

Simulations of heat and mass transfer in the capillary evaporator of a two-phase loop Towards an optimized evaporator design

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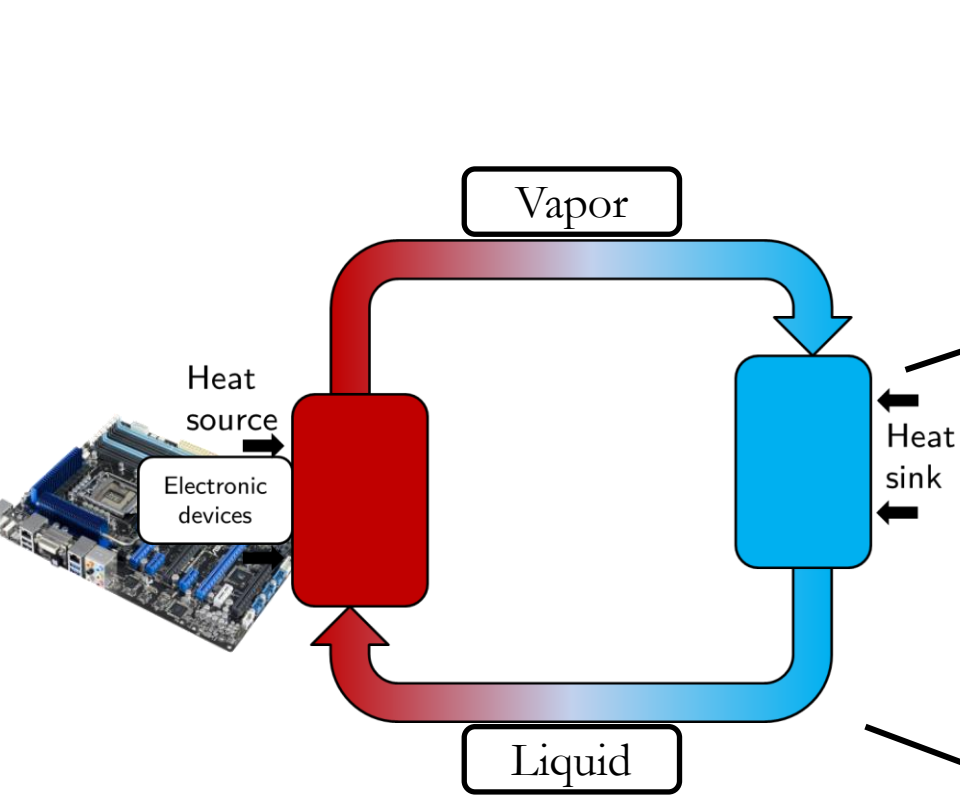
- 1 Introduction and context
- 2 Modelling
- 3 3D pore network simulations
- 4 Sensitivity study: towards an optimized design
- 5 Conclusions

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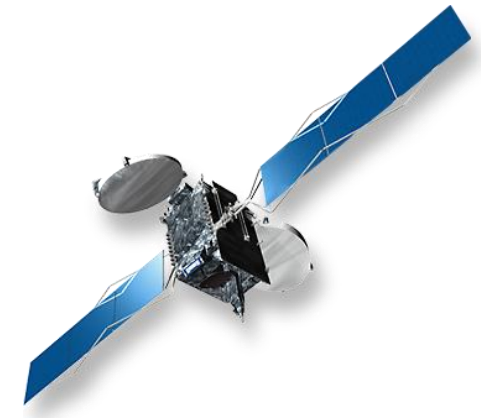
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Two-phase loop

Thermal control



Space application



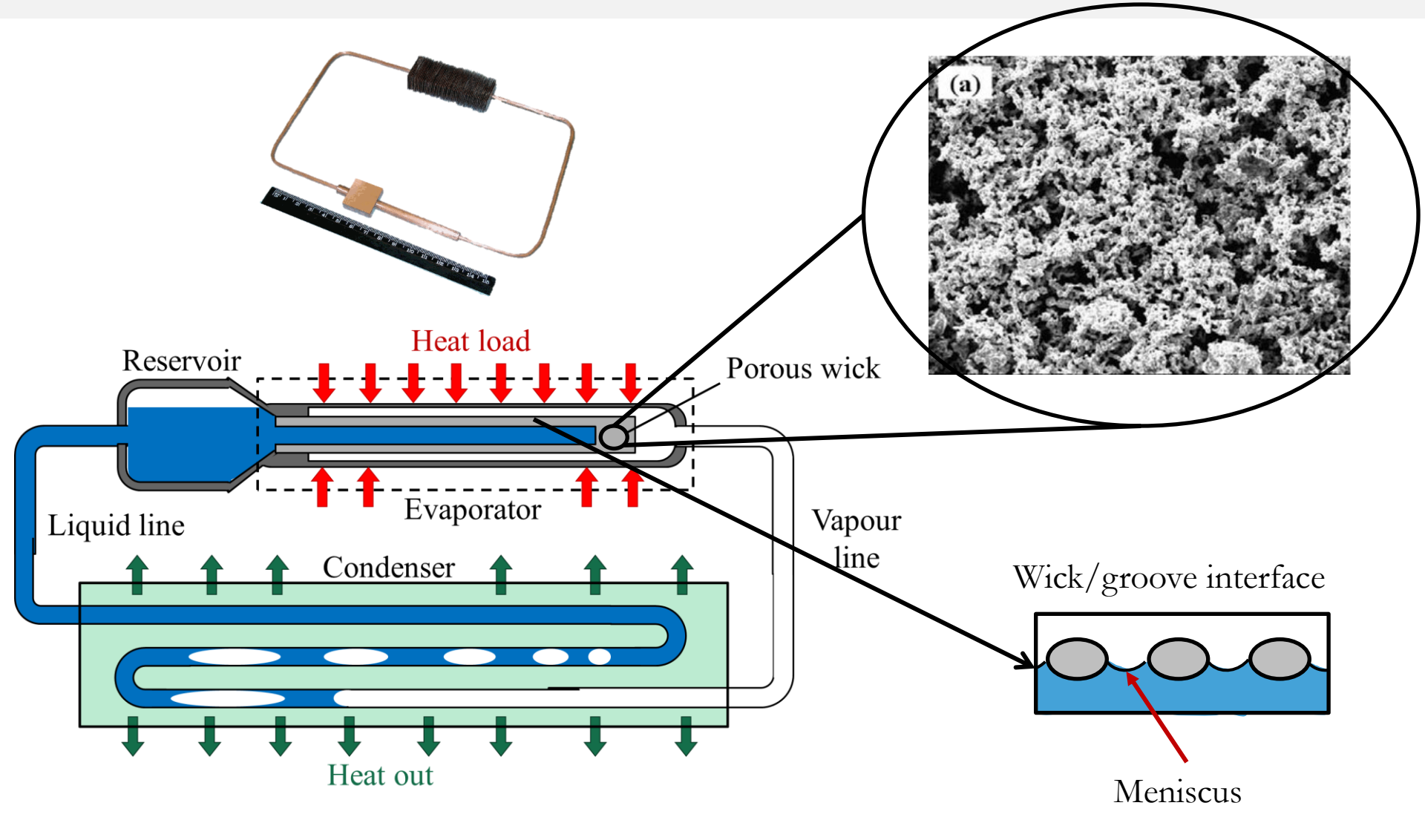
Terrestrial application



Two-phase loop
 ⇒ Loop heat pipe (LHP)

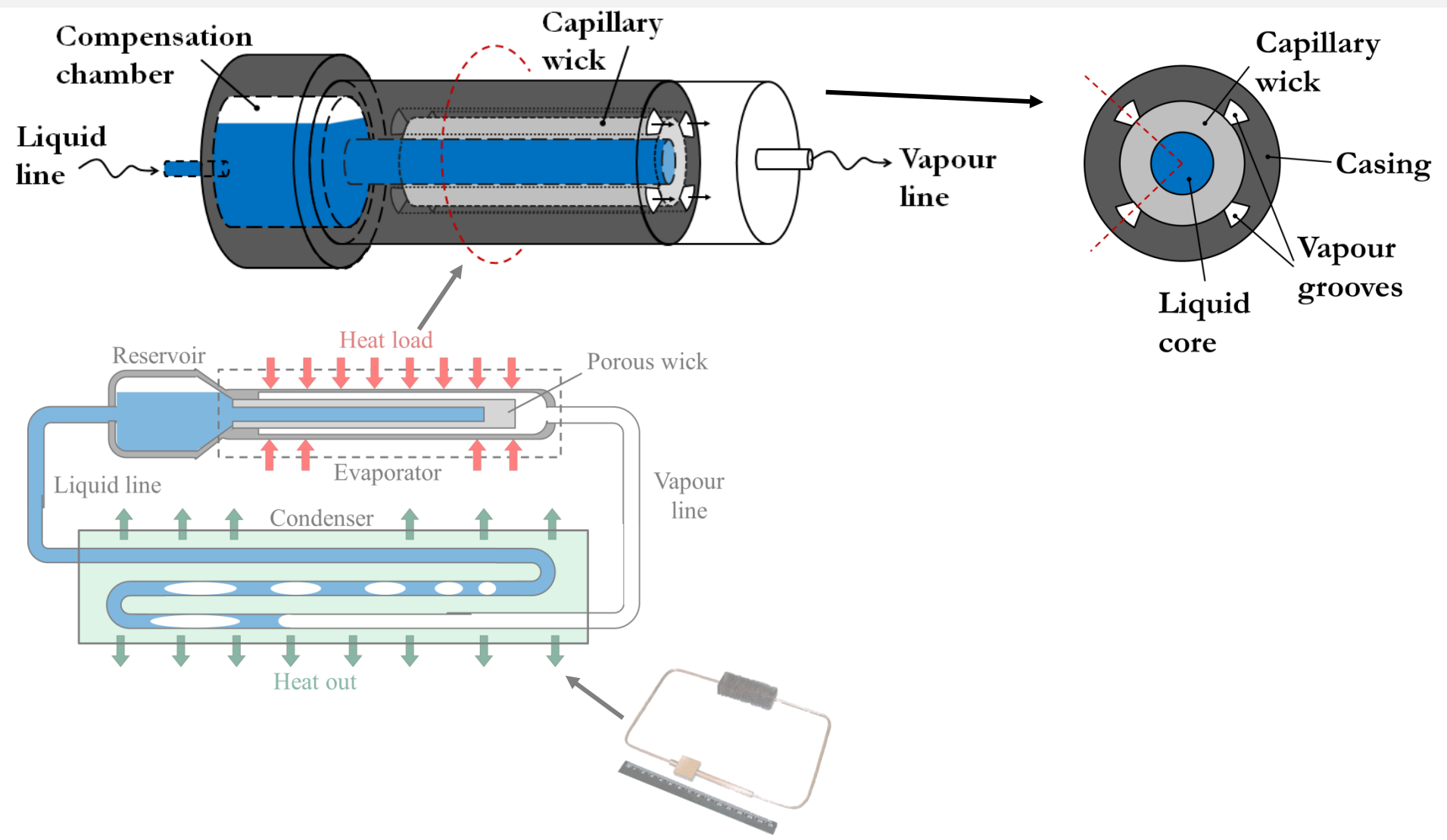
Loop heat pipe

Loop heat pipe components and fluid motion



Loop heat pipe

Cylindrical evaporator



Loop heat pipe

Evaporator unit cell

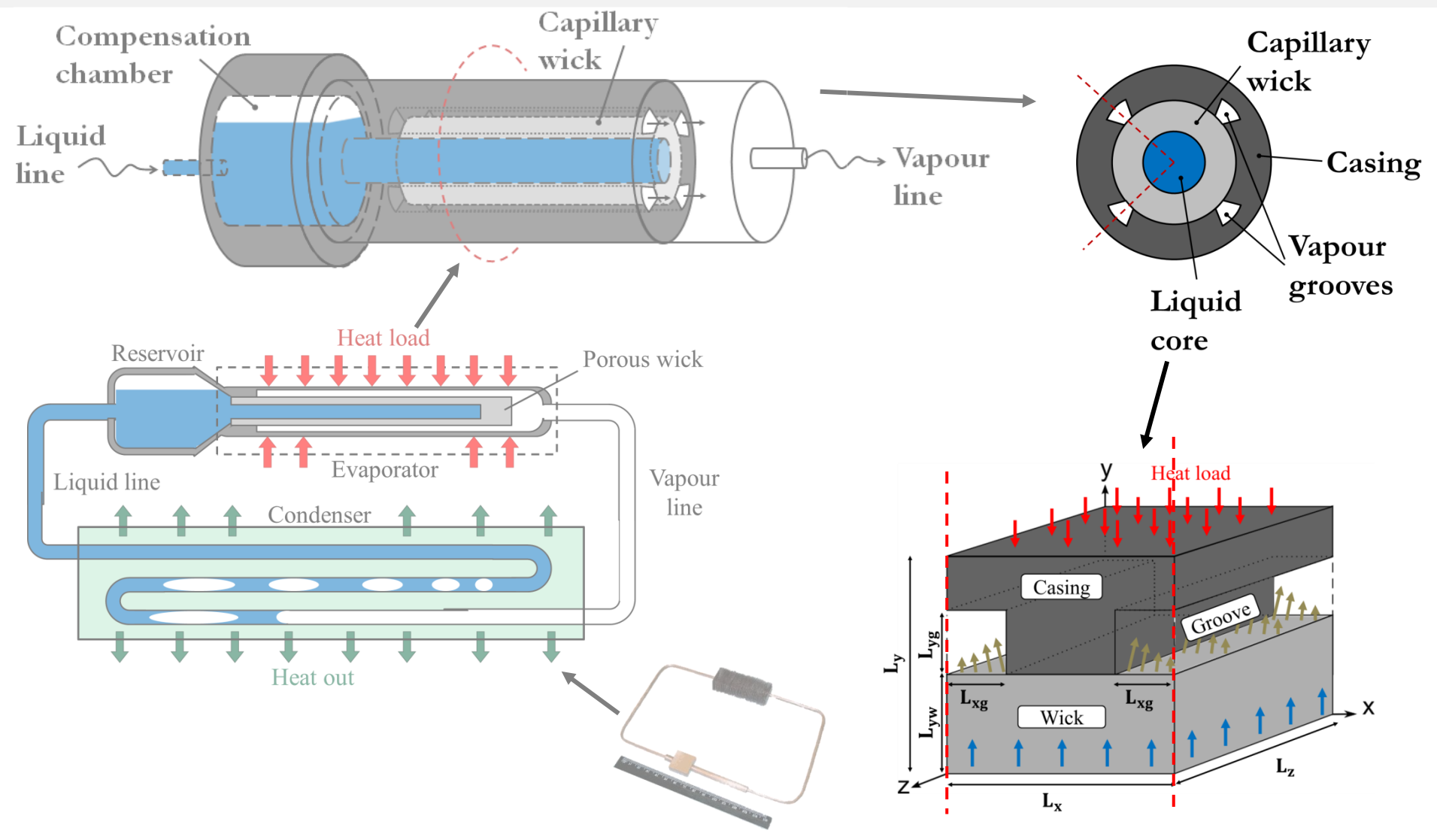
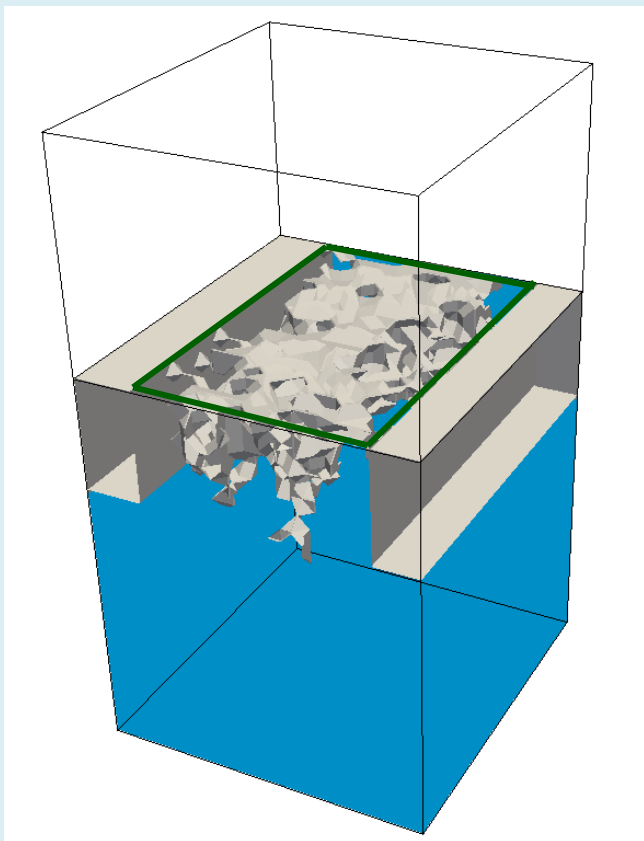


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Distribution of liquid and vapour phases

Distribution of liquid and vapour phases



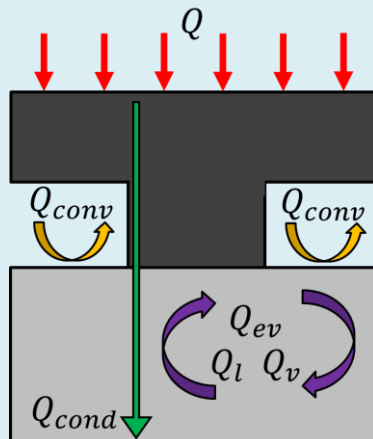
- S_{vc} and S_{lc} : fractions of pores occupied by vapour and liquid respectively right under the casing

Performances of evaporator

Heat flux balance

$$Q = Q_{ev} + Q_{cond} + Q_{conv} + Q_v + Q_l$$

- Q : heat load applied at the external surface of the casing
- Q_{ev} : heat flux actually used for vaporisation
- Q_{cond} : parasitic heat flux, i.e. flux lost by conduction towards the wick inlet



Thermal performances

- Conductance

$$h_{ev} = \frac{Q}{T_{max} - T_g}$$

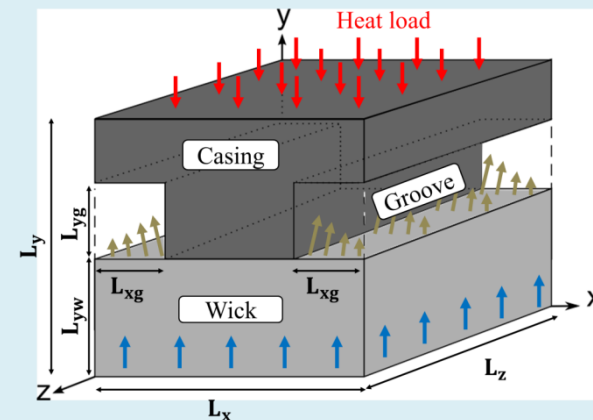
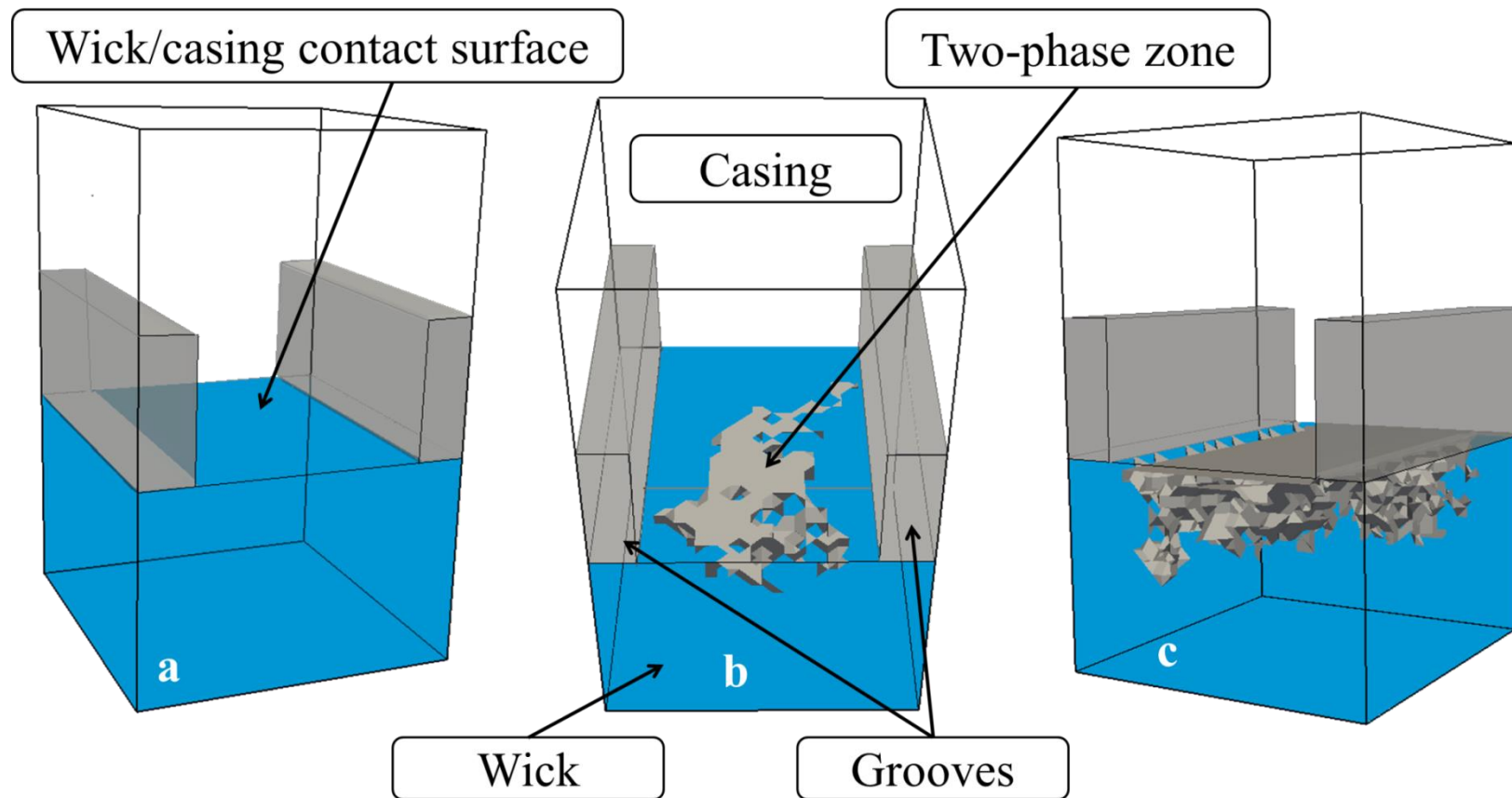


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Main operating regimes

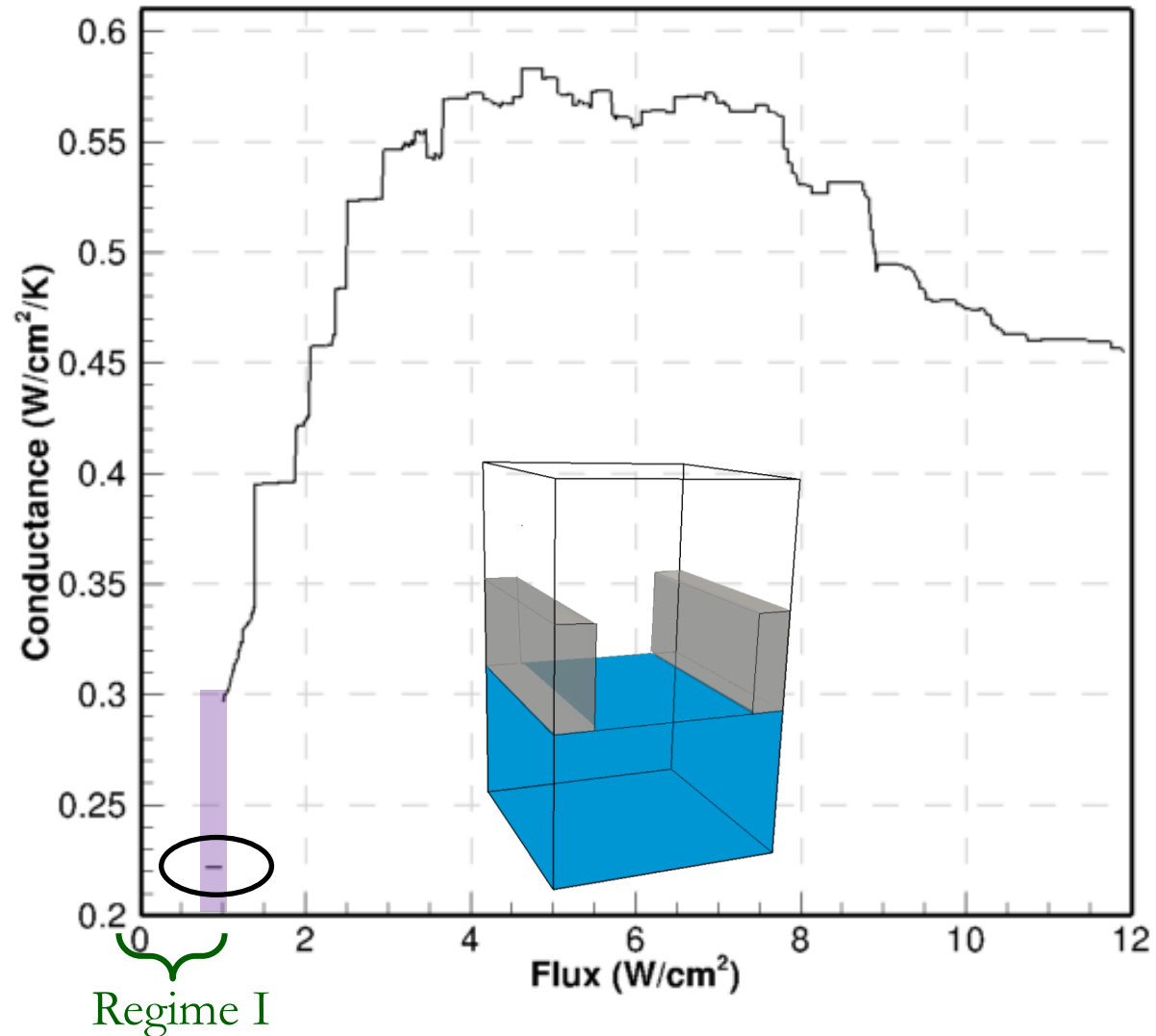


a) $0.8\text{W}/\text{cm}^2$
Regime I
Low heat flux

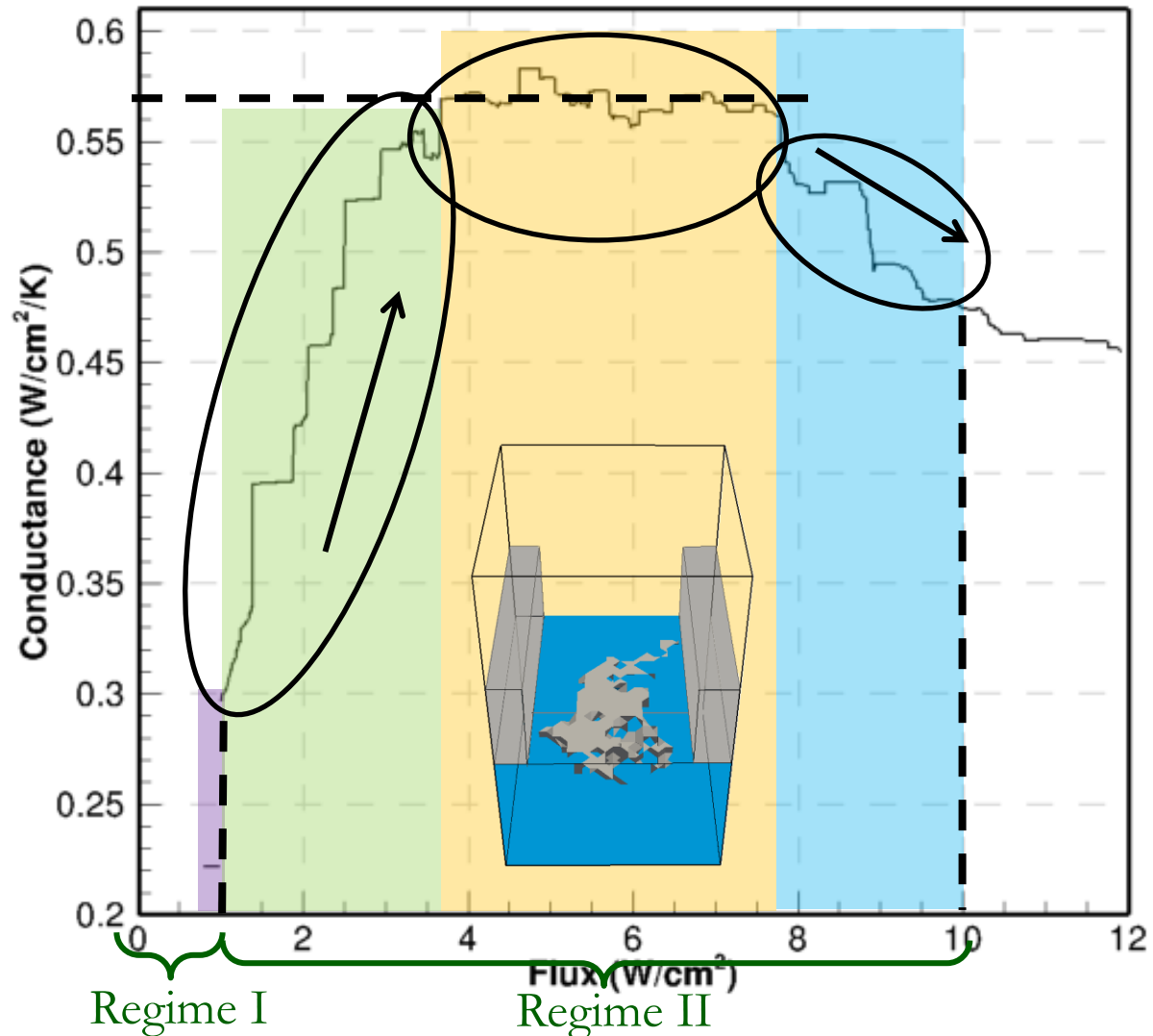
b) $1\text{W}/\text{cm}^2$
Regime II
Moderate heat flux

c) $10\text{W}/\text{cm}^2$
Regime III
High heat flux

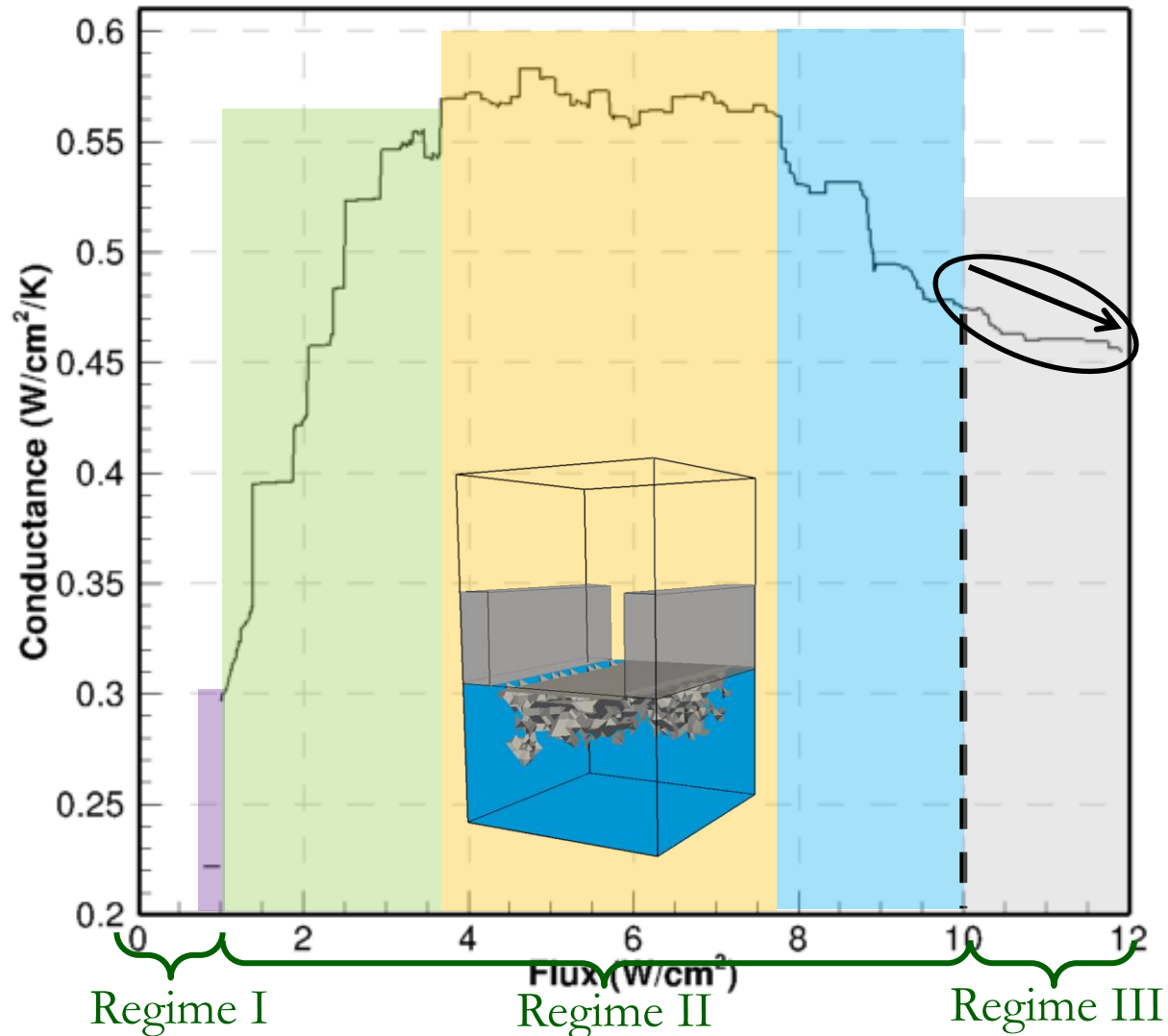
Conductance



Conductance



Conductance



Conductance

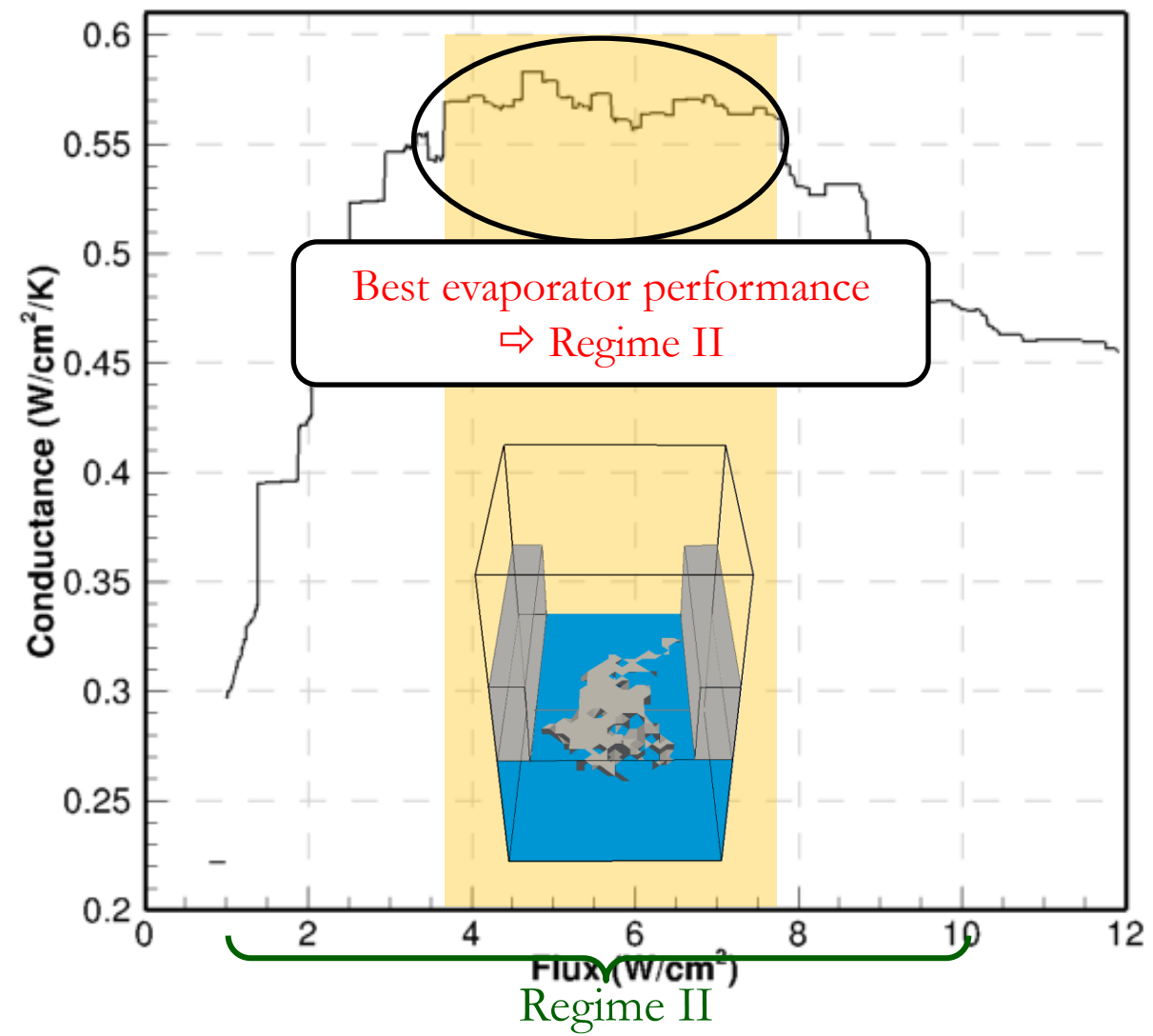
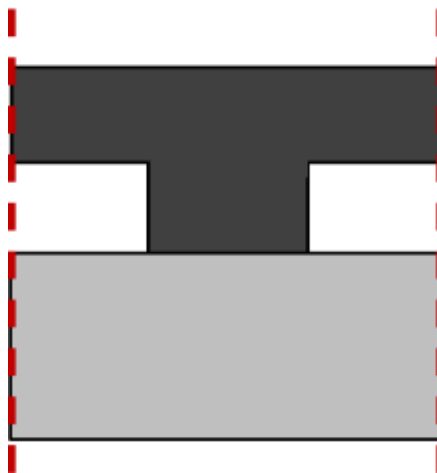


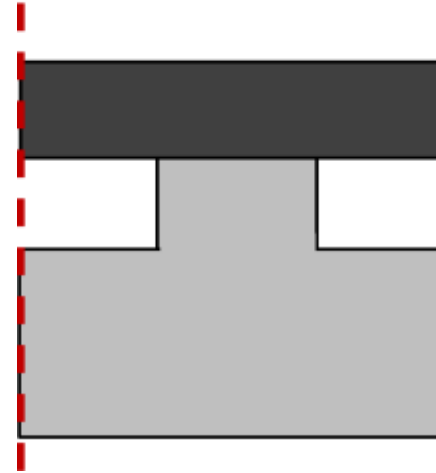
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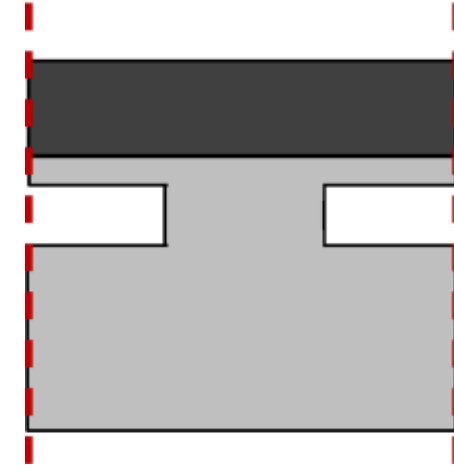
Objective



Geometry A



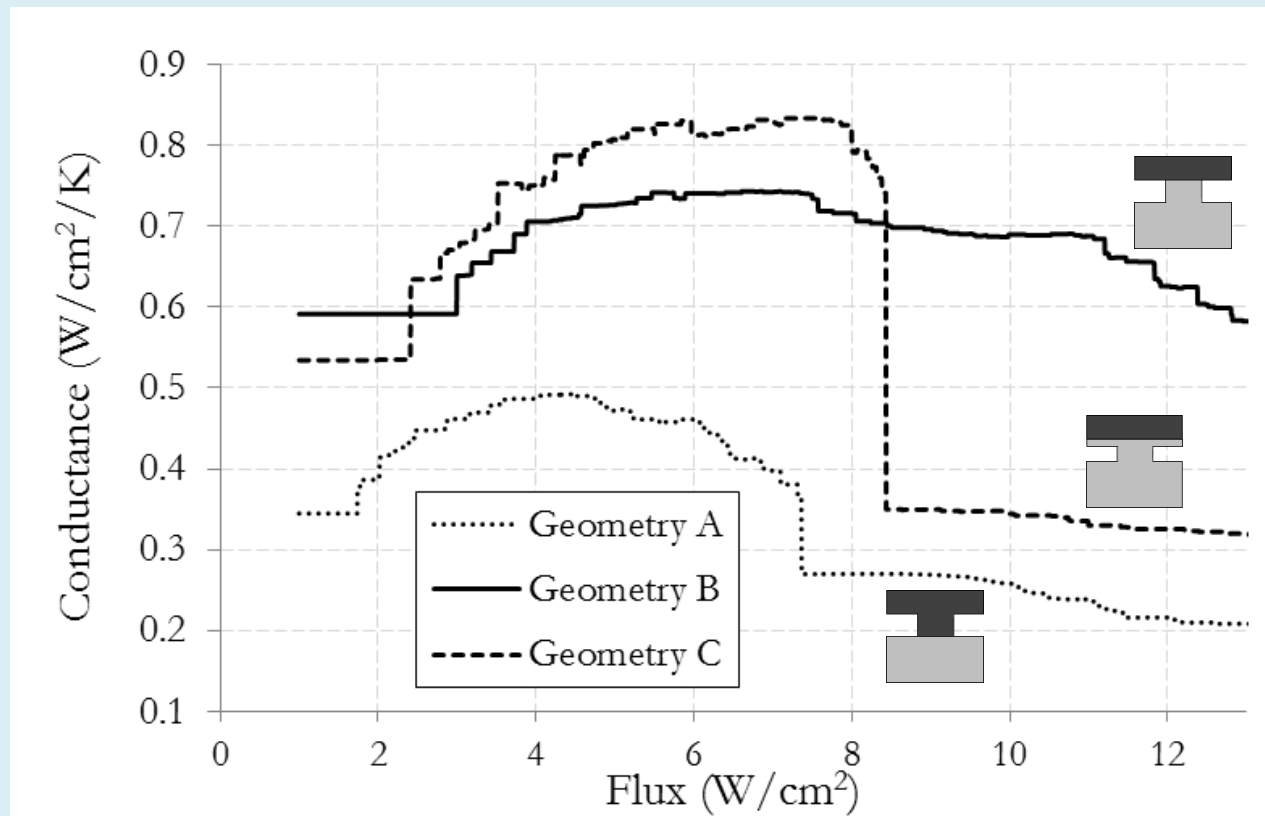
Geometry B



Geometry C

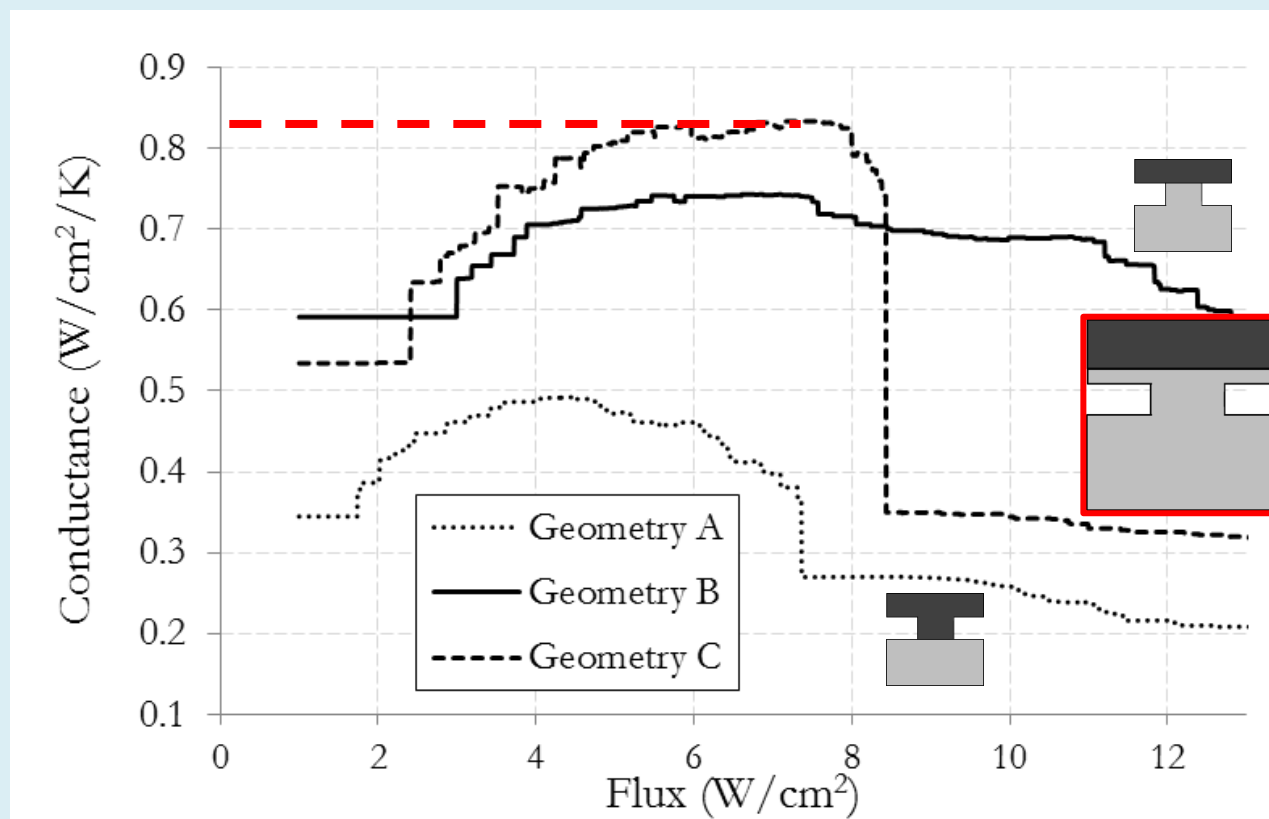
Conductance

Conductance



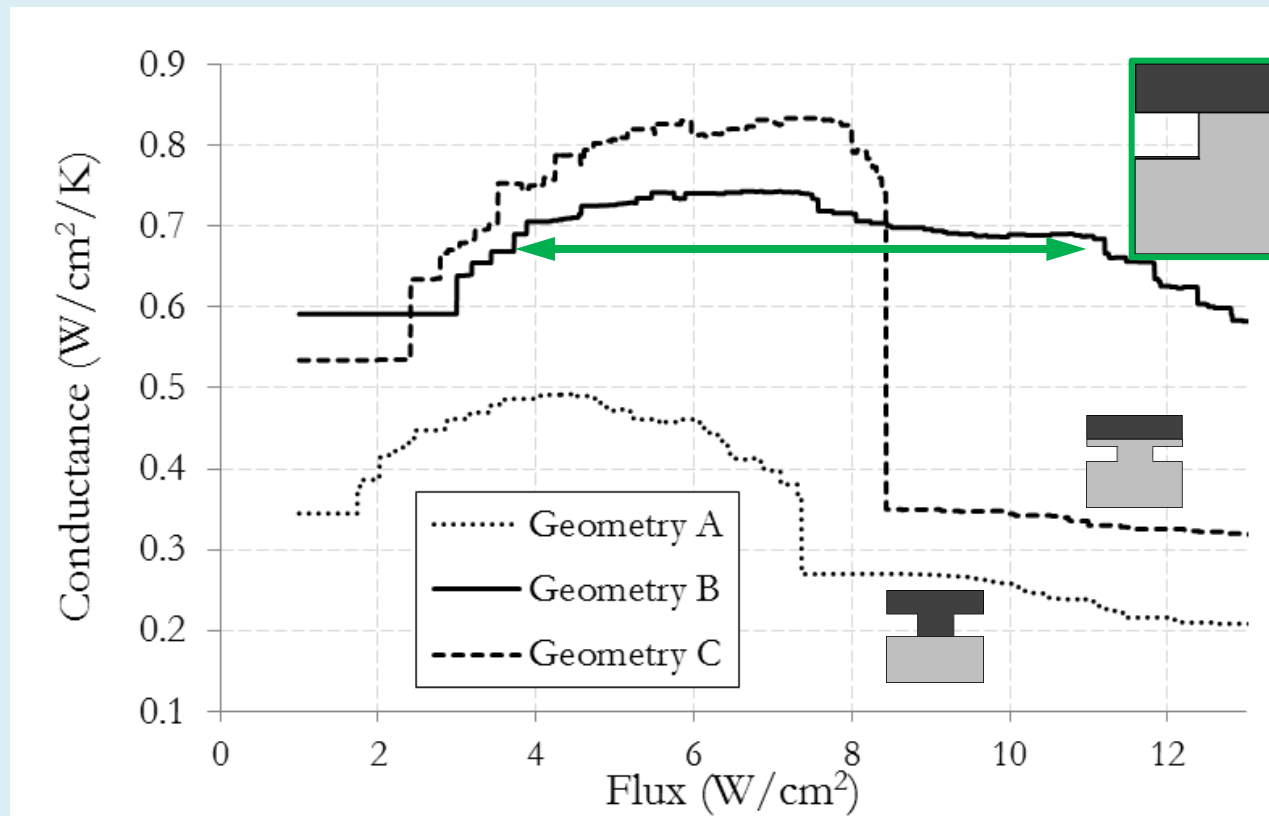
Conductance

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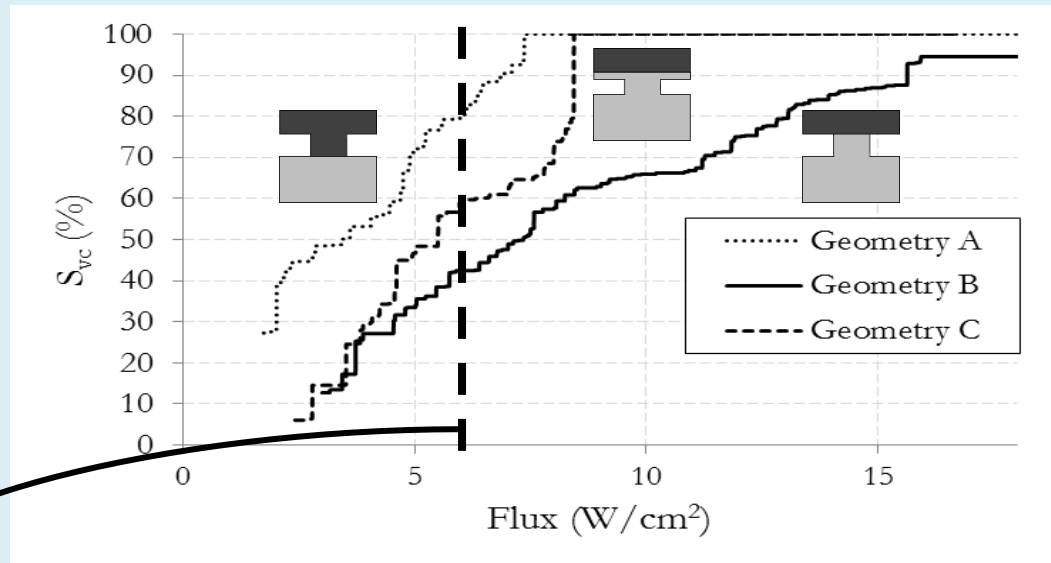
Conductance

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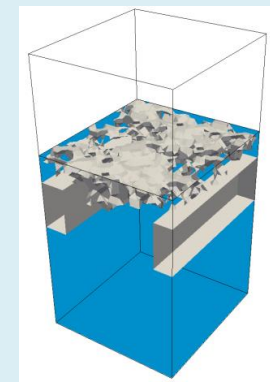
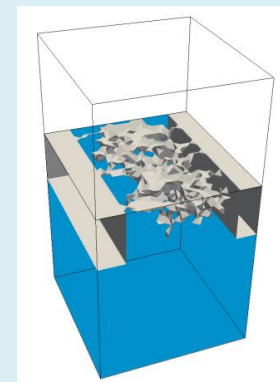
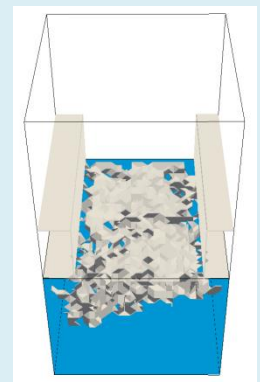


Fraction of vapour pores under the casing

Fraction of vapour pores under the casing

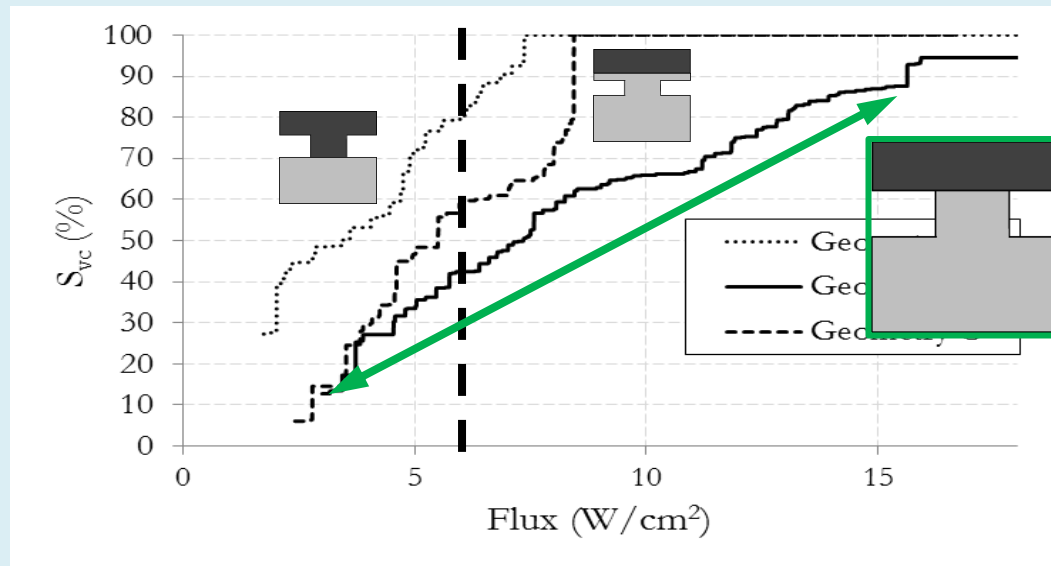
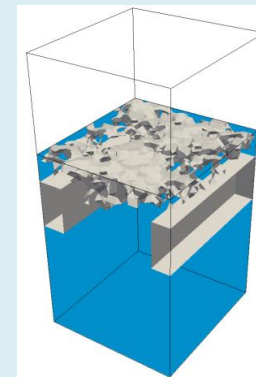
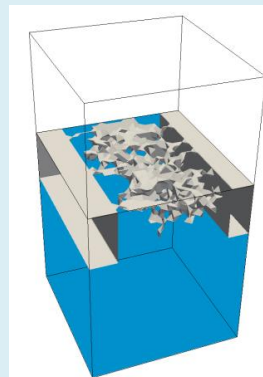
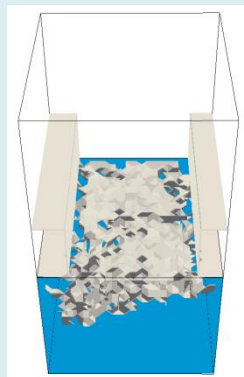


6 W/cm^2



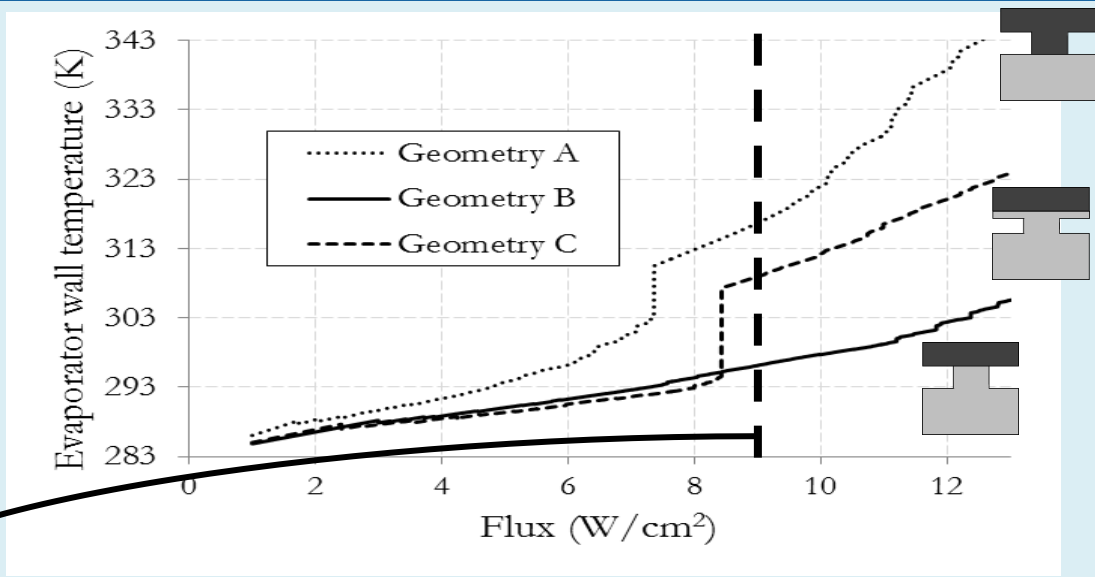
Fraction of vapour pores under the casing

Fraction of vapour pores under the casing

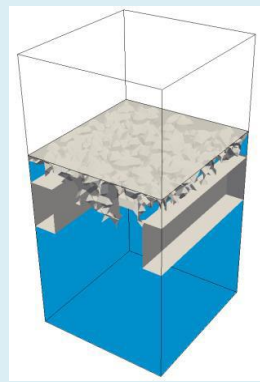
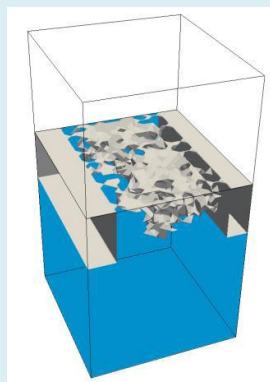
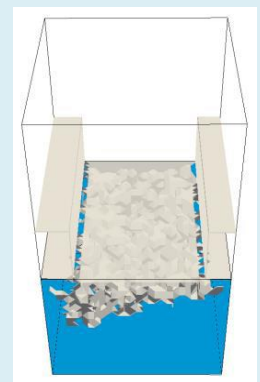
6 W/cm^2 

Evaporator wall temperature

Evaporator wall temperature

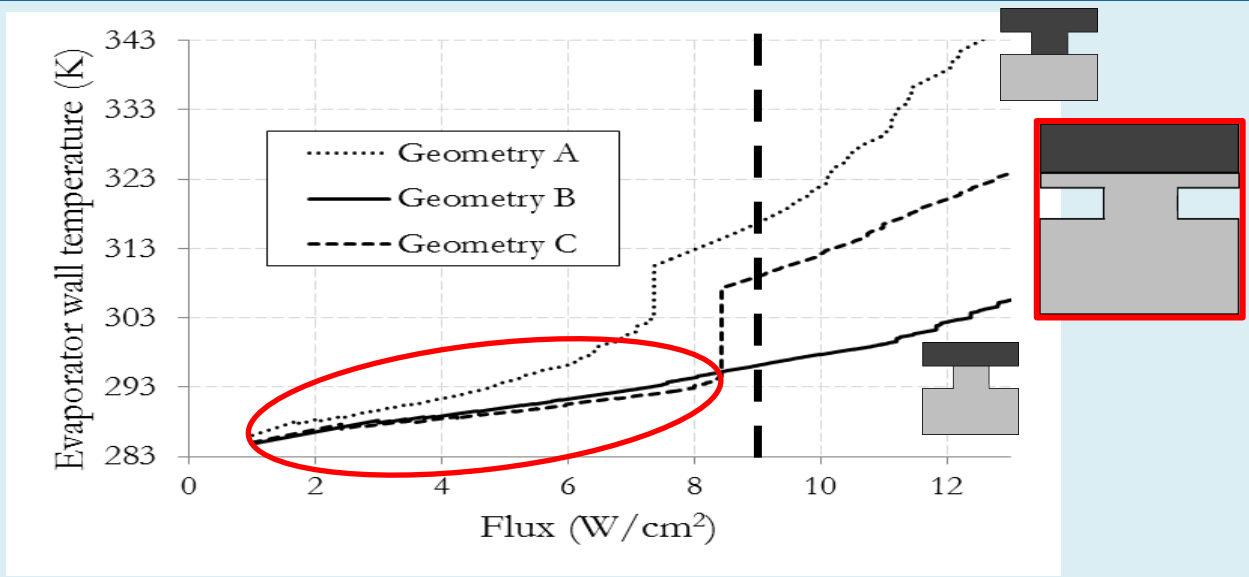


9 W/cm²

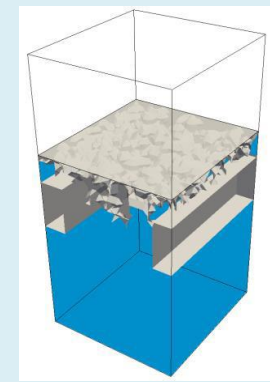
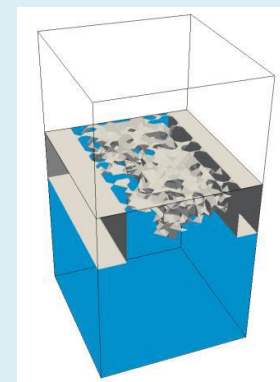
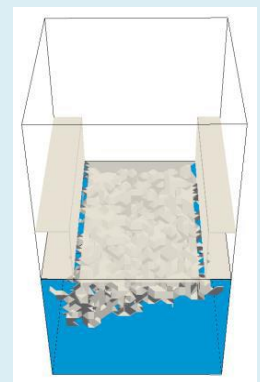


Evaporator wall temperature

Evaporator wall temperature

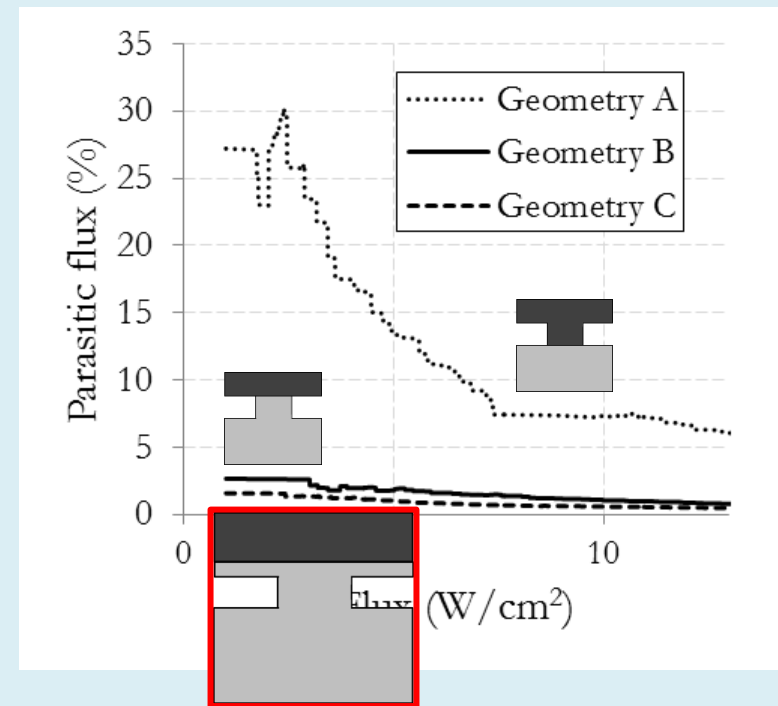
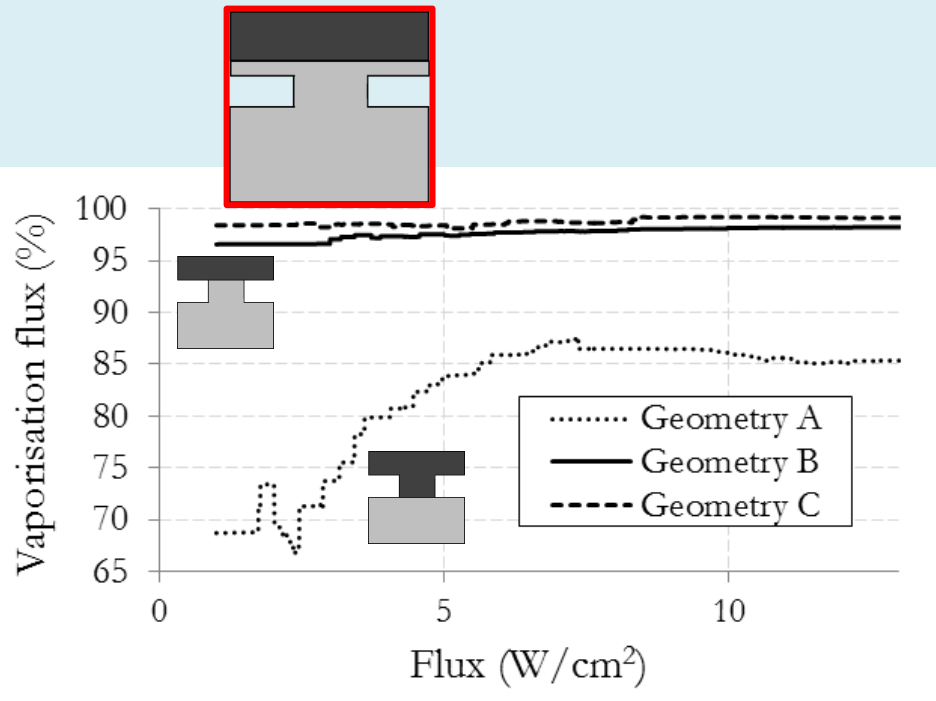


9 W/cm²



Vaporisation flux and parasitic heat flux

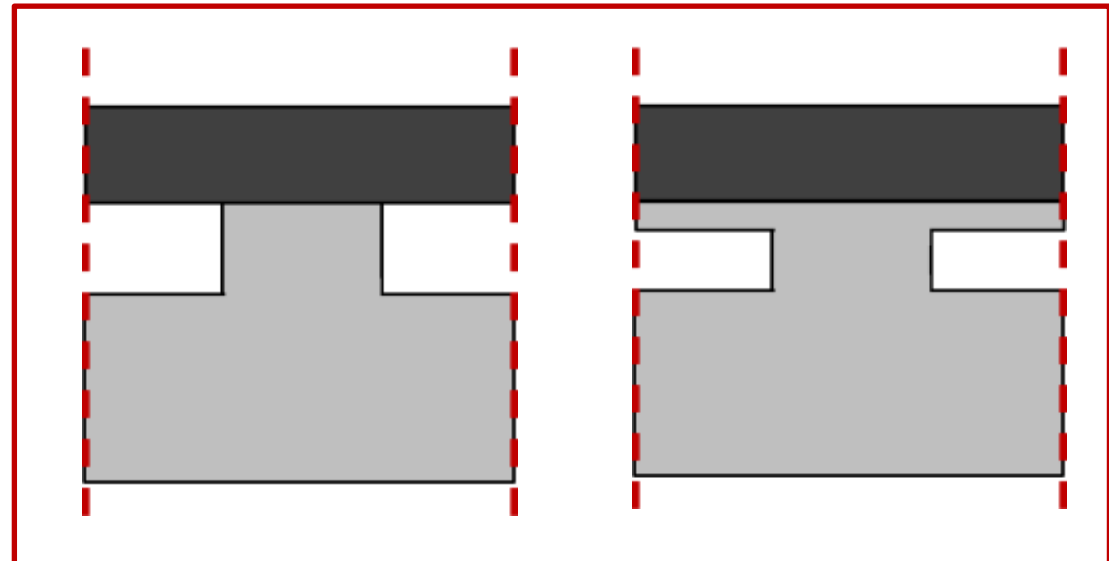
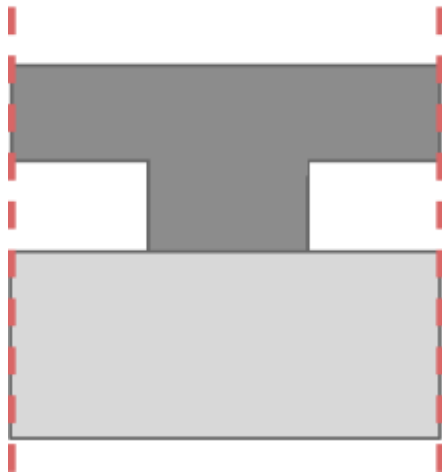
Vaporisation flux and parasitic heat flux



Summary

Geometry

- Groove locations



Summary

Geometry

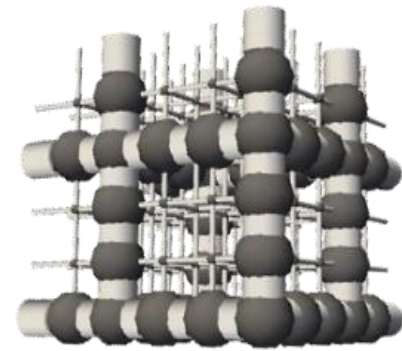
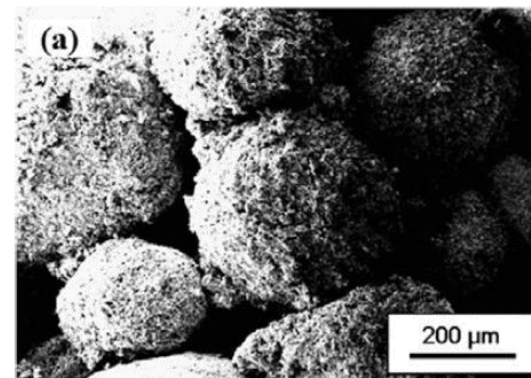
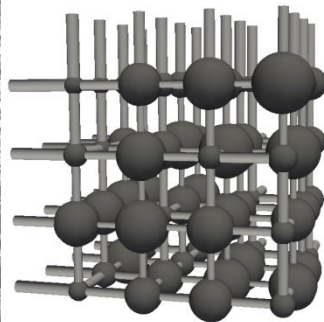
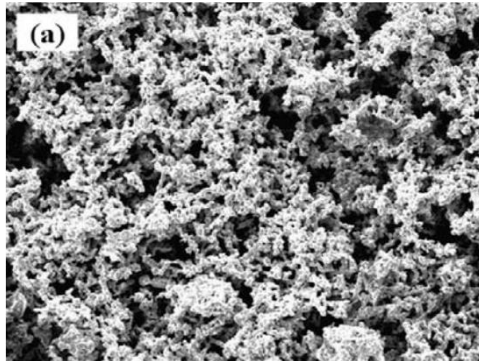
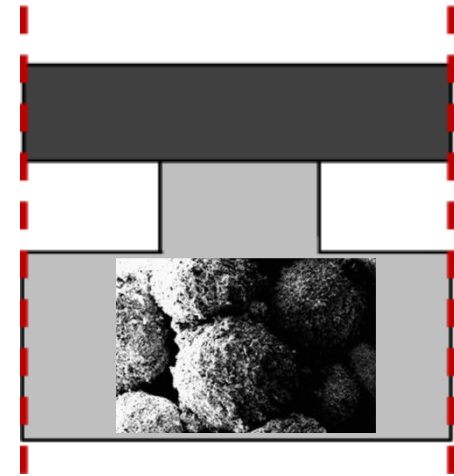
- Groove locations

Wick properties

- Monomodal vs. Bimodal
- Porosity
- Throat size distribution

Materials

- Casing
- Wick



Summary

Geometry

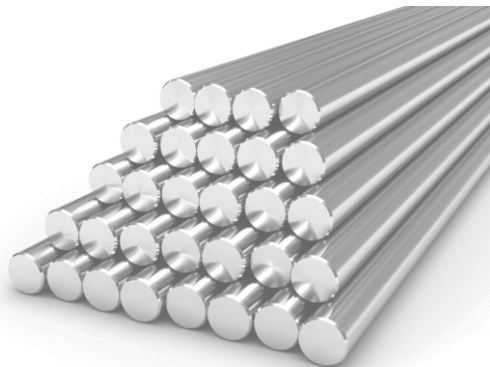
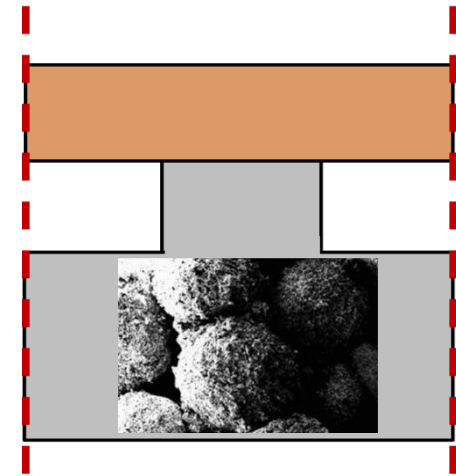
- Groove locations

Wick properties

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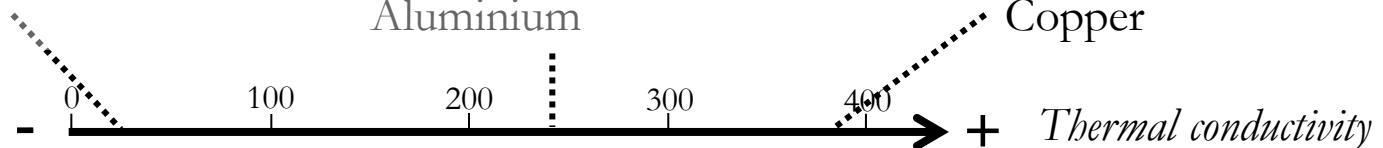
Stainless steel



Aluminium



Copper



Summary

Geometry

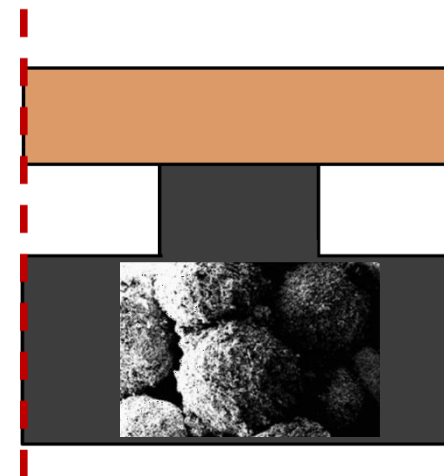
- Groove locations

Wick properties

- Monomodal vs. Bimodal
- Porosity
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Materials

- Casing
- Wick



PTFE



Nickel



Copper

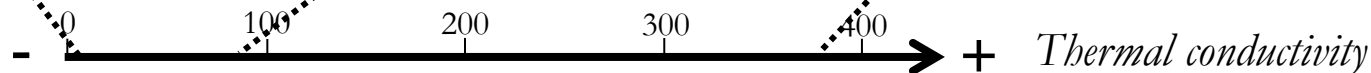


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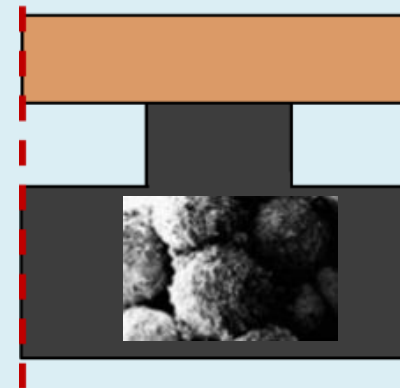
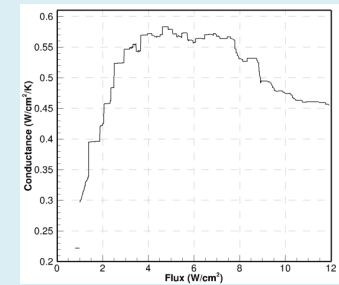
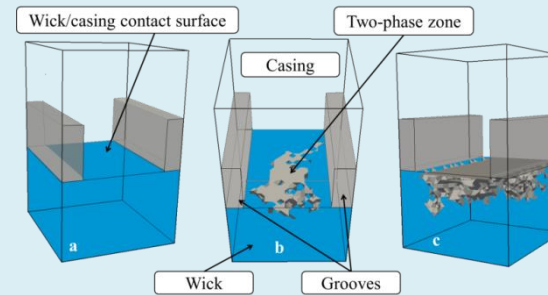
Conclusions

Main questions

- Regime(s)?
 - Saturated with liquid
 - ~~Vapour pocket mode~~
 - Two-phase zone

- Towards design optimisation?

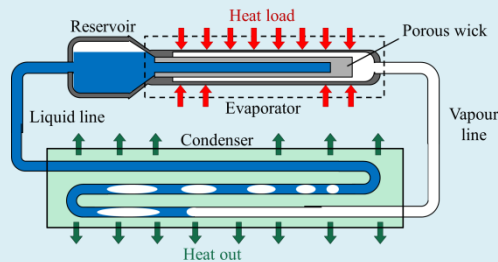
Answers



Further work

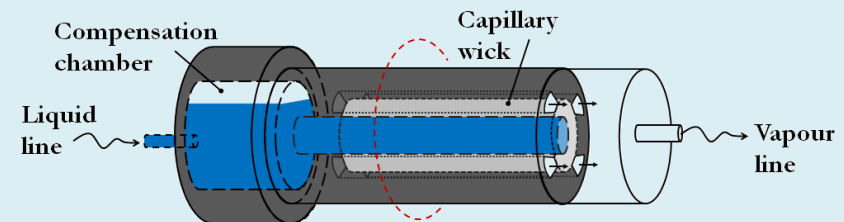
● Pore network model

- Loop model needs to be more accurate



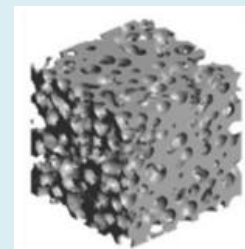
● Continuum model

- Extension to the evaporator scale



● Towards realistic microstructures

- Pore network simulations of continuum model parameters on 3D digital images of “real” microstructure

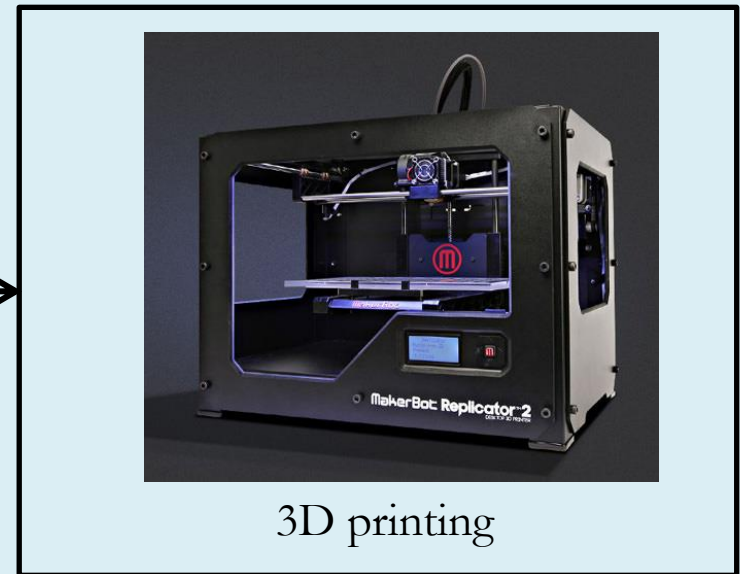
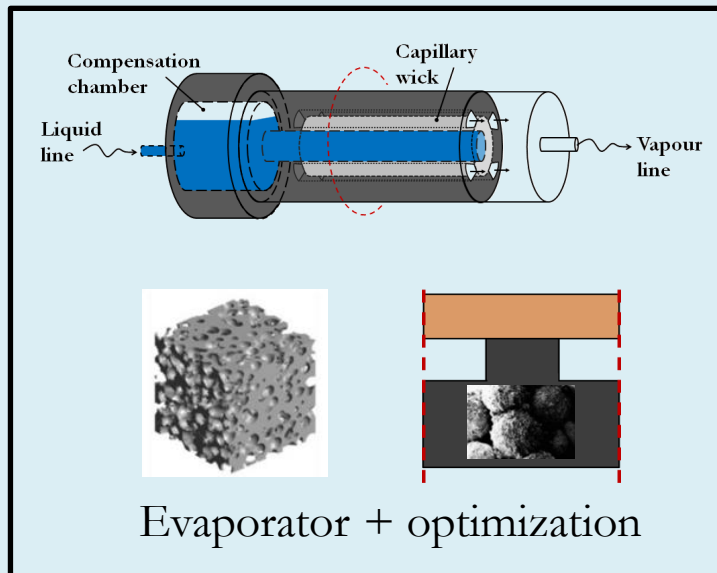


For example: sintered nickel microstructure.

Conclusions

In the longer run

● 3D printing of the evaporator



Thank you for your attention.

Financial supports from **CNES** and **Airbus Defence and Space** are gratefully acknowledged.