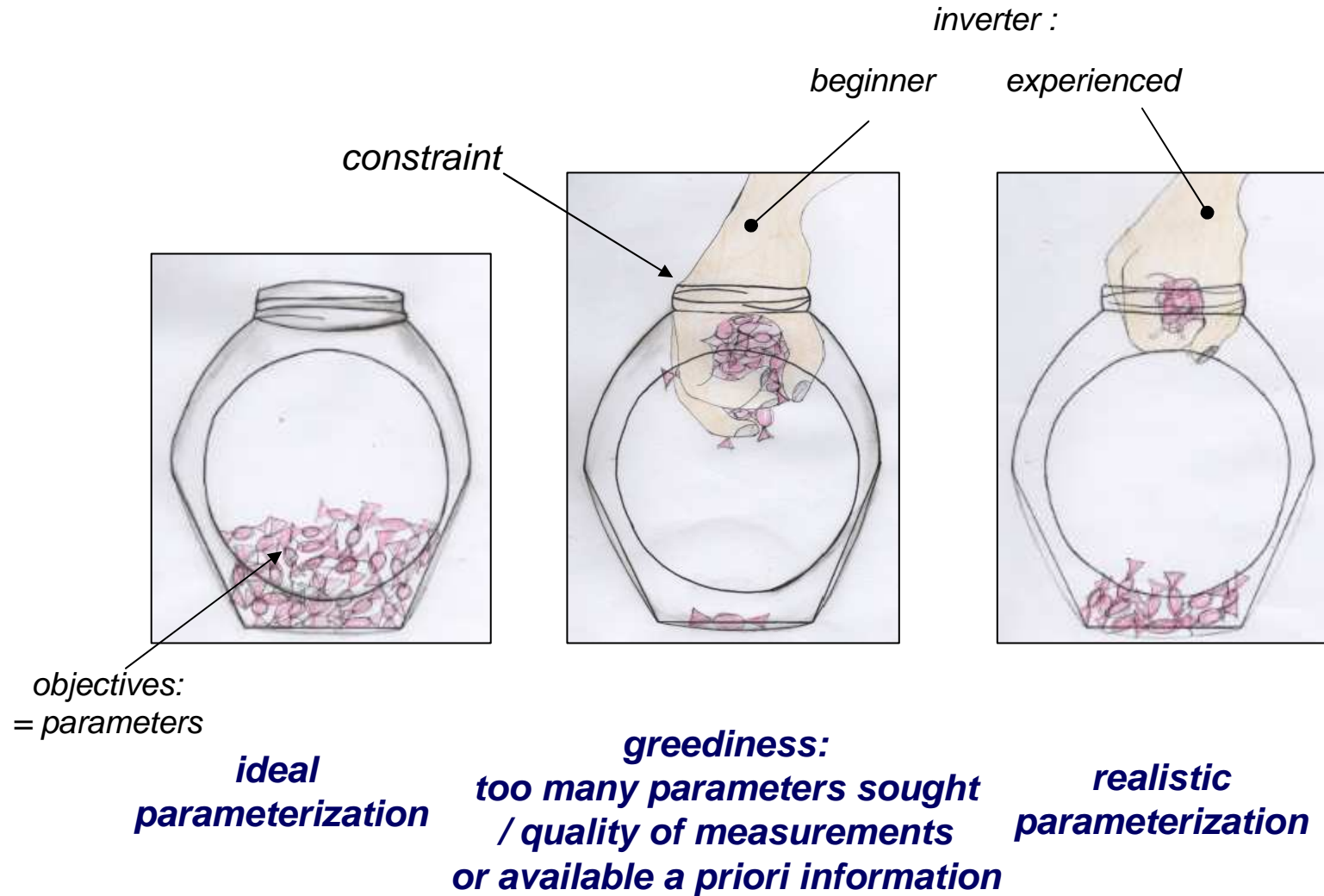
- orthogonal bases (with a unit function norm N_j) interesting: $\int_{t_{\text{inf}}}^{t_{\text{sup}}} f_j(t) f_k(t) dt = N_j \delta_{j,k}$
- non constant time step possible
- Extended parameter vector x gathering the Direct Problem data:

$$x = \{ \boldsymbol{\beta}, u(\cdot), T_0(P) \} \rightarrow x = \begin{bmatrix} \boldsymbol{\beta} \\ u \\ T_0 \end{bmatrix}$$

list parameterized functions



Parcimony principle: limitation of the number of parameters
(or of degrees of freedom) to be sought



3. State-space representation, model terminology & structure, measurements

$$\text{div} \left(\bar{\mathbf{k}} \text{grad } T \right) + q_{\text{vol}} = \rho c \frac{\partial T}{\partial t} \quad + \text{Boundary, interface and initial conditions}$$

W/m³

distributed parameter system

State of the system = continuous temperature field: $T(P, t) = T_P(t)$

Discretized state becoming = vector : $\mathbf{T}(t) = [T_1(t) \quad T_2(t) \quad \cdots \quad T_N(t)]$

in a N dimension space (number of nodes)

lumped parameter system:

$$\frac{d\mathbf{T}}{dt} = \mathbf{E}(t, \mathbf{T}, \mathbf{U}) \quad \text{with} \quad \mathbf{T}(t = t_0) = \mathbf{T}_0$$

Linear heat source (excitation): $q_{\text{vol}}(P, t) \longrightarrow \mathbf{U}(t) = [u_1(t), u_2(t) \cdots u_p(t)]^T$

p excited nodes

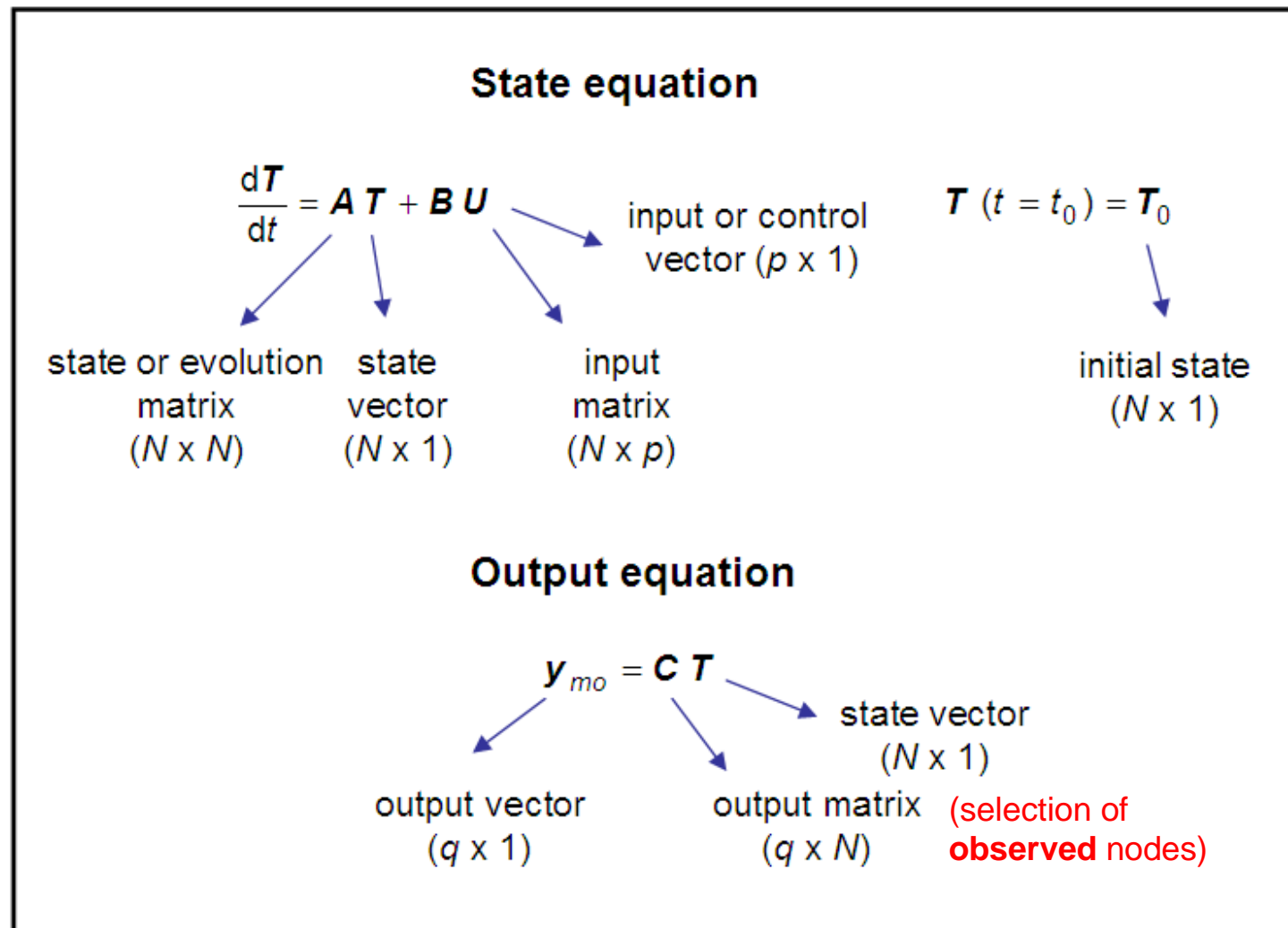
Non-linear heat source: $q_{\text{vol}}(T(P, t)) \longrightarrow \mathbf{U}(t) = [u(T_1(t)), u(T_2(t)) \cdots u(T_p(t))]^T$

p excited nodes

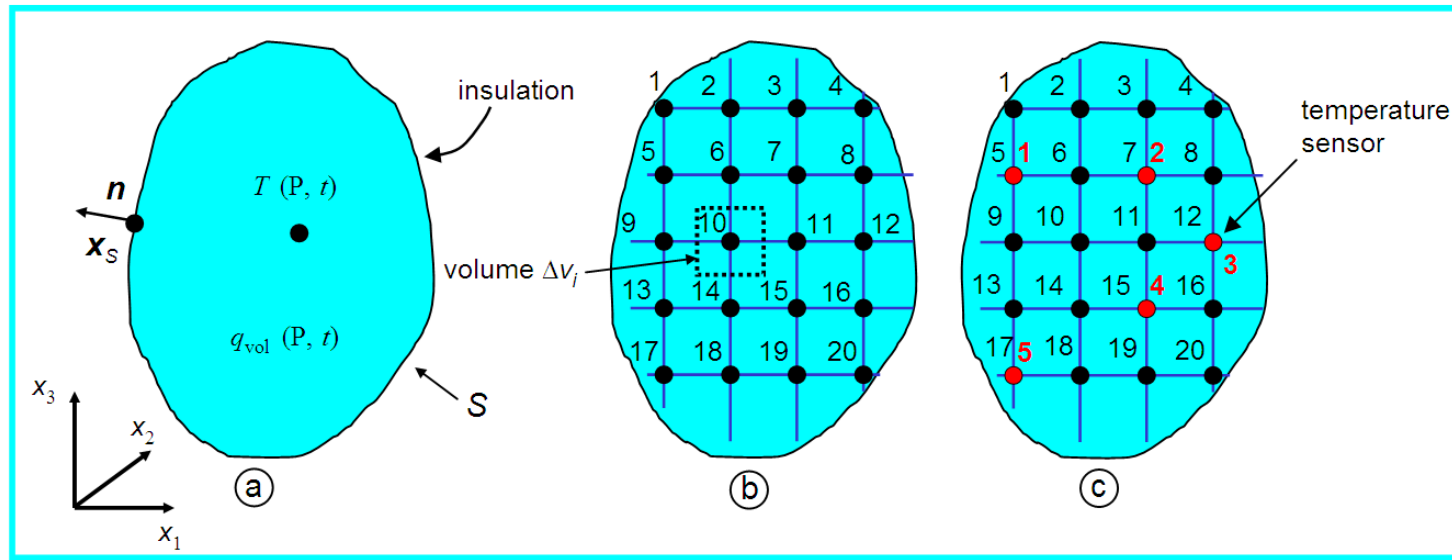
State-space representation (continued)

Case of a linear heat source with temperature independent thermophysical properties and coefficients

$$\mathbf{E}(t, \mathbf{T}, \mathbf{U}) = \mathbf{A}\mathbf{T} + \mathbf{B}\mathbf{U} \quad \text{with } \mathbf{A} \text{ and } \mathbf{B}: \text{ constant matrices}$$



Output equation: detailed



$$\mathbf{y}_{mo}(t) = \begin{bmatrix} y_{mo1}(t) \\ y_{mo2}(t) \\ y_{mo3}(t) \\ y_{mo4}(t) \\ y_{mo5}(t) \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 & 0 & 1 & \dots & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & \dots & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & \dots & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & \dots & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & \dots & 1 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} T_1(t) \\ T_2(t) \\ T_3(t) \\ T_4(t) \\ T_5(t) \\ \vdots \\ T_{17}(t) \\ T_{18}(t) \\ T_{19}(t) \\ T_{20}(t) \end{bmatrix}$$

$q = 5$ observed temperatures
(output)

$N = 20$ nodes

Linear state equation : $\frac{dT}{dt} = \mathbf{A} T + \mathbf{B} U$

Explicit solution for temperature field:

$$T(t) = \exp(\mathbf{A}(t - t_0)) T_0 + \int_{t_0}^t \exp(\mathbf{A}(t - \tau)) \mathbf{B} U(\tau) d\tau$$

and for model output:

$$y_{mo}(t) = \mathbf{C} \exp(\mathbf{A}(t - t_0)) T_0 + \mathbf{C} \int_{t_0}^t \exp(\mathbf{A}(t - \tau)) \mathbf{B} U(\tau) d\tau$$

Relaxation of initial state

forced (convolution) response

Remarks:

- advection case possible (dispersion in porous medium, one-temperature model):

$$\text{div} \left(\bar{\lambda} \text{grad } T \right) - \rho c_f \mathbf{v} \cdot \text{grad } T + q_{\text{vol}} = \rho c \frac{\partial T}{\partial t} + \text{Boundary, interface and initial conditions}$$

- coupled modes transfer : radiation in semitransparent absorbing medium
(Heat equation + radiative transfer equation) \Rightarrow composite state \mathbf{X} :

$$\mathbf{X}(t) = \begin{bmatrix} T(t) \\ I(t) \end{bmatrix} \begin{array}{l} \longrightarrow \text{Discretized temperature field: position} \\ \longrightarrow \text{Discretized intensity field: wavelength, direction, position} \end{array}$$

- steady state case (linear) :

$$\frac{dT}{dt} = \mathbf{A}T + \mathbf{B}U = 0$$

$$T = -\mathbf{A}^{-1} \mathbf{B}U \Rightarrow \mathbf{y}_{mo} = -\mathbf{C} \mathbf{A}^{-1} \mathbf{B}U$$

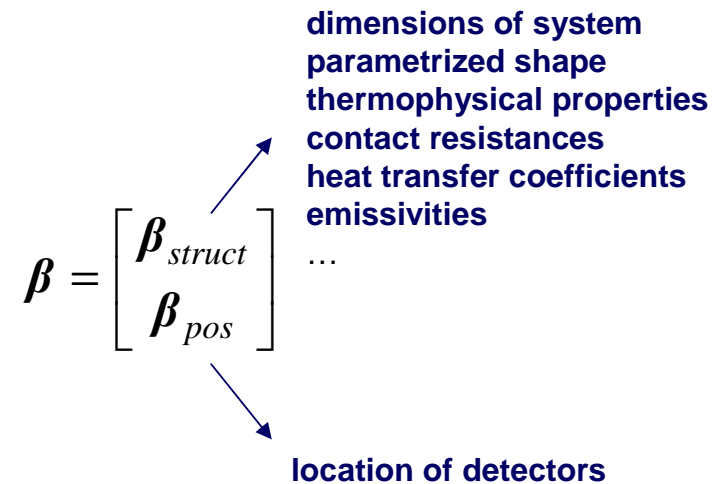
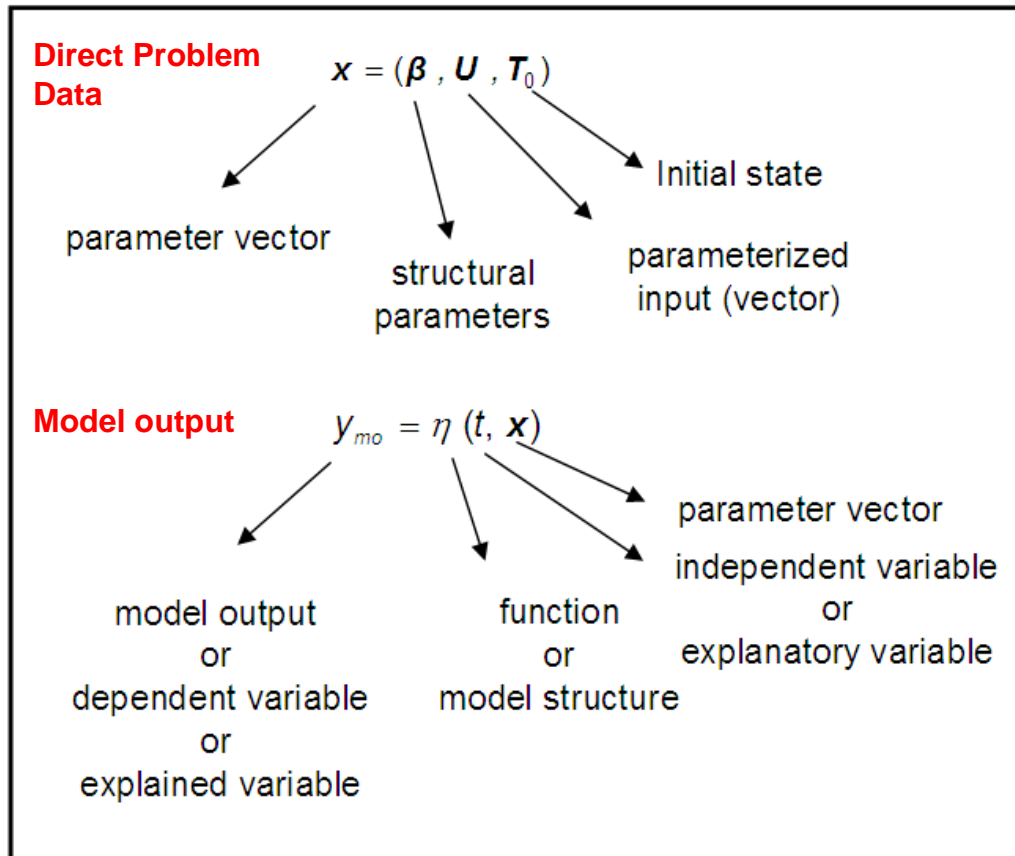
Model terminology and structure

	Before parameterization	After parameterization
• single output:	$y_{mo} = \eta(t, \mathbf{x})$ <p>↓ scalar ↓ list</p>	$y_{mo} = \eta(t, \mathbf{x})$ <p>↓ scalar ↓ extended parameter vector</p>
• multiple output:	$\mathbf{y}_{mo} = \boldsymbol{\eta}(t, \mathbf{x})$ <p>↓ column vector ↓ list</p>	$\mathbf{y}_{mo} = \boldsymbol{\eta}(t, \mathbf{x})$ <p>↓ column vector ↓ extended parameter vector</p>
• data	$\mathbf{x} = \{ \boldsymbol{\beta}, u(\cdot), T_0(P) \}$	$\mathbf{x} = \begin{bmatrix} \boldsymbol{\beta} \\ u \\ T_0 \end{bmatrix}$

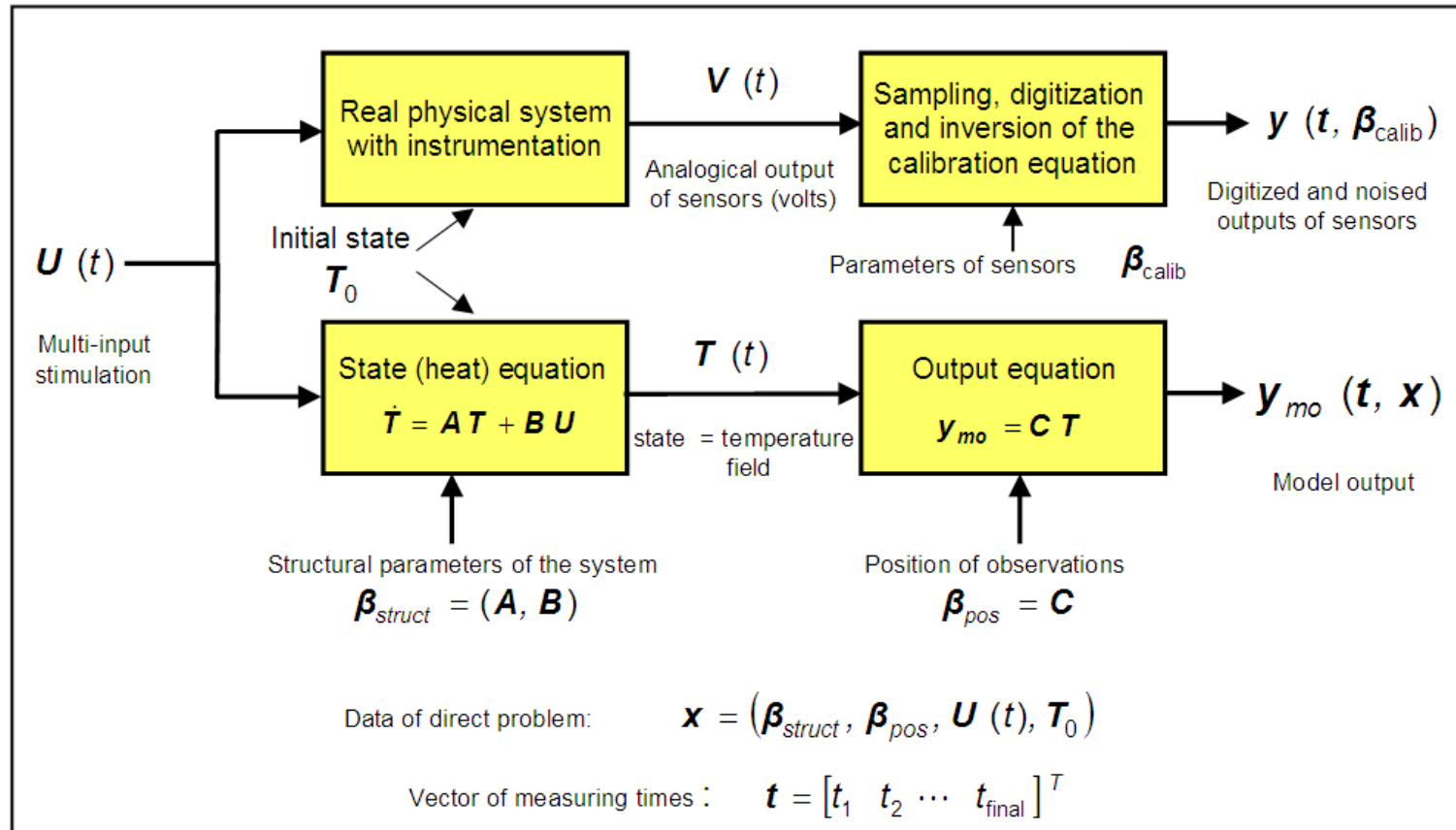
$\eta(t, \cdot)$ or $\boldsymbol{\eta}(t, \cdot)$: scalar or vector function
 x or \mathbf{x} : corresponding data (list/vector)

= structure of the model

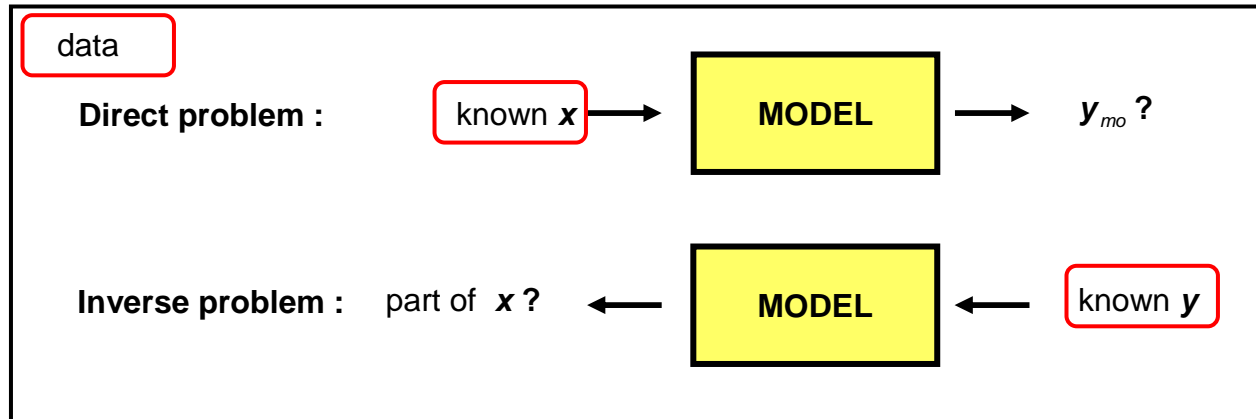
Structure of a parameterized model in heat transfer



Comparison between measurements and state model



Direct and inverse problems



Objective of inverse problem: finding a part \mathbf{x}_r of \mathbf{x} , using additional information (output \mathbf{y} or something else)

Extended parameter vector:
$$\mathbf{x} = \begin{bmatrix} \boldsymbol{\beta}_{struct} \\ \boldsymbol{\beta}_{pos} \\ \mathbf{U}(t) \\ \mathbf{T}_0 \end{bmatrix}$$

$$\mathbf{x} = \begin{bmatrix} \mathbf{x}_r \\ \mathbf{x}_c \end{bmatrix} \begin{array}{l} \longrightarrow \text{sought (researched) parameters} \\ \longrightarrow \text{complementary part: known} \end{array}$$

4. Different types of inverse problems in heat transfer, measurements & noise, bias

- **inverse measurement problems** → additional information stems from output signal \mathbf{y} of sensors

- **control problems**

- additional information = desired (target) values of state \mathbf{T} or output \mathbf{y}

- sought quantity = excitation \mathbf{U} , initial state \mathbf{T}_0 or velocity/flowrate in β

- **system identification problems** = model construction

- **model reduction**

- additional information = output of detailed model $\eta_{\text{det}}(t; \mathbf{x}_{\text{det}})$

- sought quantity = structure + parameter vector of a reduced model $\eta_{\text{det}}(t; \mathbf{x}_{\text{det}}) \approx \eta_{\text{red}}(t; \mathbf{x}_{\text{red}})$

$$\text{with: } \mathbf{x}_{\text{det}} = [\boldsymbol{\beta}_{\text{det}} \quad \mathbf{U}_{\text{det}} \quad \mathbf{T}_{0 \text{ det}}]^T \quad \text{and} \quad \mathbf{x}_{\text{red}} = [\boldsymbol{\beta}_{\text{red}} \quad \mathbf{U}_{\text{red}} \quad \mathbf{T}_{0 \text{ red}}]^T$$

1) <i>mathematical reduction</i> :	$u_{\text{red}}(P, t) = u_{\text{det}}(P, t)$	\Rightarrow	$\mathbf{U}_{\text{red}} = \mathbf{U}_{\text{det}}$
GREY BOX type	$T_{0 \text{ red}}(P, t) = T_{0 \text{ det}}(P, t)$	\Rightarrow	$\mathbf{T}_{0 \text{ det}} = \mathbf{T}_{0 \text{ red}}$

2) <i>physical reduction</i> :	$u_{\text{red}}(P, t) \approx u_{\text{det}}(P, t)$	\Rightarrow	$\mathbf{U}_{\text{red}} = f_U(\mathbf{U}_{\text{det}})$
WHITE BOX type	$T_{0 \text{ red}}(P, t) \approx T_{0 \text{ det}}(P, t)$	\Rightarrow	$\mathbf{T}_{0 \text{ red}} = f_{T0}(\mathbf{T}_{0 \text{ det}})$

In both cases: $\boldsymbol{\beta}_{\text{red}} = f_{\beta}(\boldsymbol{\beta}_{\text{det}})$ \longrightarrow *explicit for physical reduction*

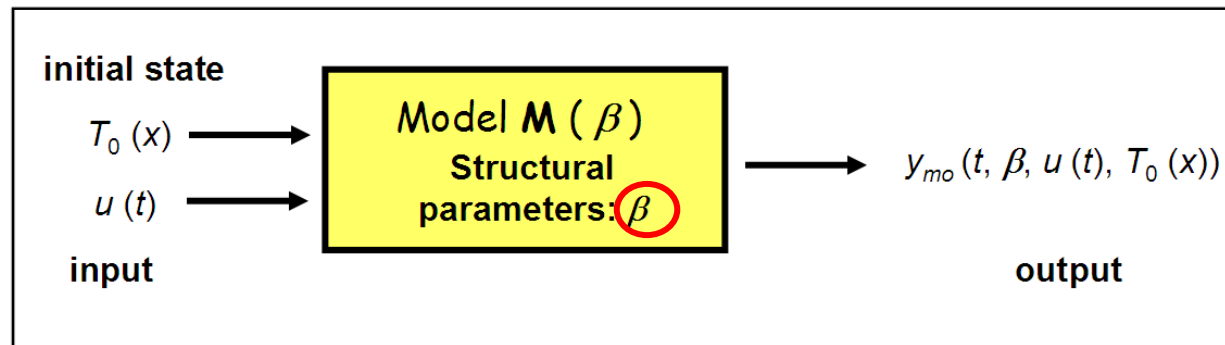
- **experimental model identification** (belongs to inverse measurements problem class)

→ **additional information** = y , U and T_0 are measured, or supposed to be known

→ **sought quantity** = parameter vector β for a model of given structure

3 types of identified models:

- *white box* type, based on first principles: physical meaning for β
- *black box* type: general structure, no physical meaning for β (neural networks)
- *grey box* type : in between, physical structure, no physical meaning for β



- **optimal design problems**

→ **additional information**: quality criterion to satisfy

→ **sought quantity** = parameter vector β with constraints, for a model of given structure

Inverse measurement problems in heat transfer

Extended parameter vector: $\mathbf{x} = \begin{bmatrix} \boldsymbol{\beta}_{struct} \\ \boldsymbol{\beta}_{pos} \\ \mathbf{U}(t) \\ T_0 \end{bmatrix} = \begin{bmatrix} \mathbf{x}_r \\ \mathbf{x}_c \end{bmatrix}$

→ sought (researched) parameters
→ complementary part: known

Measurements $y(t)$ available on $[t_0, t_{final}]$ interval

a) Inverse problems of **structural parameters estimation** : $\mathbf{x}_r \equiv \boldsymbol{\beta}_r$

- example 1: thermophysical property « measurement » : $\mathbf{x}_r = k \text{ or } \rho c \text{ or } a... + h!$

- example 2: calibration of a sensor/acquisition chain: $V_{mo}(T, \boldsymbol{\beta}_{calib})$

b) Inverse **input problems** : $\mathbf{x}_r \equiv u(P, t)$

- example: “inverse heat conduction” = wall heat flux “measurement”

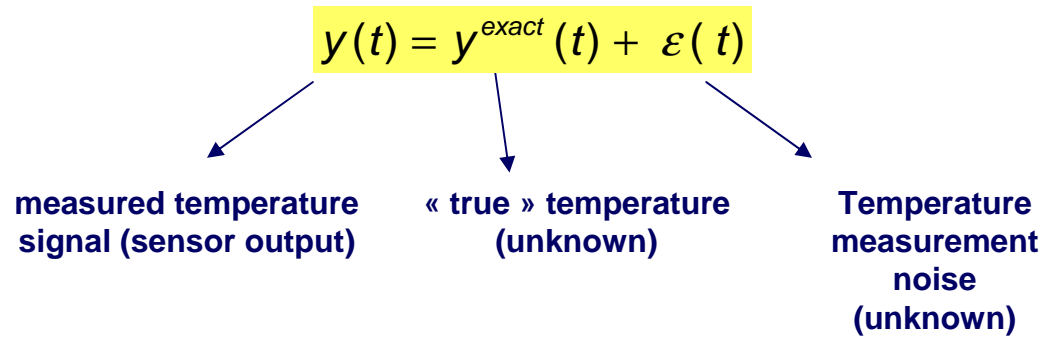
c) Inverse **initial state problems** : $\mathbf{x}_r \equiv T_0(P)$

d) Inverse **shape reconstruction** problems

e) Inverse problems of **optimal design/control**

(of a characterization experiment, for example)

Measurement and noise



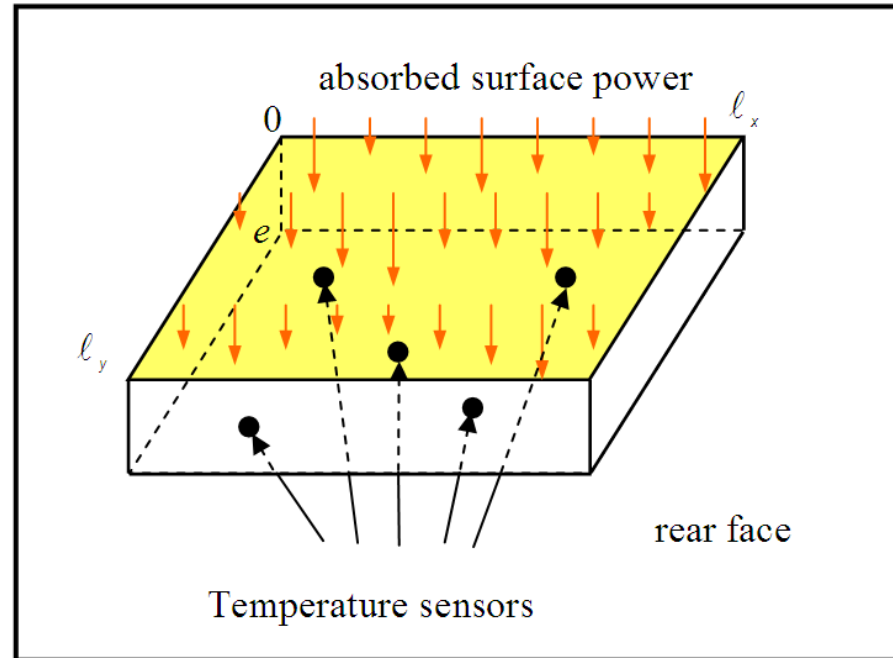
- only discrete values available $y_i = y(t_i) \Rightarrow \varepsilon_i = \varepsilon(t_i)$

- **(implicit) assumptions:**

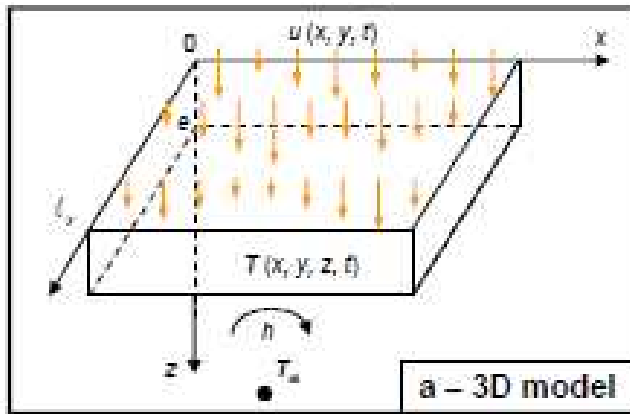
→ unbiased model: $y^{exact}(t) = y_{mo}(t, \mathbf{x}^{exact})$

→ unbiased noised: $E(\varepsilon_i) = 0$

5. Physical model reduction on an example



- homogeneous rectangular slab, thickness e , lengths
- thermal diffusivity and conductivity a and k , volumetric heat $\rho c = k/a$
- 4 lateral sides insulated, h heat exchange coefficient over rear face
- uniform initial temperature T_0
- two dimensional heat flux absorption over front face
- q temperature sensors inside the slab



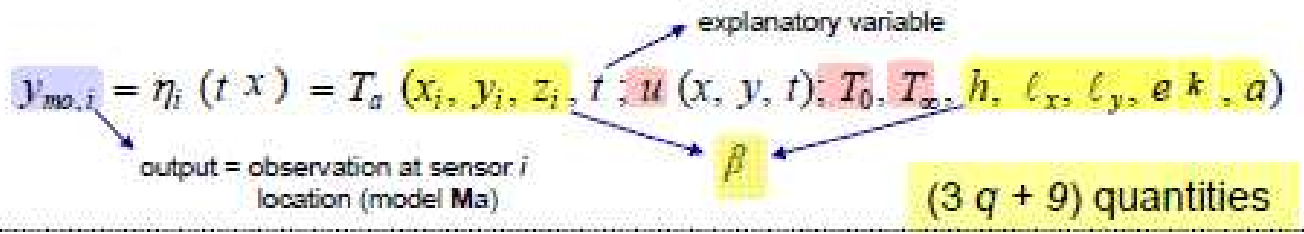
$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} = \frac{1}{a} \frac{\partial T}{\partial t}$$

input quantities

$$T = T_0 \quad \text{for} \quad t = 0$$

$$\frac{\partial T}{\partial x} = 0 \quad \text{at} \quad x = 0, l_x; \quad \frac{\partial T}{\partial y} = 0 \quad \text{at} \quad y = 0, l_y$$

$$-k \frac{\partial T}{\partial z} = u(x, y, t) \quad \text{at} \quad z = 0; \quad -k \frac{\partial T}{\partial z} = h(T - T_\infty) \quad \text{at} \quad z = e$$



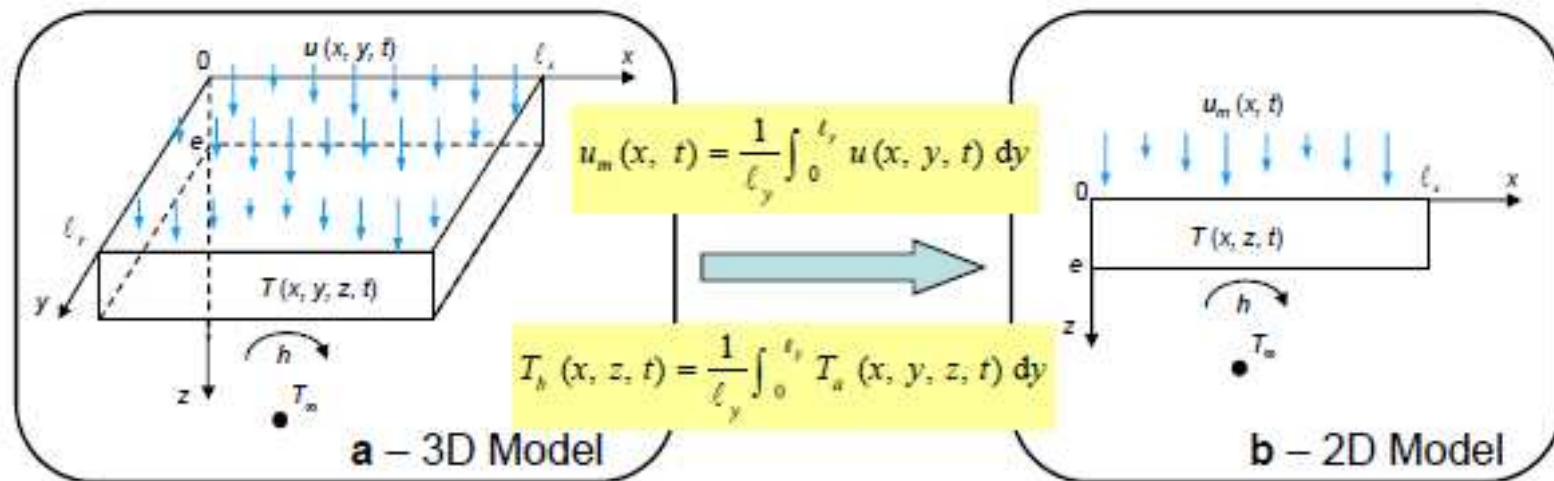
→ Dimensionless form: $T^* = (T - T_\infty) / \Delta T$

$$y_{mo,i} = \eta^*(t, x^*) = \Delta T \cdot T^*(x_i^*, y_i^*, z_i^*, t / \tau_{diff}, R, u(x, y, t) / \Delta T, H, l_x^*, l_y^*) + T_\infty$$

$$\Delta T = T_0 - T_\infty \quad x_i^* = x_i / e \quad y_i^* = y_i / e \quad z_i^* = z_i / e \quad l_x^* = l_x / e \quad l_y^* = l_y / e \quad R = e / k \quad \tau_{diff} = e^2 / a \quad H = h e / k$$

parameter list: $x^* = (\beta^*, u, \Delta T, T_\infty)$ (3q + 8) quantities or (+9)

structura/positions parameter vector: $\beta^* = ((x_i^*, y_i^*, z_i^*), \text{for } i=1 \text{ to } q), \tau_{diff}^*, R, H, l_x^*, l_y^*$



$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial z^2} = \frac{1}{a} \frac{\partial T}{\partial t} \quad T = T_0 \quad \text{at} \quad t = 0$$

$$-k \frac{\partial T}{\partial z} = u_m(x, t) \quad \text{at} \quad z = 0; \quad -k \frac{\partial T}{\partial z} = h(T - T_\infty) \quad \text{at} \quad z = e$$

Model b

Observations :

$$y_{mod,i}(t_k) = T_b(x_i, z_i, t_k)$$

$$x = \{\beta, u_m, \Delta T, T_\infty\} \quad (2q + 7) \text{ quantities}$$

$$\beta = ((x_i^*, z_i^*, \text{ for } i = 1 \text{ à } q), \tau_{diff}, R, H, \ell_x^*)$$

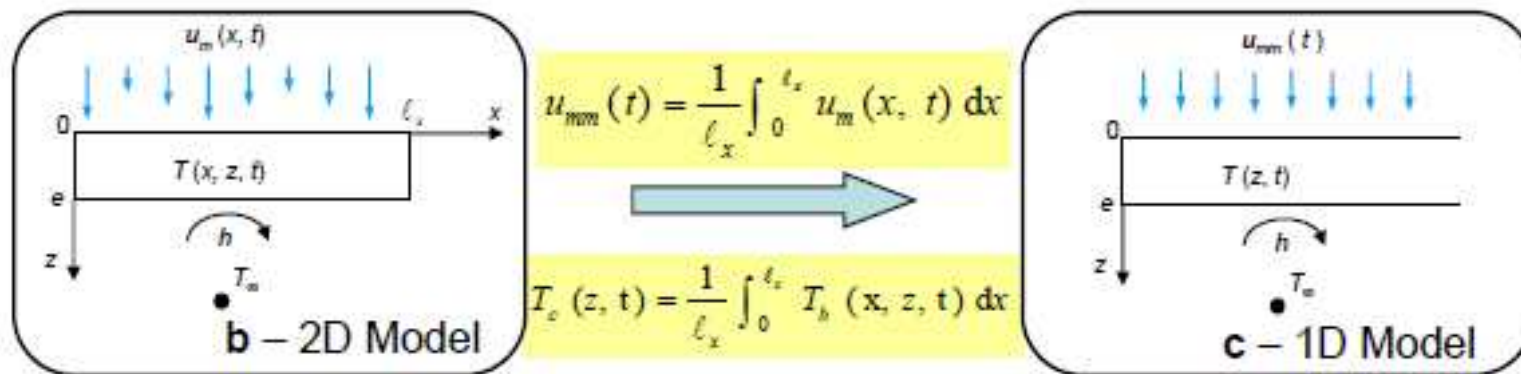
Measurements

$$y_i(t_k) = T_k^{exp}(x_i, \cancel{y_i}, z_i)$$

$$y_i(t_k) = \frac{1}{n_i} \sum_{j=1}^{n_i} T_k^{exp}(x^m, y^j = y_i)$$

averaging in y-direction
 $\rightarrow x-t$ temperature signal

y location disappears



$$\frac{\partial^2 T}{\partial z^2} = \frac{1}{a} \frac{\partial T}{\partial t}$$

$$\beta = ((x_i^*, z_i^*, \text{ for } i = 1 \text{ à } q), \tau_{diff}, R, H, \ell_x^*)$$

$$T = T_0 \text{ for } t = 0$$

$$-k \frac{\partial T}{\partial z} = u_{mm}(t) \text{ at } z = 0; \quad -k \frac{\partial T}{\partial z} = h(T - T_\infty) \text{ at } z = e$$

Model c

$$y_{mod,i}(t_k) = T_c(z_i, t_k)$$

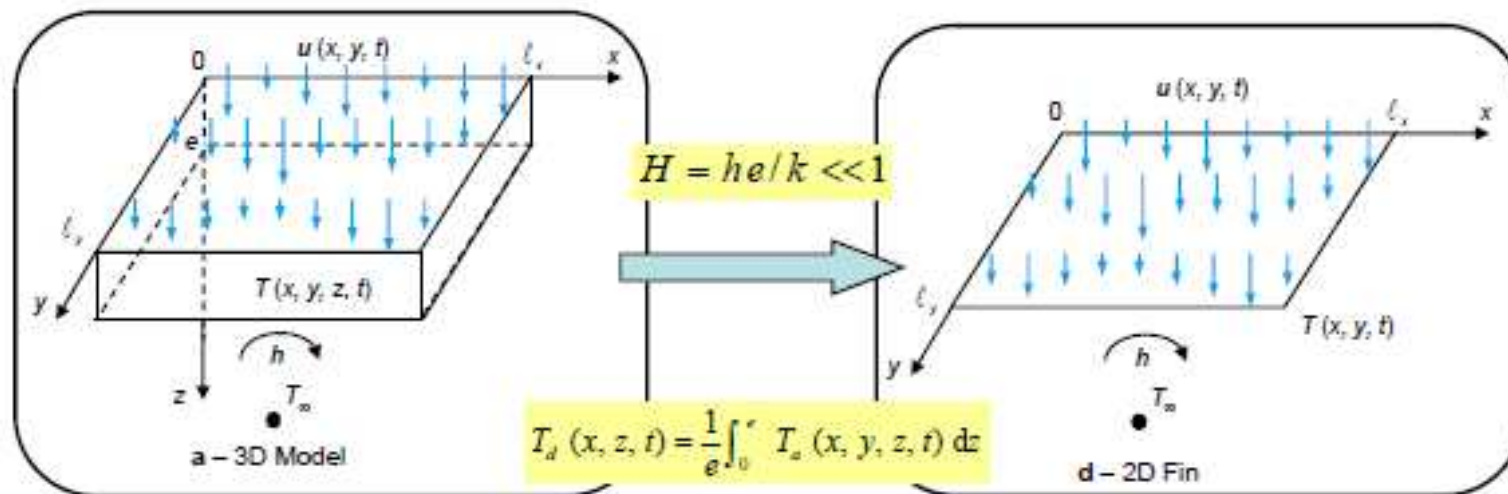
$$x = \{ \beta, u_{mm}, \Delta T, T_\infty \} \quad (q + 6) \text{ quantities}$$

$$\beta = (z_i^*, \text{ for } i = 1 \text{ à } q), \tau_{diff}, R, H$$

Measurements

$$y(t_k) = \frac{1}{q} \sum_{i=1}^q T_k^{exp}(x_i, y_i)$$

averaging in x - and y - direction
 $\rightarrow t$ temperature signal



2D Fin equation:
$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} - \frac{h(T - T_\infty)}{k e} + \frac{u(x, y, t)}{k} = \frac{1}{a} \frac{\partial T}{\partial t}$$

$$T = T_0 \quad \text{at} \quad t = 0 \quad \frac{\partial T}{\partial x} = 0 \quad \text{in} \quad x = 0, l_x; \quad \frac{\partial T}{\partial y} = 0 \quad \text{in} \quad y = 0, l_y$$

Model d

$$y_{mod,i}(t_k) = T_d(x_i, y_i, t_k)$$

$$x = \{ \beta, u_m, \Delta T, T_\infty \} \quad (2q + 8) \text{ quantities}$$

$$\beta = ((x_i^*, y_i^*, \text{ for } i = 1 \text{ à } q), \tau_{diff}, R, H, l_x^*, l_y^*)$$

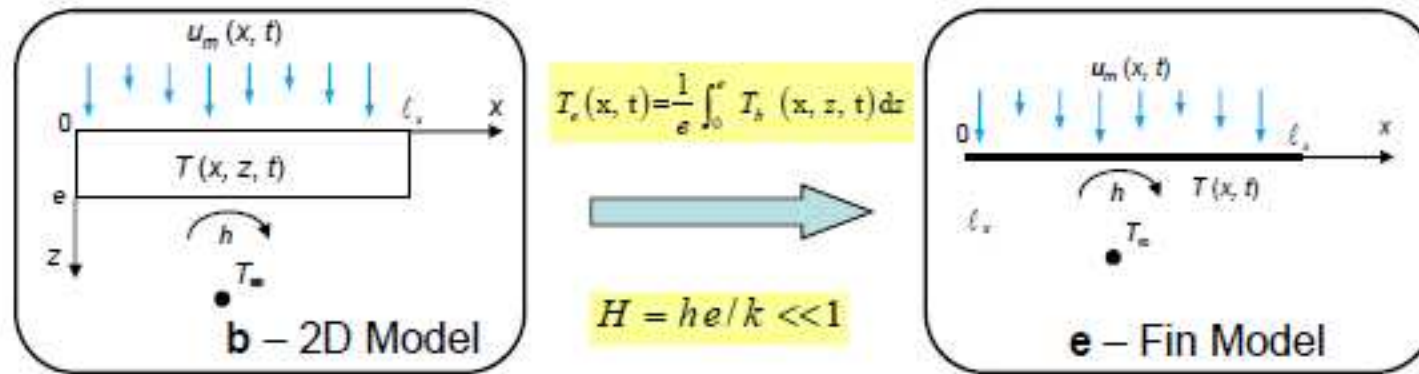
Measurements

$$y_i(t_k) = T_k^{exp}(x_i, y_i, \times)$$

or

$$y_i(t_k) = \frac{1}{n_i} \sum_{p=1}^{n_i} T^{exp}(x_i, y_i, z^p = z_i)$$

z_i disappears



1D Fin equation:
$$\frac{\partial^2 T}{\partial x^2} - \frac{h(T - T_\infty)}{k e} + \frac{u_m(x, t)}{k e} = \frac{1}{a} \frac{\partial T}{\partial t}$$

$T = T_0 \quad \text{at} \quad t = 0 \quad \frac{\partial T}{\partial x} = 0 \quad \text{in} \quad x = 0, \ell_x$

Model e

$y_{mod}(t_k) = T_c(x_i, z_i, t_k)$

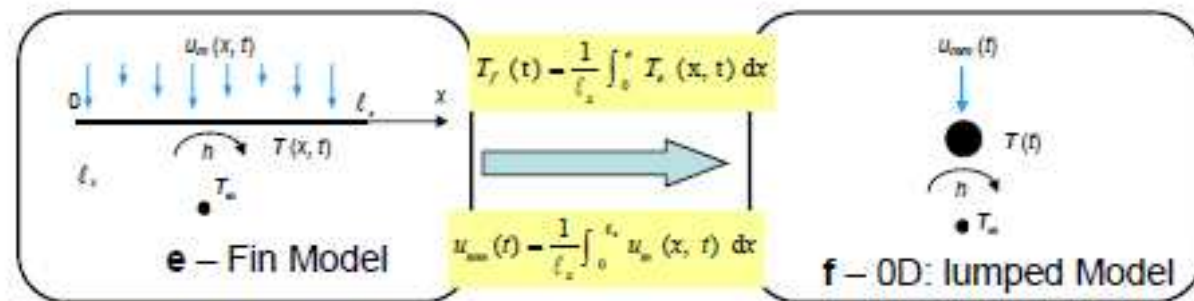
$x = \{\beta, u_m, \Delta T, T_\infty\}$ (q + 7) quantities

$\beta = (x_i^*, \cancel{x_i^*}, \text{for } i = 1 \text{ à } q), \tau_{diff}, R, H)$

Measurements

$y_i(t_k) = \frac{1}{n_i} \sum_{j=1}^{n_i} T_k^{exp}(x^m, y^j = y_i)$

averaging in y-direction
 → x - t temperature signal



$$H_x, H_y, H \ll 1$$

$$H_x = h \ell_x / k$$

$$H_y = h \ell_y / k$$

$$H = h e / k$$

Lumped capacitance model:

$$\rho c e \frac{dT}{dt} + h (T - T_\infty) = u_{mm}(t)$$

$$T = T_0 \quad \text{at} \quad t = 0$$

$$\tau = \rho c e / h = \tau_{diff} / H \quad G = 1 / h$$

Model f bulk temperature observation

$$T = T_\infty + \Delta T \exp(-t/\tau) + \frac{G}{\tau} \int_0^t u_{mm}(t') \exp\left(-\frac{t-t'}{\tau}\right) dt'$$

$$x = (\beta, u_{mm}, \Delta T, T_\infty) \quad \Delta T = T_0 - T_\infty$$

$$\beta = (\tau, G) \quad \text{5 independent variables}$$

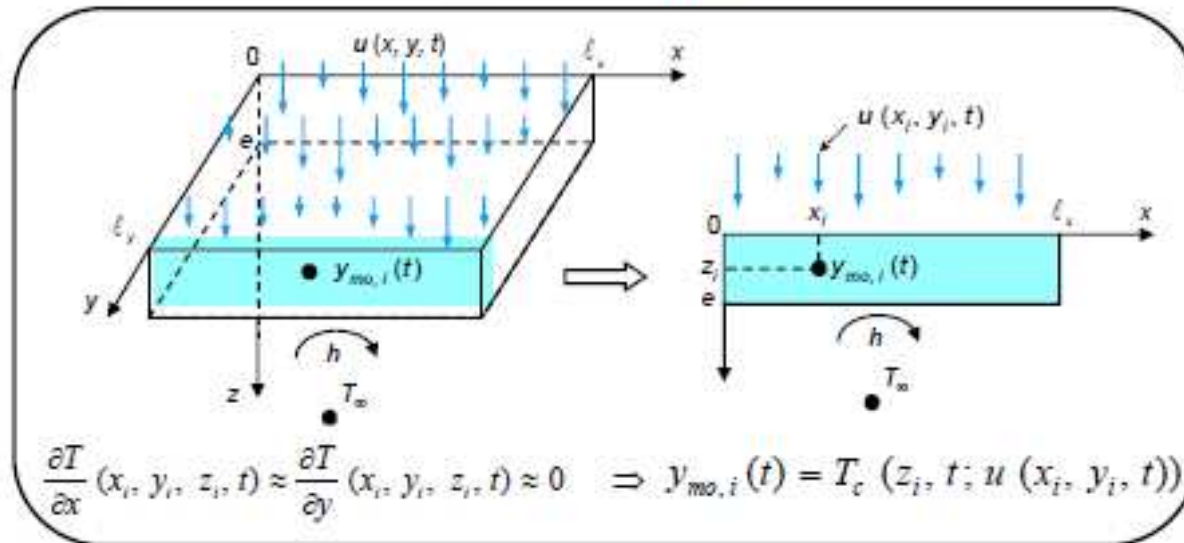
Measurements

$$y(t_k) = \frac{1}{q} \sum_{i=1}^q T_i^{exp}(x_i, y_i)$$

averaging in x- and y- direction

→ t temperature signal

g - 1D local Model



Anisotropic material : $k_x = k_y = 0$ $k_z = k$ or $\frac{\partial u}{\partial x} \approx 0$ and $\frac{\partial u}{\partial x} \approx 0$ (weak $\frac{\partial e}{\partial x}, \frac{\partial e}{\partial y}, \frac{\partial h}{\partial x}, \frac{\partial h}{\partial y}, \dots$)

Model f q independent local temperature models

$$y_{mo,i} = T_{g,i}(t) = T_c(z_i, t; \beta_i, u(x_i, y_i, t), \Delta T, T_\infty)$$

$$x_i = \{ \beta_i, u(x_i, y_i, t), \Delta T, T_\infty \}$$

$$\beta_i = (z_i^*, \tau_{diff,i}, R_i, H_i) \quad (q + 6) \text{ independent quantities if same } z_i \text{'s}$$

Measurements

$$y_i(t_k) = T_k^{exp}(x_i, y_i, z_i)$$

Remarks on physical reduction (for later inversion)

- the **simpler** the model, the **higher** the possible bias (for direct simulation)

but

- detailed model may be biased too /**experiment**
- decrease in **number** of parameters
- inversion more **robust**/noise amplification (inversion)
- parameters keep **explicit** physical meaning (white box): can be **exported** !
- **first step** for later finer inversion (non linear estimation)

Thank you for your attention !