



Experimental Heat Transfer Characterization of Outdoor PV Modules

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Thermal Challenges in PV

Why heat exchanges are important for PV plants ?

Predicting energy production is crucial for energy market

Need to balance electricity production on a daily basis → Intraday or day-ahead trading

Precise forecasting of electricity production is a matter of competitiveness

→ **Efficiency is temperature – dependent (cristalline silicon)**

Power temperature coefficient: $\frac{dP}{P} = \frac{d\eta}{\eta} \sim -\beta(T_{cell} - T_{STC})$

For cristaline silicon: $\beta \simeq -0.004 \text{ K}^{-1}$

$$T_{cell} = T_{STC} + 25^{\circ}\text{C} \Rightarrow \frac{dP}{P} \simeq -10\%$$

→ **PV module ageing :**

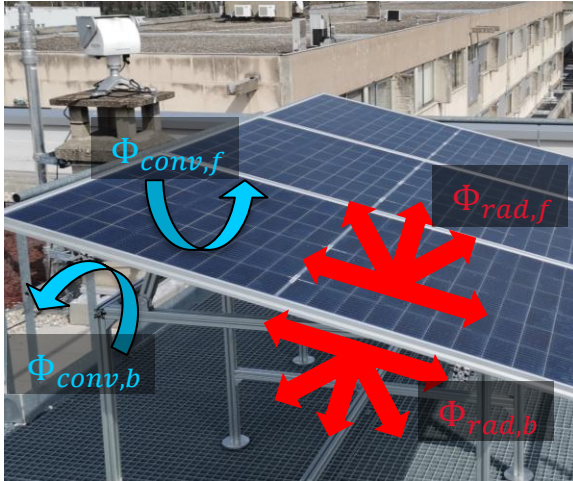
+1°C causes 3-5-years drop in modules lifetime(*)

(*) Xia, Z., Wohlgemuth, J. H. & Cunningham, D. W. *IEEE Photovolt. Spec. Conf.* (2009)

Thermal Challenges in PV

Heat exchange on PV module

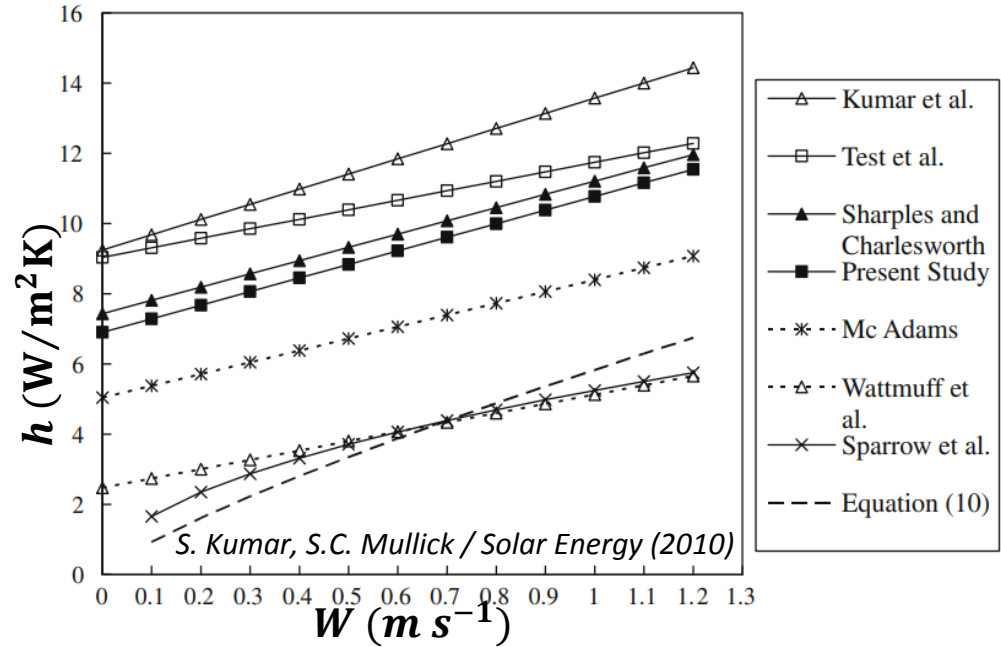
$$h_{global} = h_{conv} + h_{rad}$$



Forced convection:

$$\rightarrow Nu = aRe^b \quad Re \gtrsim Re_0 \quad Nu(Re_0) + abRe_0^{b-1} (Re - Re_0)$$

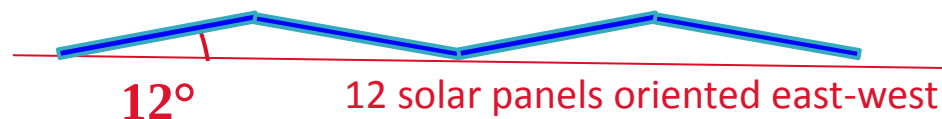
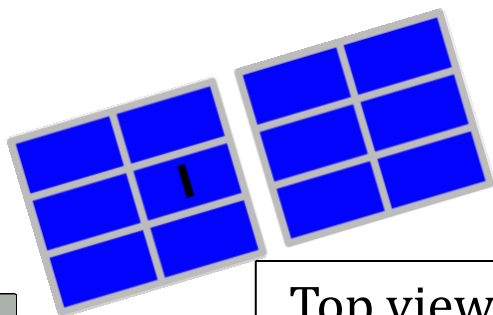
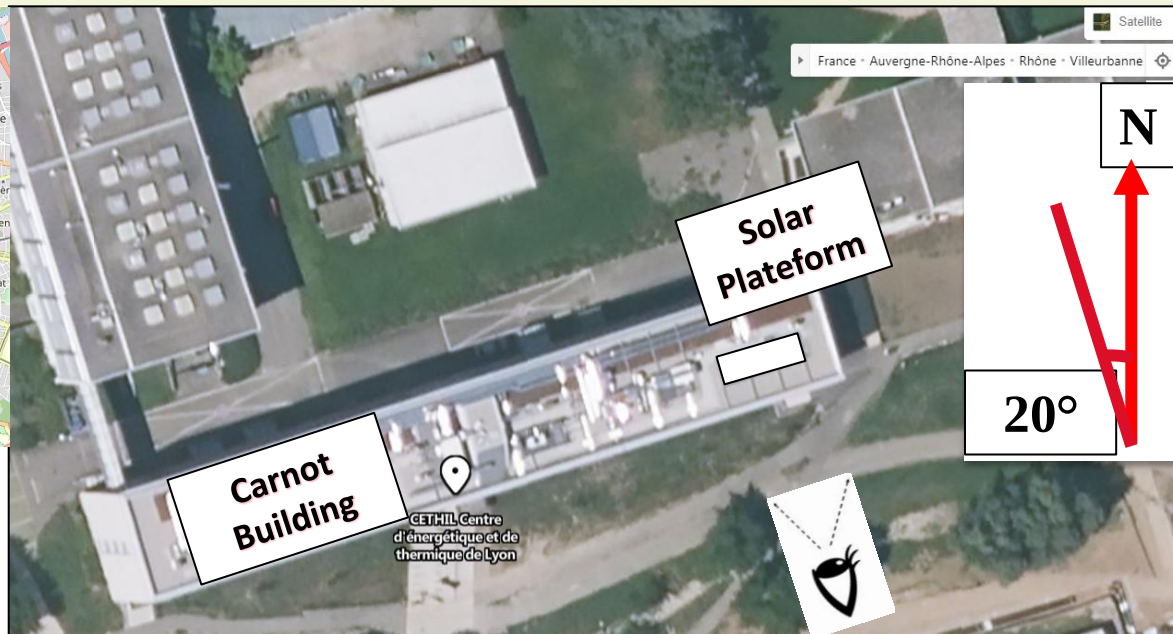
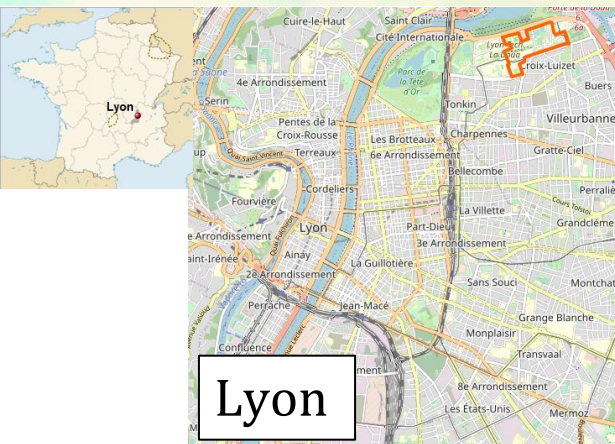
$$\rightarrow h = h_c + h_v W$$



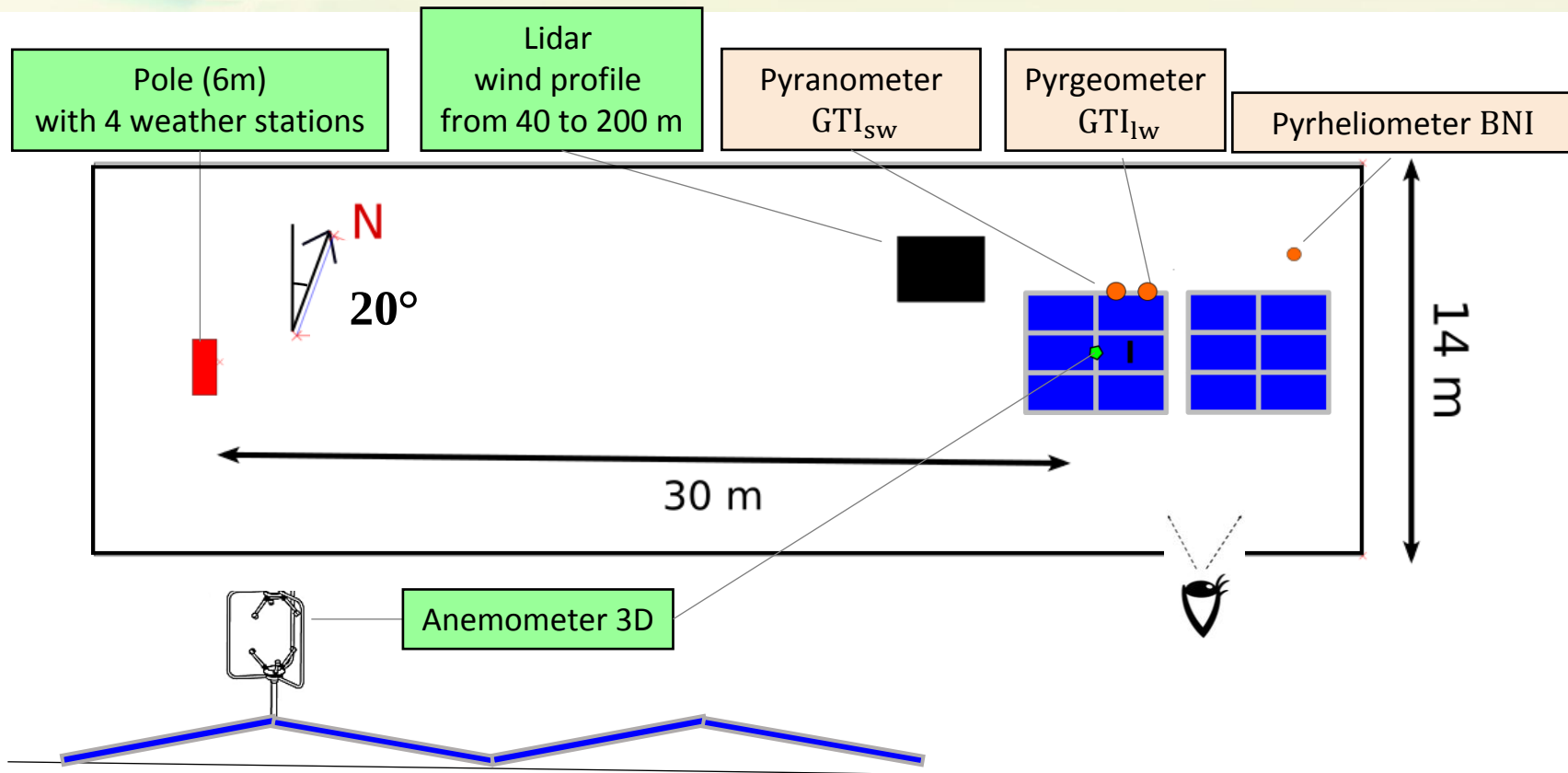
Convective heat transfer coefficient:

- Measurement ?
- Modelling ?

Solar Platform at CETHIL

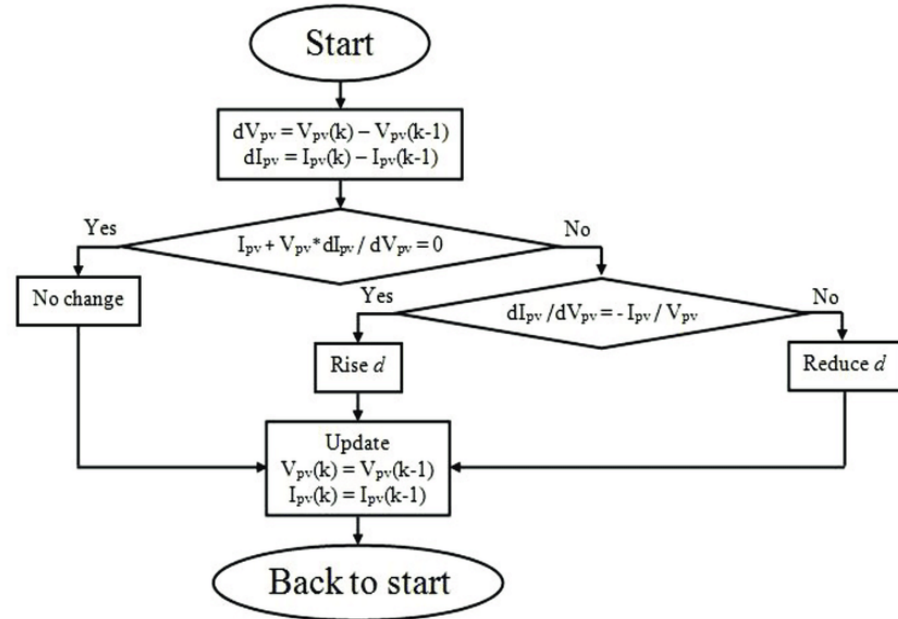
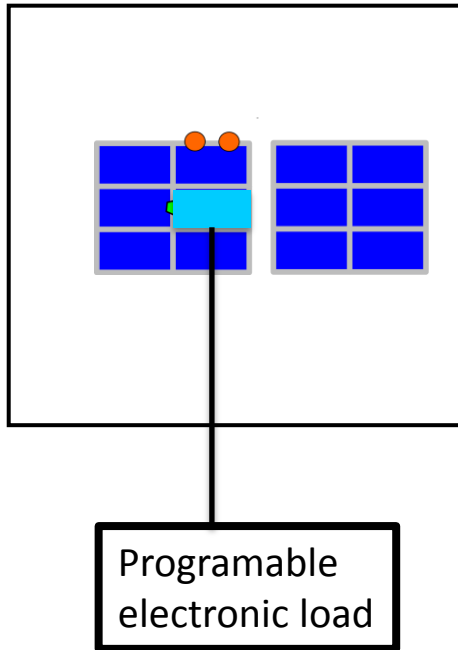


Solar Platform at CETHIL



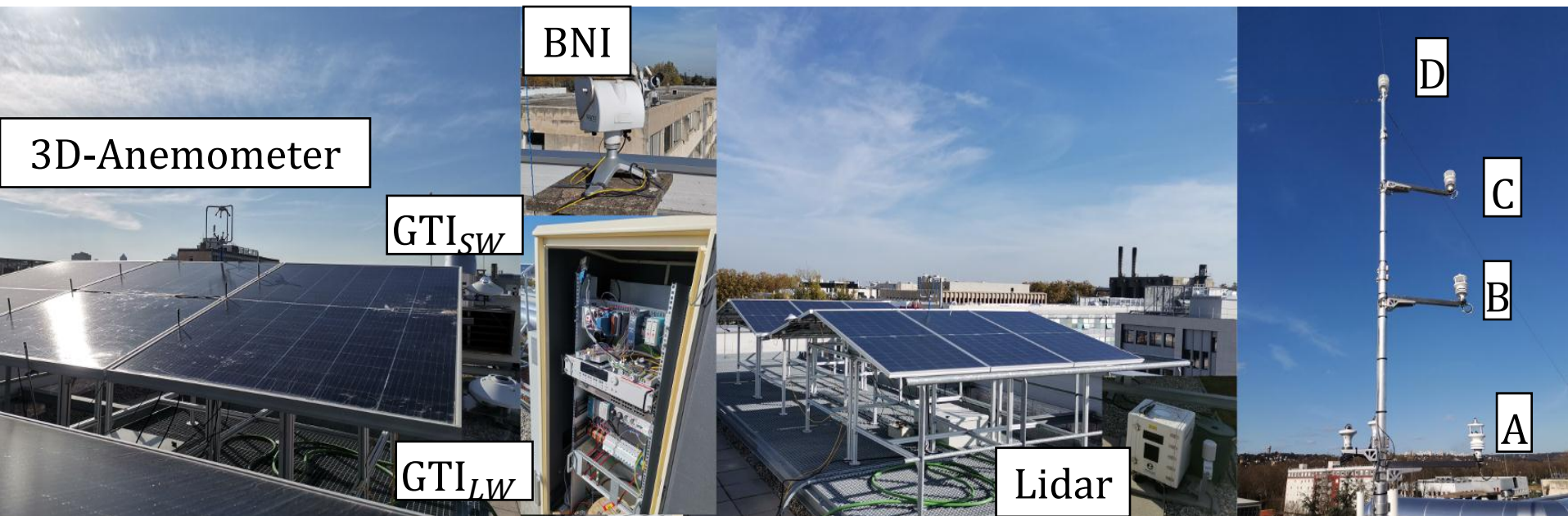
Solar Platform at CETHIL

Electrical power: 1 panel at maximum power point



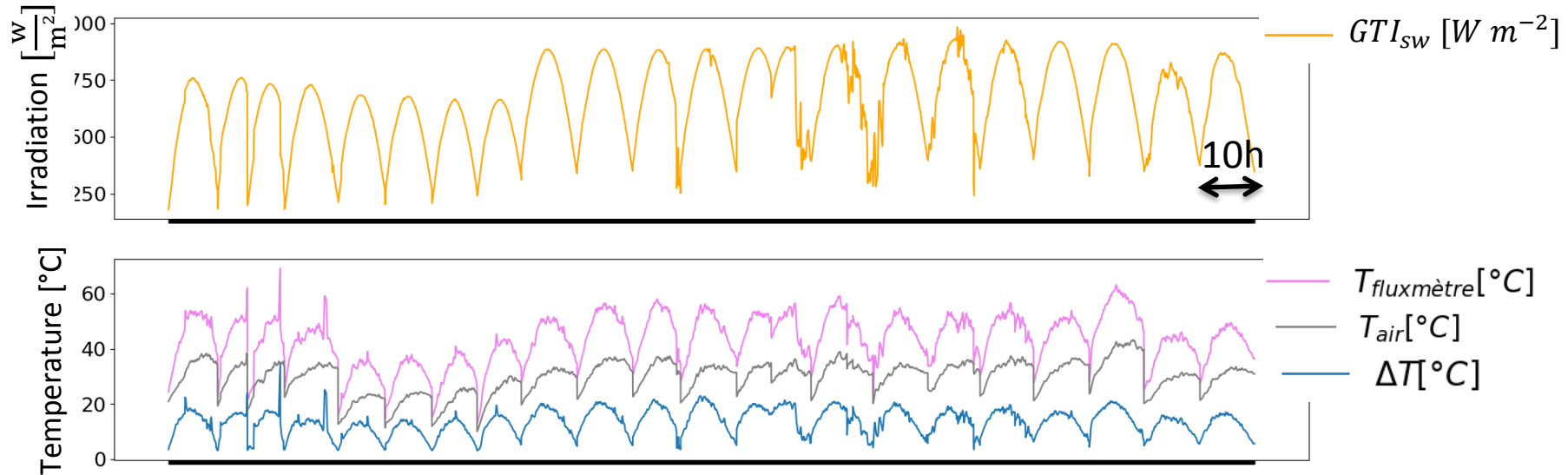
Incremental conductance algorithm

Solar Platform at CETHIL



Convective Heat Transfer - Dataset

Dataset: 10h/d x 22d = 220h ; 10min-rolling-average ; $f=1\text{min}^{-1}$



Convective Heat Transfer – Reference Measurement

Heat balance of an instrumented plate

$$m_p C_p \frac{dT_p}{dt} = \alpha_{sw} GTI_{sw} S_p - \varepsilon_{lw,f} (\sigma T_m^4 - GTI_{lw}) S_p - h_f (T_m - T_{air}) S_p - \Phi_b$$

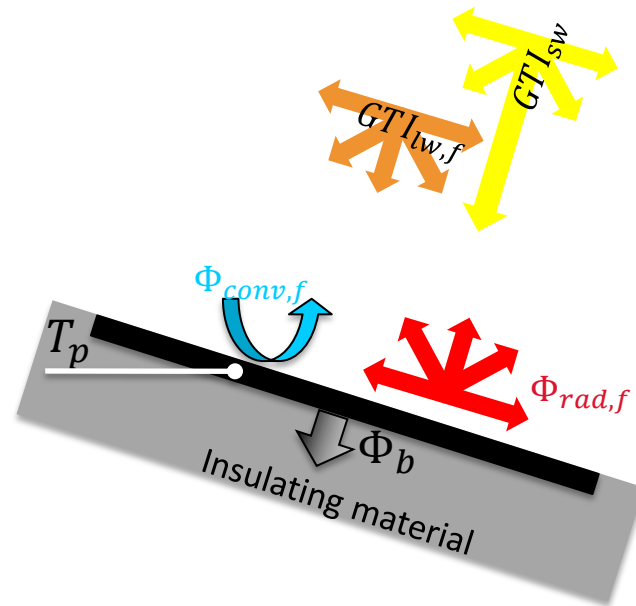
$$h_f = \frac{1}{(T_m - T_{air}) S} \left[\alpha_{sw} GTI_{sw} S_p - \varepsilon_{lw,f} (\sigma T_m^4 - GTI_{lw}) S_p - \Phi_p - m_p C_p \frac{dT_p}{dt} \right]$$

Hypothesis

- View factor $F_{p \rightarrow sky} = 1$
- h_f independent on temperature

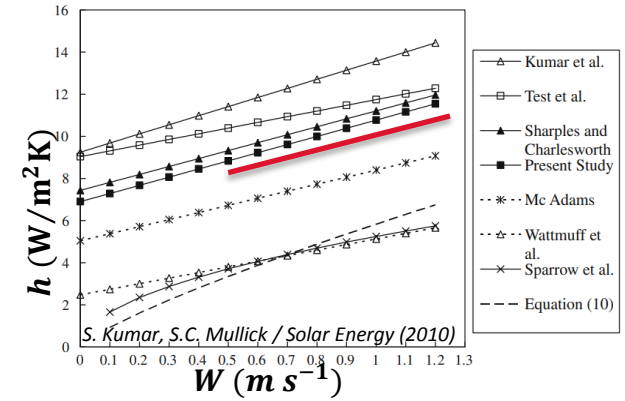
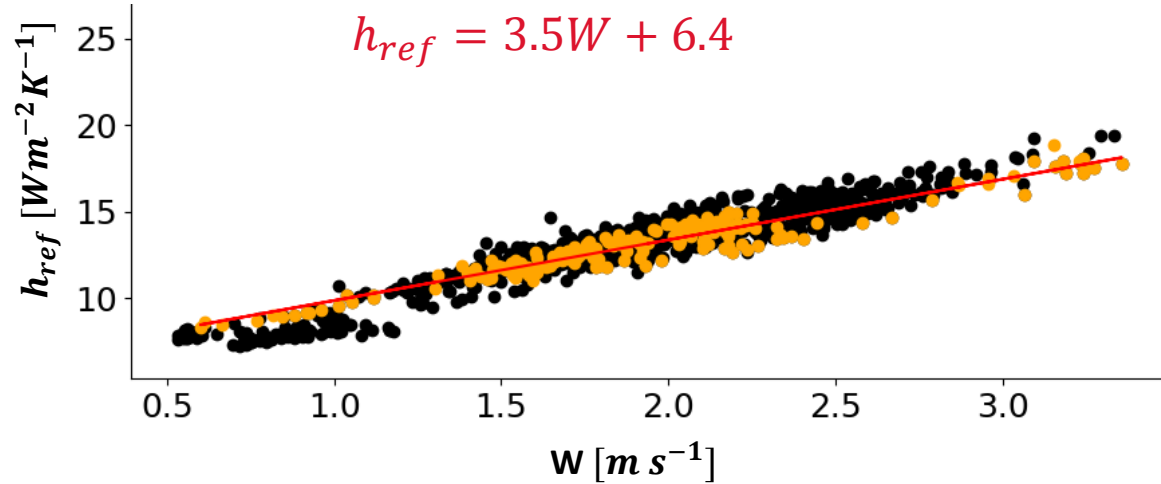
Measurements

- T_p, T_{air} : thermocouples
- $\alpha_{sw}, \varepsilon_{lw,f}$: spectrometer measurements
- GTI_{sw} : pyranometer
- GTI_{lw} : pyrgeometer or empirical law for T_{sky}
- Φ_p : heat flux meter or neglected
- $m_p C_p \frac{dT_p}{dt}$: may be neglected for thin film



Warning: plate can heat up to 90°C !!

Convective Heat Transfer – Reference Measurement



How to measure h_f on a PV panel in operation ?

Convective Heat Transfer – Direct Measurement

Heat flux sensor

- Custom-made sensor to avoid shading effect
- Sensitivity: $6 \mu V / (W m^{-2})$

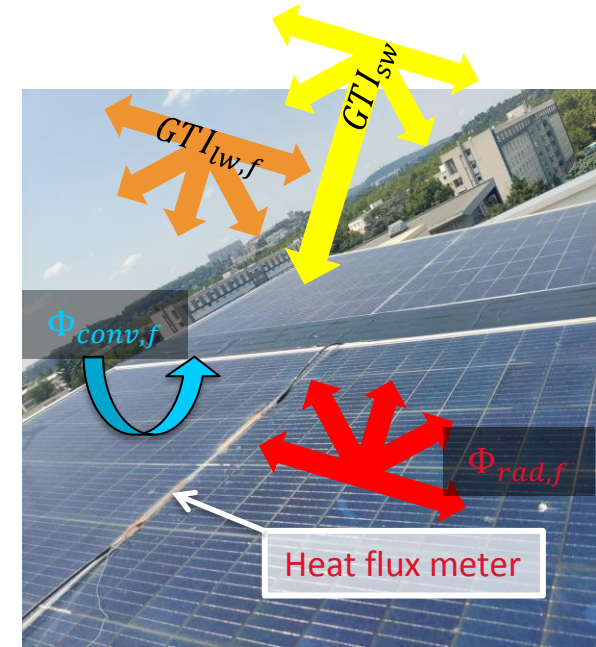
$$\Phi_{meas} = \Phi_{conv} + \Phi_{net_rad}$$

$$\Phi_{net_rad} = [\cancel{\Phi_{emi}} - \Phi_{abs}]_{sw} + [\Phi_{emi} - \Phi_{abs}]_{lw}$$

$$\Phi_{meas} = \underbrace{h_f(T_{fl} - T_{air})}_{conv} - \underbrace{\alpha_{fl,sw} GTI_{sw}}_{net_rad_{sw}} + \underbrace{\alpha_{fl,lw}(\sigma T_{fl}^4 - GTI_{lw})}_{net_rad_{lw}}$$

T_{fl} : temperature of the heat flux meter

$\alpha_{fl,sw}$; $\alpha_{fl,lw}$: absorptivity of the heat flux meter in the short and long wavelength ranges



Convective Heat Transfer – Direct Measurement

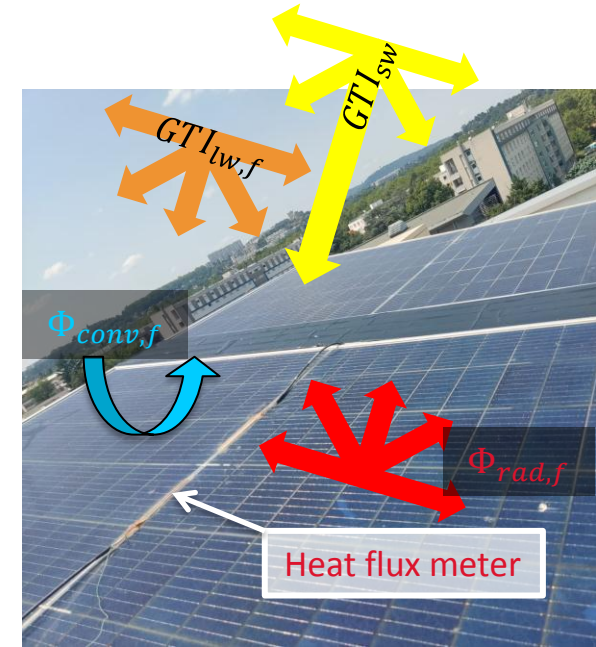
$$\Phi_{meas} = \underbrace{h_f(T_{fl} - T_{air})}_{conv} - \underbrace{\alpha_{sw}GTI_{sw}}_{net_rad_{sw}} + \underbrace{\alpha_{lw}(\sigma T_{fl}^4 - GTI_{lw})}_{net_rad_{lw}}$$

Measurements:

- Φ_{meas}
- $T_{fl}; T_{air}$
- $\alpha_{fl,sw}; \alpha_{fl,lw}$ (black coating)
- $GTI_{sw}; GTI_{lw}$ or T_{sky}

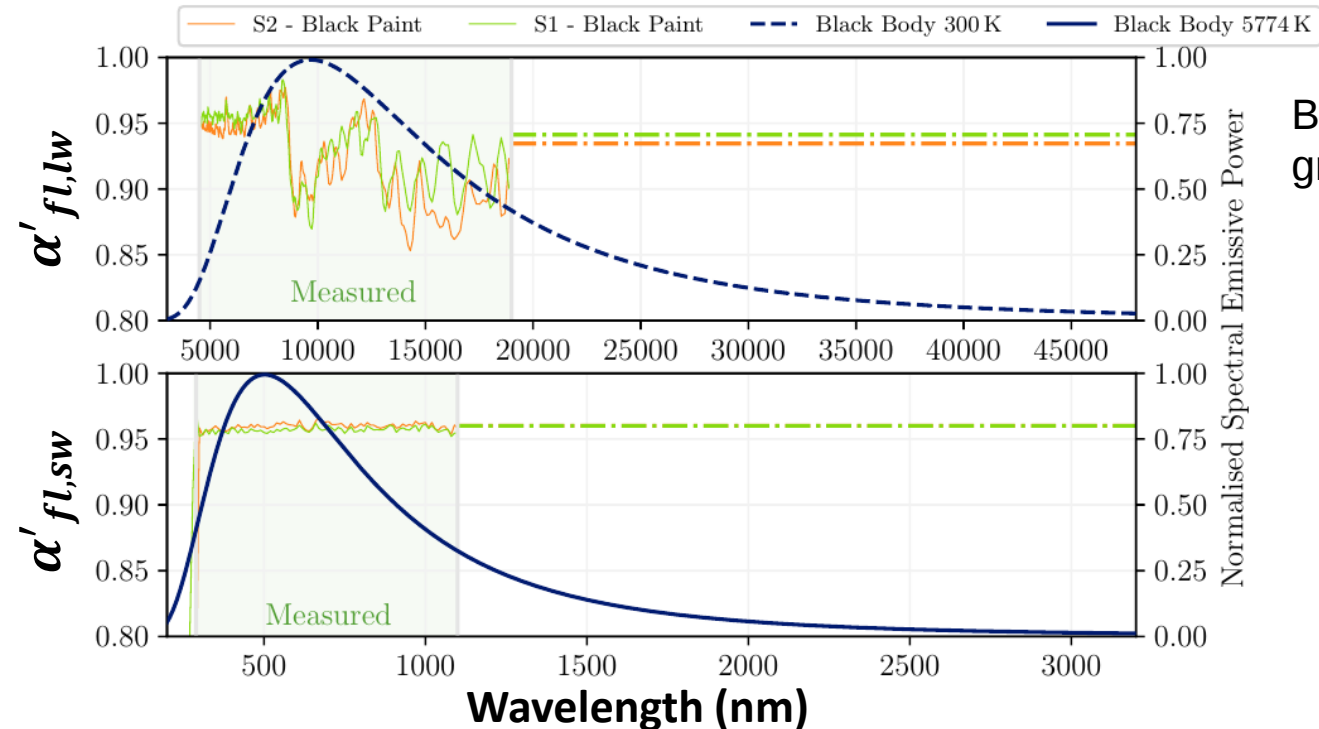


h_f



Convective Heat Transfer – Direct Measurement

Absorptivity of black painted heat flux meter measured by spectrometer



Black coating is assumed to be grey on the sw and lw bands

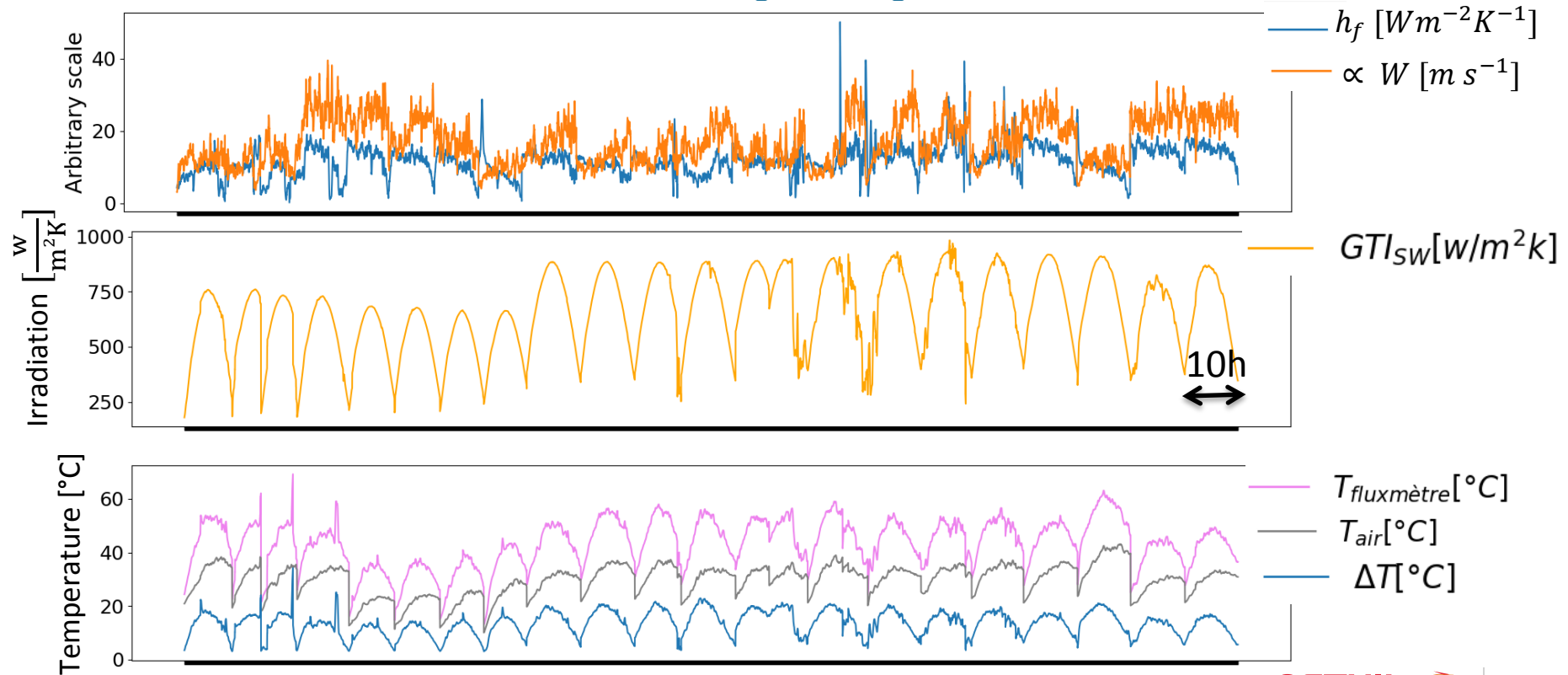
$$\alpha_{fl} = \frac{1}{\lambda_2 - \lambda_1} \int_{\lambda_1}^{\lambda_2} \alpha'_{fl}(\lambda) d\lambda$$

$$\alpha_{sw} = 0.9$$

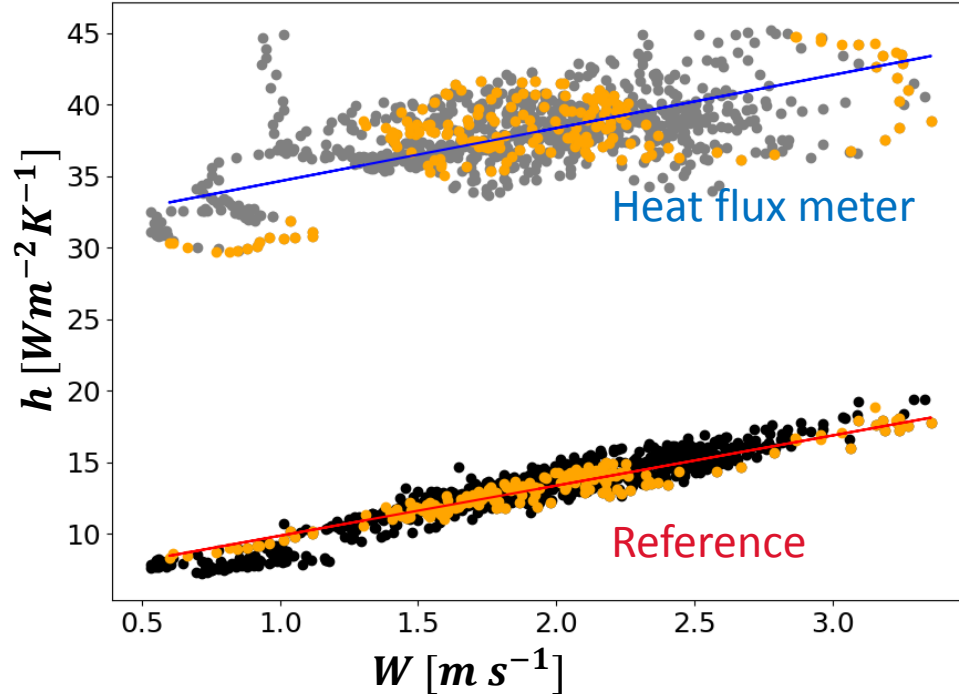
$$\alpha_{lw} = 0.95$$

Convective Heat Transfer – Direct Measurement

Dataset: 10h/d x 22d = 220h ; 10min-rolling-average ; $f=1\text{min}^{-1}$



Convective Heat Transfer – Comparison with Reference



$$h_f = 3.7W + 30.9$$

?

$$h_{ref} = 3.5W + 6.4$$

Convective Heat Transfer – Comparison with Reference

$$h_f(T_{fl} - T_{air}) = \Phi_{meas} + \alpha_{sw}GTI_{sw} - \alpha_{lw}(\sigma T_{fl}^4 - GTI_{lw})$$

Visual inspection of the sensor 6 months after installation: sand soiling

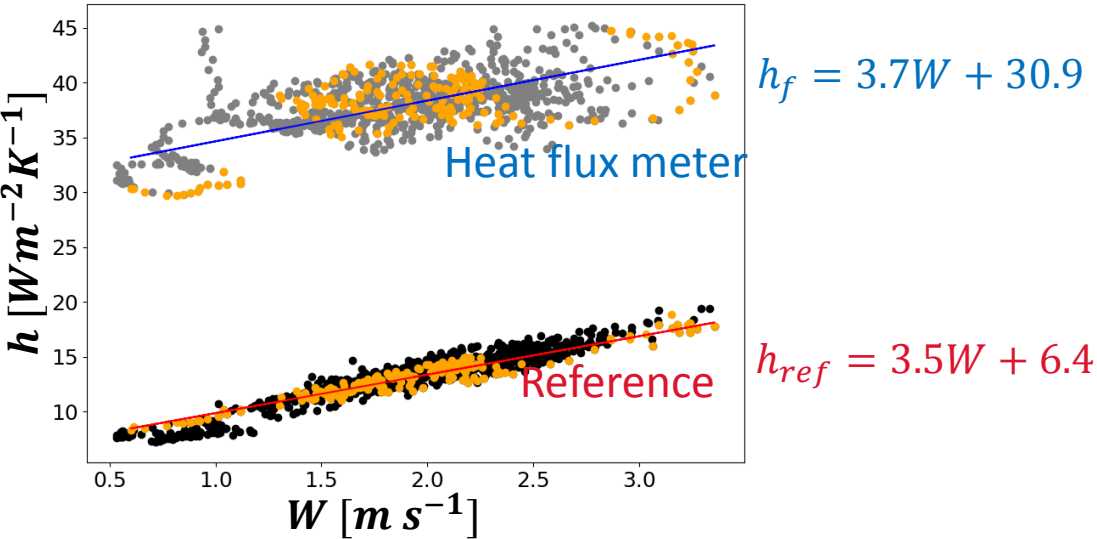
=> Hypothesis: uncertainty comes from a change in the absorptivity in the short wave range

$$\alpha_{sw_true} = \alpha_{sw_meas} + \delta\alpha_{sw}$$

$$\underbrace{h_f(\alpha_{sw_true})}_{corrected} = \underbrace{h_f(\alpha_{sw_meas})}_{measured} + \frac{\partial h_f}{\partial \alpha_{sw}} \delta\alpha_{sw} \quad \text{with} \quad \frac{\partial h_f}{\partial \alpha_{sw}} = \frac{GTI_{sw}}{T_{fl} - T_{air}}$$

$$h_{f,corr} = h_{f,meas} + \frac{GTI_{sw}}{T_{fl} - T_{air}} \delta\alpha_{sw}$$

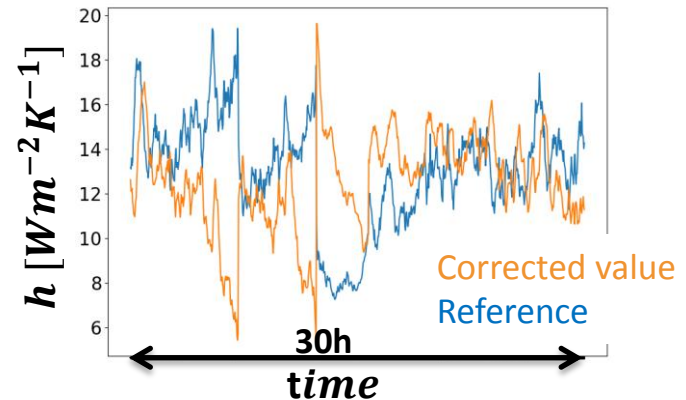
Convective Heat Transfer – Comparison with Reference



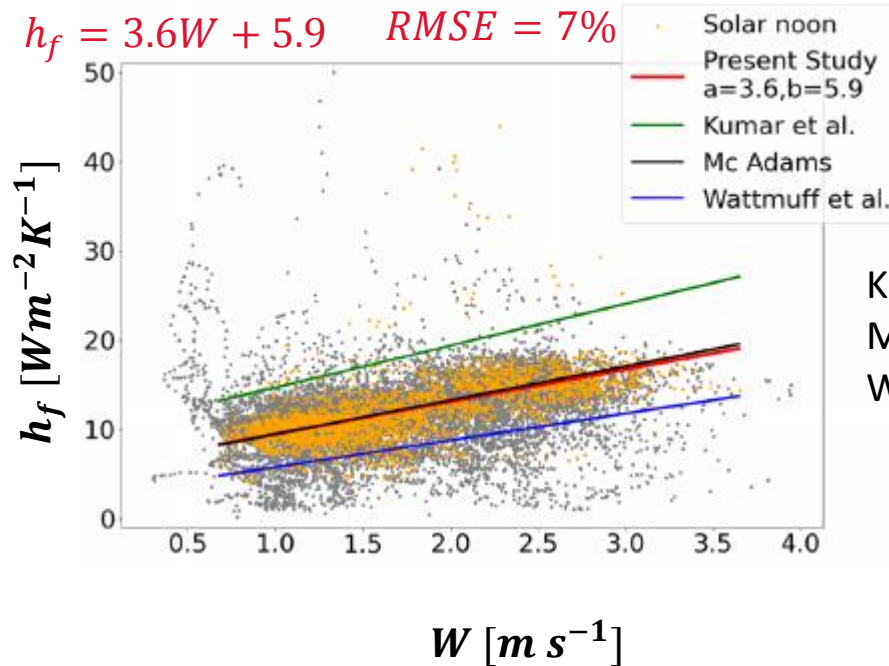
Correction is consistent

$$h_{f,corr} = h_{f,meas} + \underbrace{\frac{GTI_{sw}}{T_{fl} - T_{air}}}_{\approx U_{val}} \delta \alpha_{sw}$$

Best choice $\rightarrow \alpha_{sw,corr} = 0.45$

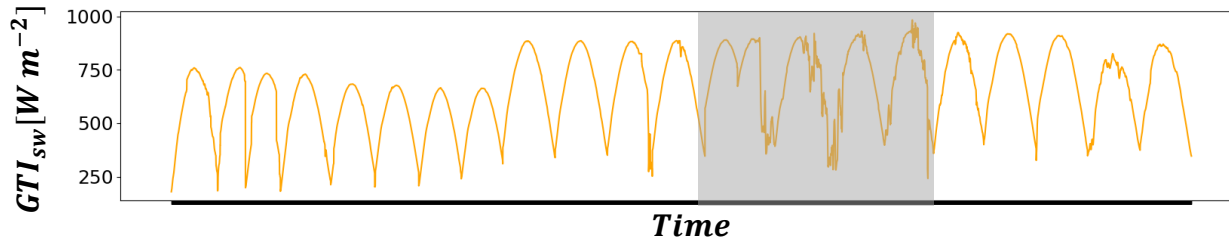
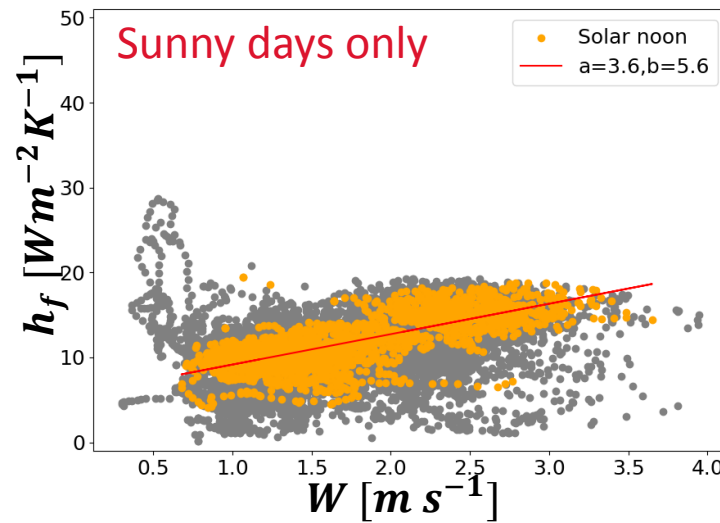
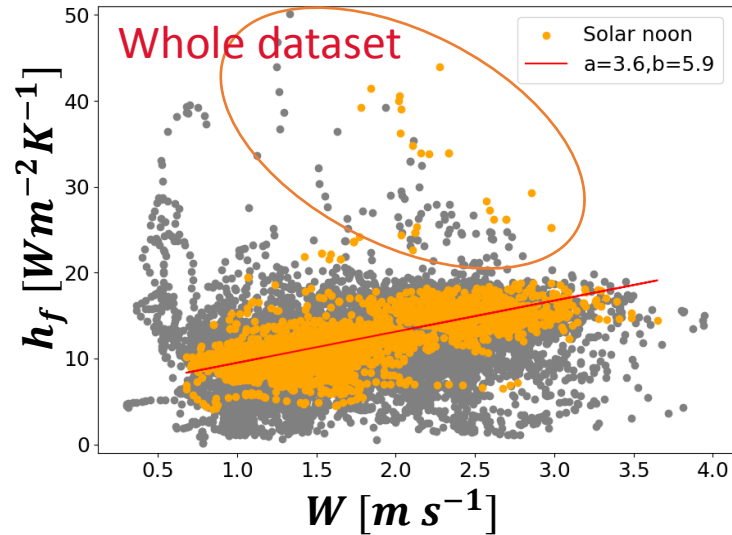


Convective Heat Transfer – Linear Regression at solar noon



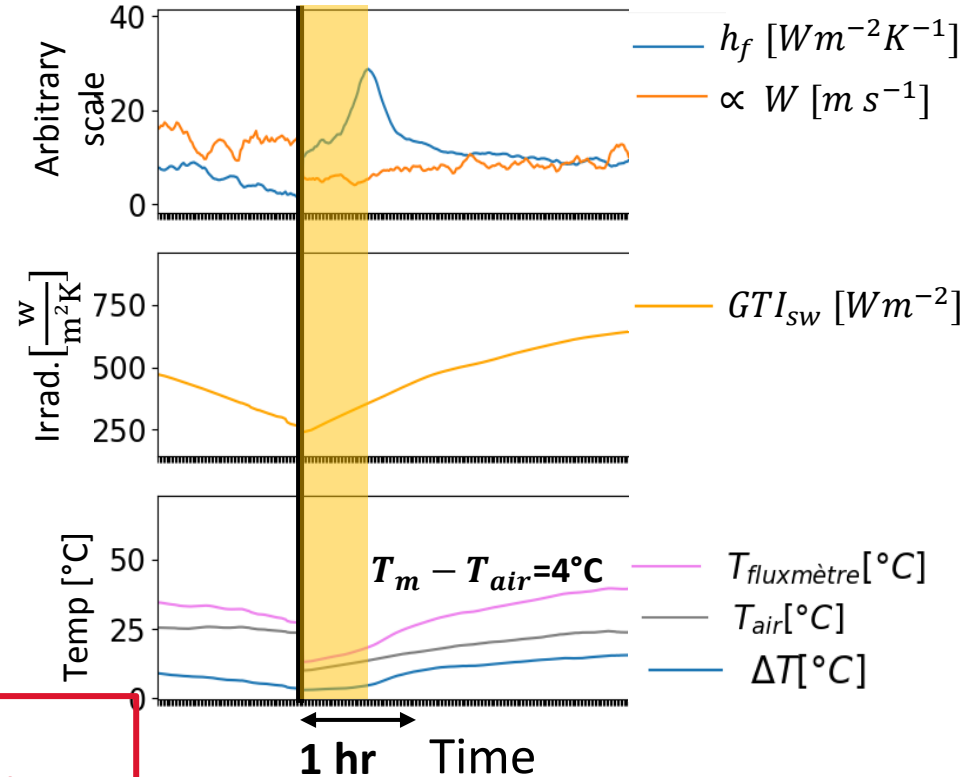
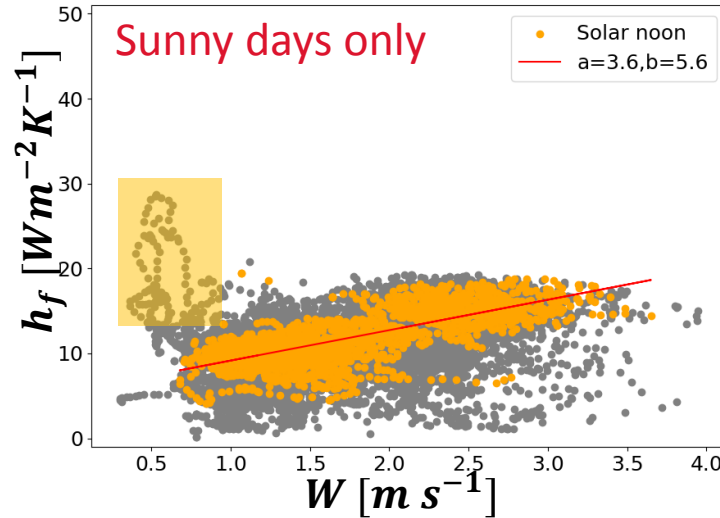
Kumar et al. : indoor experiment
Mc Adams: Wind tunnel
Wattmuff et al. : Wind tunnel

Convective Heat Transfer – Outliers points



Most of outliers
occur during cloudy days

Convective Heat Transfer – Outliers points



Other outliers values due to inertia effect

→ Φ_{meas} and $T_m - T_{air}$ are sometimes out of phase

Conclusions

Direct measurement of h_f

- Custom-made heat-flux meter
- The measurements are valid as long as the coating remains clean or if the measurements are taken immediately after installation
- Promising solution but improvements are needed especially in unsteady conditions (cloudy days) → unsteady model for heat flux meter ?

